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Pg. 62

REPORT

PHASE I

SPACE STATION

LONG TERM

LUBRICATION

ANALYSIS

PHASE I

PRELIMINARY

TRIBOLOGICAL

SURVEY

To

GEORGE C. MARSHALL SPACE FLIGHT

CENTER

MARSHALL SPACE FLIGHT CENTER,

ALABAMA

SEPTEMBER 21, 1990

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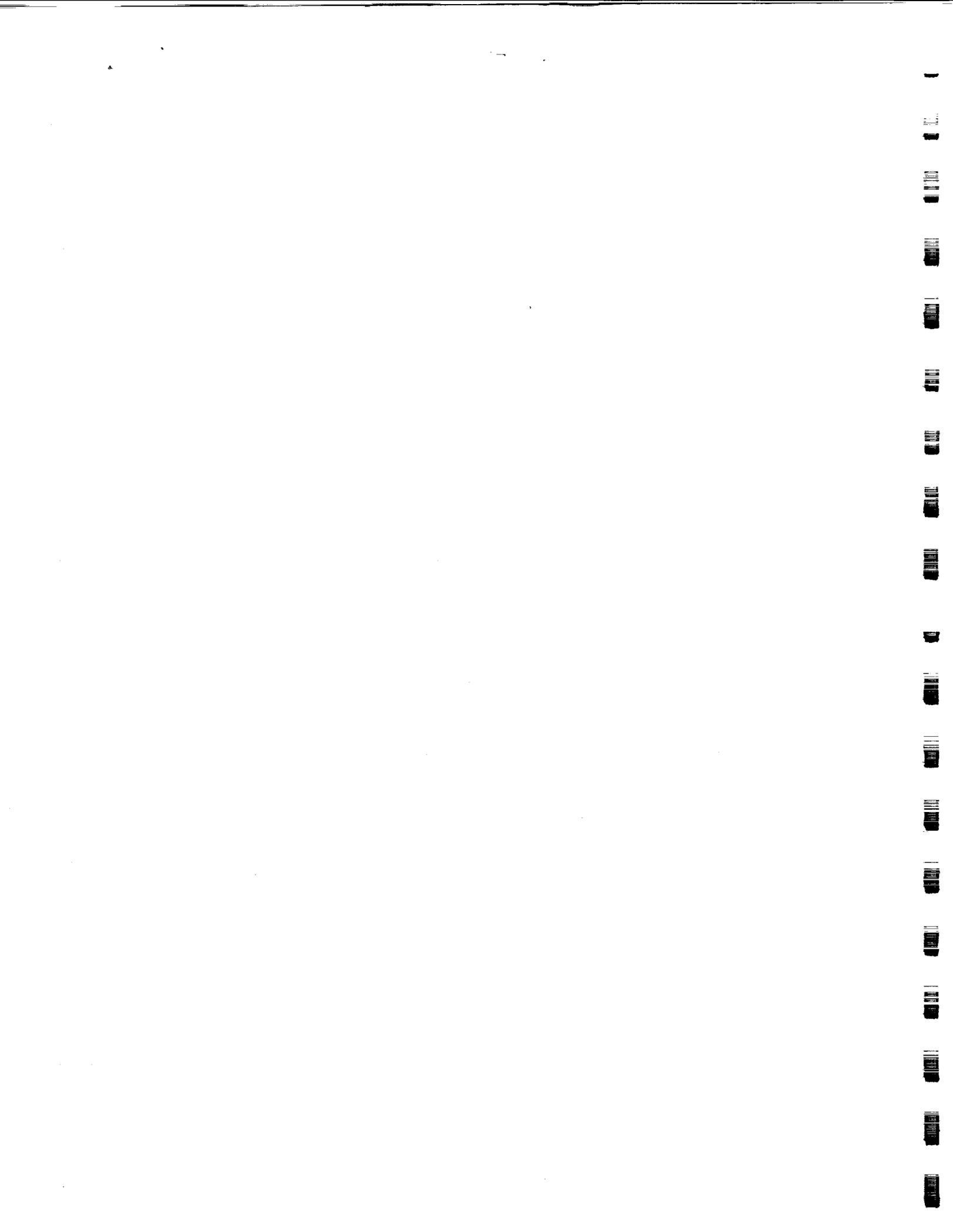
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(NASA-CR-184365) SPACE STATION
LONG TERM LUBRICATION ANALYSIS.
PHASE I PRELIMINARY TRIBOLOGICAL
SURVEY (Battelle Columbus Labs.)
62 P



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	<u>Monthly Progress Report</u>	<u>Final Progress Draft</u>	<u>Final Progress Approved</u>	<u>Financial Management Report</u>
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PHASE I REPORT

on

SPACE STATION LONG TERM
LUBRICATION ANALYSIS
PHASE I PRELIMINARY TRIBOLOGICAL SURVEY

to

GEORGE C. MARSHALL SPACE FLIGHT CENTER
Marshall Space Flight Center, Alabama 35812

September 21, 1990

by

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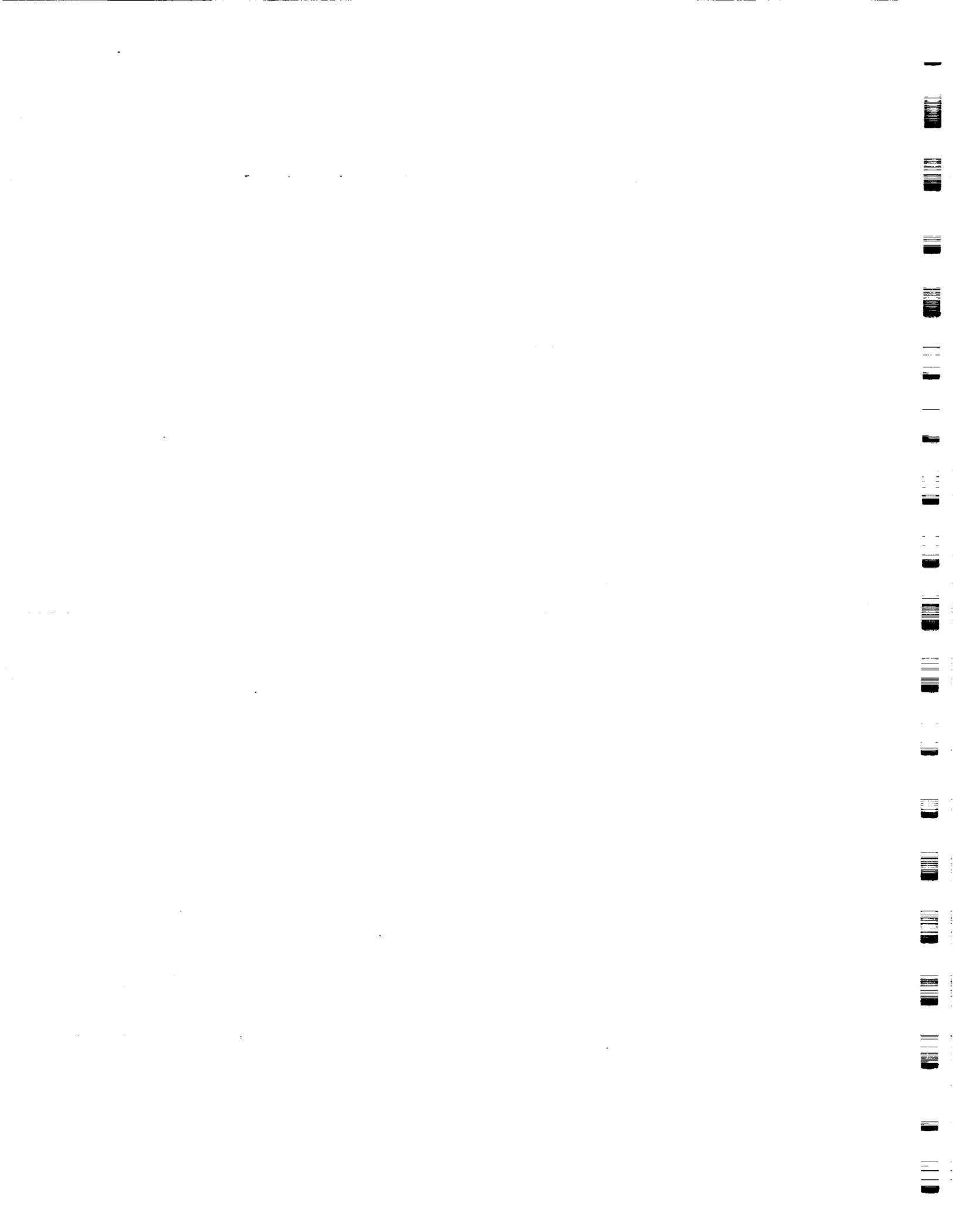


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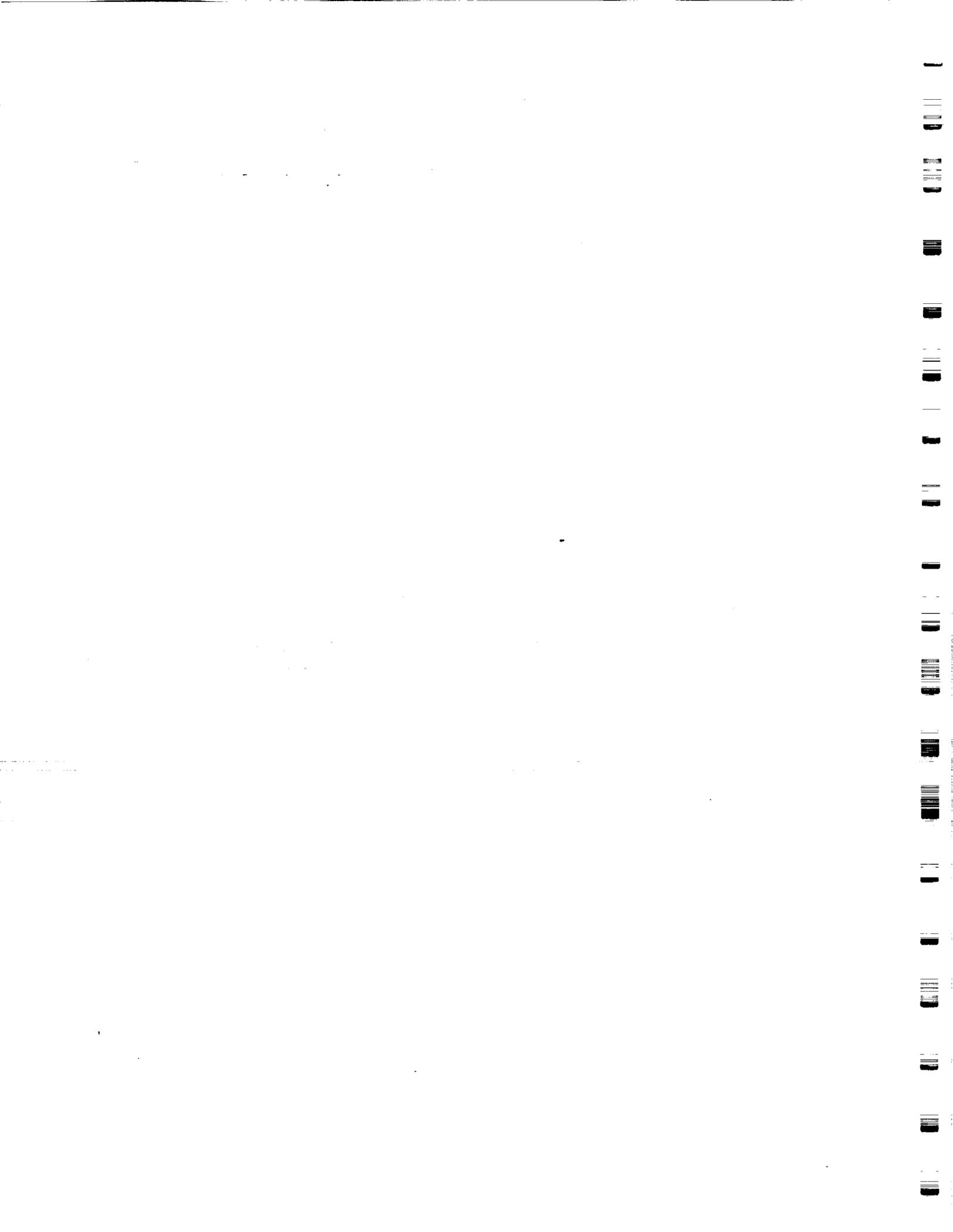
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INTRODUCTION

Increases in the size, complexity, and life requirements of satellites and space vehicles have put increasing demands on the lubrication requirements for trouble-free service. Since the development costs of large systems are high, long lives with minimum maintenance are dictated. The Space Station represents the latest level of size and complexity in satellite development; it will be nearly 100 meters in major dimensions and will have a life requirement of thirty years. It will have numerous

mechanisms critical to its success, some of which will be exposed to the space environment. Designing long-life lubrication systems and choosing appropriate lubricants for these systems will be necessary for their meeting the requirements and for avoiding failures with associated dependent mechanisms. The purpose of this program was to identify the various critical mechanisms and review their designs during the overall design and development stage so that problem areas could be avoided or minimized prior to the fabrication of hardware.

The specific objectives were fourfold:

- (1) To perform a tribology survey of the Space Station for the purpose of documenting each wear point as to materials involved, environmental conditions, and operating characteristics.
- (2) To review each wear point (point of relative motion) as to the lubrication used and substrate materials selected in the context of its operating characteristics and the environmental conditions imposed.
- (3) To make recommendations for improvement in areas where the lubricant chosen and/or where the substrate (materials of the wear couple) are not considered optimum for the application.
- (4) To make or recommend simulated or full scale tests in tribological areas where the state-of-the-art is being advanced, in areas where new designs are obviously being employed and a critical review would indicate that problems are a strong possibility, and/or where excessive wear, a malfunction, or excessive leakage

would create fluid systems problems or contamination of exposed optical equipment.

The contract, which was initiated on June 24, 1985, was originally to be conducted in two Phases over a three-year period. Phase I was a preliminary tribological survey of the Space Station structure, systems, and equipment and a forecast of probable problem areas. Phase II was to be a detailed tribological survey to be conducted in concert with the maturing design. Because of the lack of availability of detailed design data and of repeated scheduling and funding delays with the overall Space Station program, the detailed tribological survey of Phase II was not initiated in January, 1987, which would have been in accordance with the original schedule. Instead, the program was halted until April, 1990, when the overall scope was modified and the program was reinstated. The modified scope has included three phases.

Phase I. Preliminary Tribology Survey, Preliminary Assessment, and Forecast of Probable Problem Areas.

The Phase I activities were conducted in accordance with the original program scope and were essentially completed by early 1987. This report summarizes the Phase I activities.

Phase II. Lubrication Evaluation of the Alpha and Beta joints.

The Phase I efforts have identified the alpha and beta joints as critical and potentially problem-prone because of their size, service life requirements, and lack of previous comparable experience. Phase II

Phase II. (cont.) will consist of a review of these joints. The intent is to evaluate whether alpha and beta joints of the current design (incorporating reasonable modifications if found necessary) are likely to meet the requirements of consistent torque and outgassing specifications for a service life of 25 years. The review will include the lubrication methodology (grease, transfer films, or solid films) and the trade-offs considered in arriving at the methodology (maintenance intervals, outgassing, seals, friction levels, and debris generation). In addition, a review will be made of the tests already performed in support of the design and those specified for future evaluations. The results of the review, including an evaluation of the current design and recommendations for modifications or needed tests, will be documented in the Phase II section of the final report.

Phase III. Lubricant Selection Guide

Phase III will consist of preparation of a practical lubricant selection guide for use by designers of Space Station components. While the potentially available liquid and dry-film lubricants number in the hundreds, a large percentage are not practical for Space Station components because of such factors as outgassing properties, creep properties, viscosity, long-term stability, corrosive properties, or wear protection. The purpose of the Phase III effort will be to reduce the number of lubricants to a practical

working number for designers to consider. Guides will be given for the selection of specific lubricants from the recommended list. The design guide will be incorporated in the final report.

This report is to document the Phase I activities. It describes the general Space Station requirements that will affect lubrication and the basic lubrication approaches available to address these requirements. The report by SRS Technologies, a major subcontractor on Phase I, is included in its entirety as Appendix A. SRS Technologies had responsibility for documenting the Space Station components and mechanisms regarding lubrication requirements.

SUMMARY

The Phase I activities began with a review of the various work packages of the primary contractors to identify tribological points of interest (SRS Technologies) and a review of pertinent literature to identify likely lubricants for use in the tribological applications (Battelle). As details of the various tribological points were identified, Battelle was to perform a review to assess the likelihood of success. Difficulties were encountered in attaining the needed design details because of the early stage of the design (details on loads and sizes were not available) and because of overall program delays. Consequently, a comprehensive review could not be made of most of the 72 specific components identified in the survey of 14 target projects. Of these, the most critical by far were the alpha and beta joints because of their size, lack of prior history, and their criticability to the Space Station. Since a change was made in contractors for these joints from the time the Phase I survey was conducted, the detailed design information obtained in the survey was useful only in a general sense. Because of the importance of these joints, the modified scope for the Phase II activities of the program concentrates specifically on these joints.

The literature review identified papers, reports, and books dealing with developments in lubricants for aerospace mechanisms. Recent advances in solid lubricants have involved the additions of antimony and antimony compounds to improve the friction and wear properties of bonded molybdenum disulfide solid lubricants. The selection criteria are based on the longevity needed and the possible deterioration of the binder from exposure to the space environment. Of the various candidate liquid (and grease) lubricants, the perfluoro ethers have come to dominate the field because of their low vapor pressure and acceptable lubricating properties. Friction and wear data and flight history data are available to assist in the selection and application of the perfluoro ethers. The Lubricant Selection Guide of Phase III will detail the specific preferred lubricants for example families of tribological contacts and provide data on advantages, disadvantages, and life-limiting considerations for their application.

SPACE STATION LUBRICATION REQUIREMENTS

The combination of size, complexity, and desired life of the Space Station is unprecedented in satellite design. As shown in Figure 1, the overall layout of the Space Station includes a central module that will be held in alignment to the surface of the earth. Large solar panels to provide power will be attached through "alpha" and "beta" joints to permit orienting the panels toward the sun. The size of the joints (approximately 3 meters in diameter for the bearing at the alpha joint) and associated structure demands that these bearings be capable of operation for the intended thirty year life. Changing the bearings would require major unacceptable provisions for astronaut time and for restraining the panel assembly. Inherent reliability of the bearing and lubrication systems for these large structures is therefore required.

The thirty year life will also expose components to 175,000 thermal cycles. The range of these cycles will be controlled primarily by the optical properties of the surfaces involved unless active thermal control is provided. Thermal cycles can affect lubricants in several ways. For liquid lubricants, the immediate effect is a change in viscosity. If the lower temperature extreme is below the pour point of the

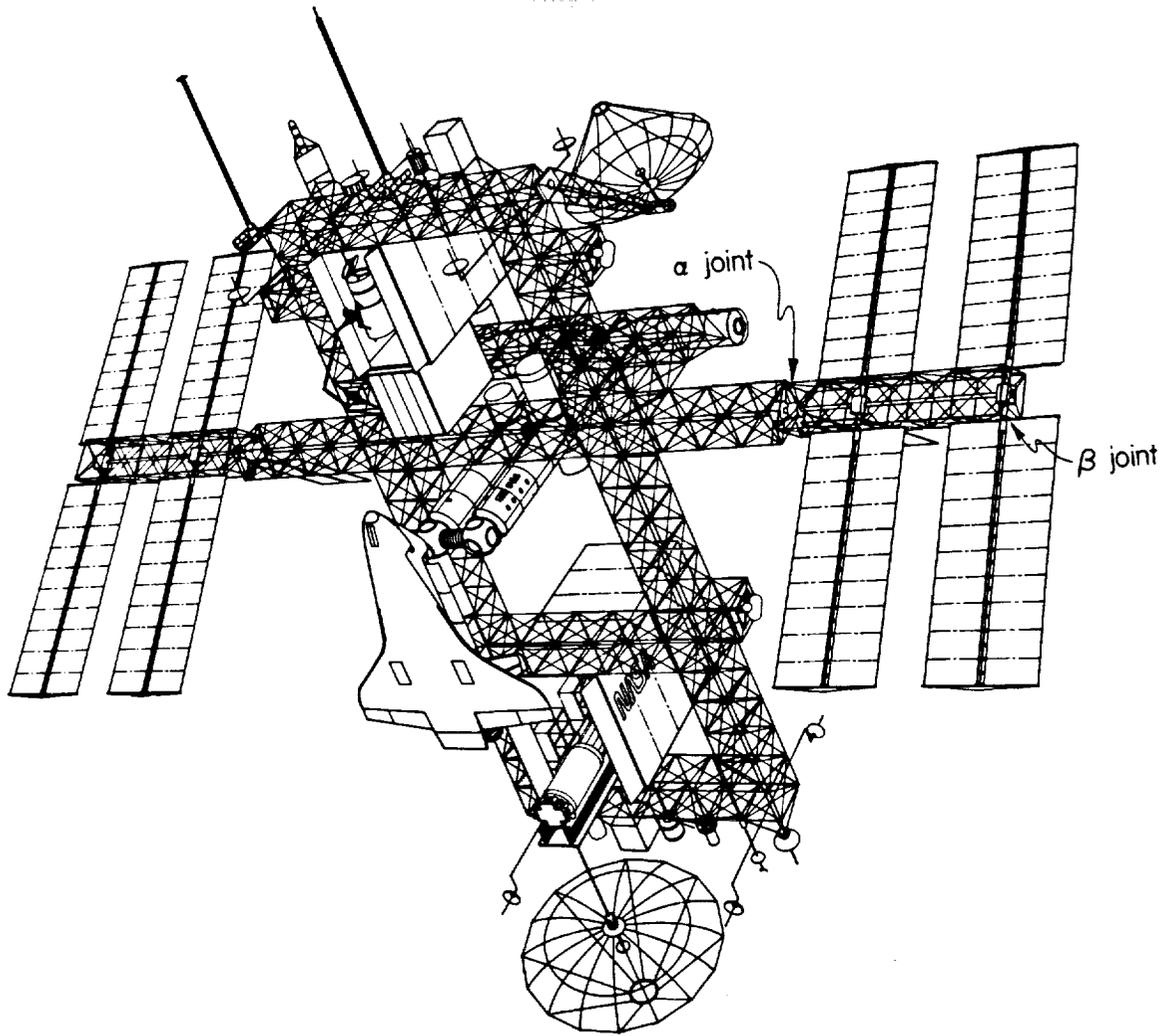


FIGURE 1. OVERALL LAYOUT OF SPACE STATION

liquid, it may be unable to reach the critical surfaces in the bearing or mechanism needing lubrication. Similarly, the high temperature extreme may lower the viscosity to a level where the lubricant cannot function properly. High temperatures can also degrade the lubricant through a variety of thermal degradation mechanisms. While dry film (solid) lubricants are relatively unaffected by temperature cycles, the binding agent must be chosen appropriately to avoid thermal degradation at the upper temperatures.

The thirty year life requirement introduces time-related lubrication degradation mechanisms, which must also be considered. Evaporation, creep, and degradation by atomic oxygen all represent potential problems that must be addressed in the design of mechanisms and selection of lubricants. Evaporation and creep of liquid lubricants present the dual problems of lubricant loss at the needed surface and of potential contamination of optical surfaces by the lost lubricant. These problems are controlled primarily by the basic chemical composition of the lubricant. Atomic oxygen, however, presents a new challenge for spacecraft design in that a variety of organic materials are attacked. Experiments on two Space Shuttle missions, Space Transportation Systems 5 and 8 (STS-5 and STS-8), provided data that showed materials containing carbon, silver, and osmium react with atomic oxygen to form volatile oxides⁽¹⁾. Table 1 lists the reaction efficiencies for a representative set of materials⁽²⁾. Because of the reactions with organic materials, liquid lubricants and dry film lubricants having organic binders must be protected from exposure or selected with a knowledge that continual degradation will occur. In the case of thin solid films using an epoxy-based binder, exposure to atomic oxygen may degrade the binder in a matter of days⁽²⁾. For practical systems requiring exposure, therefore, inorganic binders, such as silicates, may be required to avoid the problem.

TABLE 1. REACTION EFFICIENCIES OF SELECTED MATERIALS WITH ATOMIC OXYGEN IN LOW EARTH ORBIT (REFERENCE 2)

Material	Reaction Efficiency, 10^{-24} cm ³ /atom
Kapton	3
Mylar	3.4
Tedlar	3.2
Polyethylene	3.7
Polysulfone	2.4
Graphite/epoxy	
1034C	2.1
5208/T300	2.6
Epoxy	1.7
Silicones	< 0.02*
White paint A276	0.3 to 0.4*
Black paint Z302	2.3*
Perfluorinated polymers	
Teflon, TFE	< 0.05
Teflon, FEP	< 0.05
Carbon (various forms)	0.9 to 1.7
Silver (various forms)	Heavily attacked

*Units of mg/cm² for STS-8 mission. Loss is assumed to occur in early part of exposure; therefore, no assessment of efficiency can be made.

Lubricant Considerations

Lubricants can be divided into two basic classes: solid films and liquids. Both types have been used extensively in space applications. Both have advantages and disadvantages that must be carefully considered in their selection. The factors of particular importance for Space Station lubrication are considered in this section.

Solid Film Lubrication

Advantages and Disadvantages

From Reference 3, the following lists the basic advantages and disadvantages of solid film lubricants.

Advantages of Solid Lubricants

1. Do not collect grit.
2. Can be used under extremely high load conditions.
3. Excellent storage stability.
4. LOX and oxygen compatible (inorganically bonded films).
5. Suitable for use over wide temperature range.
6. Resistent to the effects of nuclear and gamma radiation.
7. No disposal problem.
8. Friction decreases with increasing load.
9. In some applications solid films will provide lubrication for the life of the parts.

Disadvantages of Solid Lubricants

1. Limited amount of lubricant available.

2. Friction coefficient higher than with hydrodynamic lubrication.
3. Provisions for the effective removal of wear debris must be provided.
4. Considerations must be given to removing heat from contact zone of bearings and gears when using solid film lubricants.
5. More expensive (costly relubrication).
6. Avoidance of contamination during coating processes and assembly of parts lubricated with solid film lubricants.
7. Elevated temperature cure cycle of some solid films will damage the mechanical properties of some materials.

Selection of Solid Lubricants

Solid lubricants provide capabilities unavailable with liquid lubricants, but they are not a universal lubricant. The requirements of some applications prevent their use entirely. Also, there is no single solid lubricant that will meet all of the requirements. Therefore, the selection of the basic class of lubricant (solid or liquid) and the specific lubricant must consider the needs of the particular application and the requirements of the system of which the application is a part.

The obvious advantages of solid films are that they add virtually no weight to the system and create virtually no problems due to outgassing. The primary disadvantages of solid films are that they have limited life and are very difficult to replenish.

The three basic solid film lubricants that have traditionally been given the most attention and used most extensively are:

- Graphite,
- Polytetrafluorethelene (PTFE), and
- Molybdenum disulfide (MoS₂).

Table 2⁽⁴⁾ lists several bonded films and application methods. Recent work in solid films has involved some compositional variations in these lubricants primarily to

TABLE 2. BONDED SOLID FILM LUBRICANTS (REFERENCE 4)

Materials		
Binder	Additives	Application Method
Polyimide	$\text{MoS}_2 + \text{Sb}_2\text{O}_3$	Sprayed, then heat-cured 1 hr. at 200 F, followed by 1 hr. at 575 F.
Polybenzothiazole	$\text{MoS}_2 + \text{Sb}_2\text{O}_3$	Sprayed, then heat-cured 1 hr. at 200 F, followed by 1 hr. at 600 F.
Polybenzimidazole	$\text{MoS}_2 + \text{Sb}_2\text{O}_3 + \text{ZnO}$	Sprayed, then heat-cured 1 hr. at 200 F, followed by 1 hr. at 575 F.
Methylphenylsilicone	$\text{MoS}_2 + \text{Sb}_2\text{O}_3$	Sprayed, then cured at room temperature for 24 hours
Sodium silicate	$\text{MoS}_2 + \text{graphite} + \text{Au}$	Sprayed, then heat-cured 2 hrs. at 180 F, followed by 2-16 hrs. at 400 F.
Aluminum phosphate	$\text{MoSe}_2 + \text{TaS}_2 + \text{graphite}$	Sprayed, then heat-cured 1 hr. at 150 F, 2 hrs. at 200 F, 8 hrs. at 400 F.
Aluminum phosphate	$\text{BaF}_2:\text{CaF}_2$	Sprayed, then heat-cured 1 hr. at 95 C, 200 C, 300 C, 1 hr. at 800-1100 C.
Polybenzimidazole	$\text{MoS}_2 + \text{Sb}_2\text{O}_3 + \text{prefused fluorides}$	Sprayed, then heat-cured 1 hr. at 95 C, 200 C, 300 C, 1 hr. at 800-1100 C.

achieve improved wear life and possibly reduced friction.

Bartz, Holeniski, and Xu⁽⁵⁾ indicate that there exists optimum concentrations for MoS₂ doped with materials such as graphite and antimony compounds to yield longer wear life than attainable with single components. A comparison of wear lives as obtained in rub block experiments is given in Table 3, and the friction coefficients are presented in Table 4. Table 5 shows the test conditions. Wear lives of 100,000 cycles are possible with this approach, which should be adequate for many components of Space Station. While graphite is probably inappropriate for vacuum service. Bartz research indicates that solid films exist that should provide good service life.

Fleshauer⁽⁶⁾ is doing extensive work in evaluation of solid films, especially MoS₂ formulations. Table 6 summarizes the wear lives obtained with three different contact conditions. The pin-on-disk tests were run at a load of 700 MPa. The results have shown that the wear life can be increased significantly if the MoS₂ is doped with antimony. Again, a wear life of 10⁵ cycles is shown to be possible.

Rolling Contacts

Friction in rolling contact can be lower with solid films than with grease lubrication. Todd and Bentall⁽⁷⁾ present data (Figure 2) that illustrate this effect. Table 7 summarizes data for solid lubricated ball bearings⁽⁷⁾. In some ball bearings, it is possible to extend the coating life by using transfer film technology^(8,9). In this technology the solid film is transferred from the bearing cage to the ball and races. The cage in essence is then the lubricant supply. Transfer film technology represents a good approach for extending the life of space bearings beyond that attained with solid lubricant coatings.

A summary of examples of successful applications of solid film lubricants to cams, gears, and bearings in spacecraft^(10,11) is given in Table 8.

TABLE 3. WEAR LIFE OF SOLID FILMS (REFERENCE 5)

Material	K Cycles	
	980 N (500 min ⁻¹)	980 N (1000 min ⁻¹)
MoS ₂	50	< 10
Graphite	< 10	< 5
Sb(SbS ₄)	--	--
MoS ₂ + Sb(SbS ₄)	140	40
Graphite + Sb(SbS ₄)	20	10
MoS ₂ + Graphite	200	75
MoS ₂ + Graphite + Sb(SbS ₄)	500	100
Bonded Solid Lubricant	200	50

TABLE 4. STABLE FRICTION COEFFICIENT (REFERENCE 5)

Load N	Speed min ⁻¹	Lubricant			
		Graphite	CSb-B	MoS ₂	MSb-B
245	500	none	0.14-0.15	0.03-0.05	0.02-0.04
980	500	none	none	0.05	0.01-0.03
1470	500	none	none	none	none

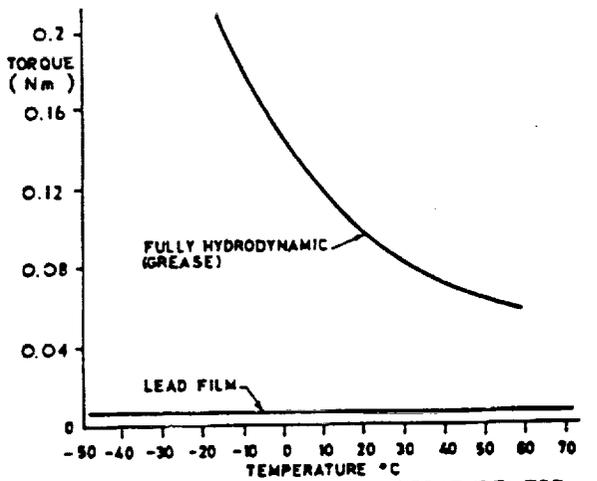
TABLE 5. EXPERIMENTAL CONDITIONS FOR DATA
IN TABLES 3 AND 4 (REFERENCE 5)

Ring	
outer diameter:	0.47 mm
material:	100 CrMn6 steel
hardness:	HRC 60
roughness of surface after sandblasting:	12 μ m CLA
Block	
dimensions:	24 x 15 x 6
material:	90 MnCrV8 steel
hardness:	HRC 54
roughness of surface:	1.6 μ m CLA

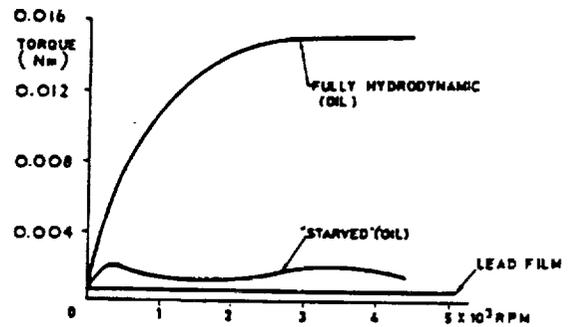
TABLE 6. WEAR TEST RESULTS FOR MoS₂ FILMS
(500 - 1000 nm THICK) FROM DIFFERENT
LABORATORIES (REFERENCE 6)

Wear Life, Thousands of Revolutions		
Pin-On-Disk	Dual-Rub-Shoe	Thrust-Washer
201	19 ^a	600-700
---	55 ^c	800
156	60 ^b	300
45	35 K ^a	900

^aTested in air; ^bAir or vacuum; ^cVacuum



a. TORQUE VARIATION WITH TEMPERATURE FOR GREASE AND LEAD FILM-LUBRICATION (60mm BALL BEARING 40N AXIAL LOAD SPEED 100r.p.m.)



b. TORQUE VARIATION WITH SPEED FOR LIGHT OIL- AND LEAD FILM-LUBRICATION (42mm BALL BEARING, 30N AXIAL LOAD, ROOM TEMPERATURE)

FIGURE 2. COMPARISON OF BALL BEARING OPERATING TORQUES SHOWING THAT SOLID FILM LUBRICATION CAN PRODUCE LOWER TORQUES THAN OBTAINED WITH LIQUIDS (REFERENCE 7)

TABLE 7. TORQUE BEHAVIOR OF SOLID LUBRICATED BALL BEARINGS
(REFERENCE 7)

Method of Lubrication	Preload (N)	Cumulative time at 100 rpm (min)	Torque (N-m x 10 ⁻⁴) to oscillation		Remarks
			Average (±)	Peak-Peak	
PTFE/MoS ₂ / glass fiber cage, degreased races and balls	40	0	21	76	Progressive torque increase
		30	25	98	
		60	57	147	
MoS ₂ -coated raceways, Phenolic cage	100	0	34.5	148	Smooth start } trace } developing } spikes
		5	31.5	156	
		15	42	246	
		30	45	282	
		60	39	168	
		74			
IP lead on raceways, lead bronze cage	100	0	50	320	Spikes during run-in } very smooth } stable } torque traces
		5	45	132	
		15	45	120	
		35	45	135	
		60	45	180	
		240	failed by excess torque		

TABLE 8. SUMMARY OF SOLID FILM APPLICATIONS FOR SPACECRAFT (REFERENCE 10, 11)

Spacecraft	Item	Pitch	Material	Part Coated	Lubricant	Thickness (mm)
TRIAD	Gear Box Gears	48	Corrosion Resistant steel	Gear teeth	Coating No. 2	0.005-0.0127
	Worm	48	Corrosion Resistant steel	Gear teeth	Coating No. 2	0.005-0.0127
	Worm gear	48	Bronze	Gear teeth	Coating No. 1	0.0127-0.0254
	Miter gears	48	Titanium	Gear teeth	Coating No. 1	0.0127-0.0254
	Miter gears	48	Aluminum	Gear teeth	Coating No. 1	0.0127-0.0254
	Thrust bearing	--	Corrosion Resistant steel	Ball Raceways	Sputtered MoS ₂	0.0015
APOLLO 17	Ball bearings	--	Corrosion Resistant steel	Inner race Outer race Retainer	Coating No. 1	0.0127-0.0254
	U. V. Spectrometer wavelength cam	--	---	Cam	Coating No. 2	> 0.0076
	"	--	52100 steel	Drive motor R6 bearings	Coating No. 1	--
	"	--	52100 steel	Follower R.8 bearings	Coating No. 1	--
SAS-C	Worm	48	Corrosion Resistant steel	Gear teeth	Coating No. 2	0.005-0.127
	Worm gear Bearings	48	Bronze	Gear teeth	Coating No. 2	0.005-0.0127
		--	Corrosion Resistant steel	Inner race Outer race Retainer	Coating No. 1	0.0127-0.0254
	Spur gear	48	Corrosion Resistant steel	Gear teeth	Coating No. 2	0.005-0.0127
	Spur gear	48	Aluminum	Gear teeth	Coating No. 1	0.0127-0.0254

Spacecraft Applications of Bonded Coatings⁽⁸⁾

Coating No. 1. Molybdenum disulfide, graphite, gold, sodium silicate.

Coating No. 2. Molybdenum disulfide, antimony trioxide, polyimide.

Liquid/Grease Lubrication

Advantages and Disadvantages

The primary advantage obtained with liquid lubricants is that bearing surfaces separated by hydrodynamic films of liquid lubricants have virtually no wear and thereby have the potential for indefinite lives. Liquid lubricants provide the viscosity needed for forming the hydrodynamic films, low shear strengths for low friction, cooling capability in recirculating systems, and the ability to minimize wear in low-speed (non-hydrodynamic) situations. Since no single lubricant can meet the often conflicting requirements of various applications for liquids, hundreds of specialty lubricants have been developed for aerospace applications⁽³⁾. The primary disadvantages of liquid lubricants are the need for containment, the propensity to creep, large changes in viscosity with temperature, and loss by evaporation under vacuum conditions. The use of thickeners to form greases provides a means of retaining the liquids in the needed region, thereby addressing one of the primary disadvantages. Greases are widely used for aerospace lubricants. The loss by evaporation greatly restricts the available liquids for vacuum applications to the few chemical species having low vapor pressures. The following sections consider the evaporation rates and the lubricating performance of liquids (and greases based on these liquids) in bearing applications.

Thermo-Vacuum Evaporation

The evaporation rate of lubricants in a vacuum is a function of their molecular weight, their vapor pressure, and the temperature. The Langmuir expression⁽¹²⁾ relates these factors and permits predicting the loss rate when the vapor pressure and temperature are known:

$$R_{\text{evap}} = \frac{P}{17.14} \left(\frac{M}{T} \right)^{1/2}$$

where,

P = vapor pressure (mm of Hg),

M = molecular weight, and

T = temperature of lubricant (°K).

The vapor pressure is strongly dependent on temperature, as shown for a perfluoro ether in Figure 3⁽²⁾. Perfluoro ethers are among the fluids having the lowest vapor pressures and are leading candidates for satellite applications exposed to vacuum. At the top of Figure 3 is the time predicted to evaporate a film 2.5×10^{-4} cm (100 microinches) thick in accordance with the Langmuir expression. With this strong temperature dependence, two conclusions are drawn:

1. The temperature of lubricant films exposed to vacuum must be controlled to retain the lubricant.
2. Provisions must be made for the reapplication of lubricant if temperature cannot be controlled to acceptable levels.

The chemical composition of a lubricant and its molecular weight are the dominating factors in determining the resulting vapor pressures and loss rates. For vacuum applications, silicones and perfluoro ethers have lower loss rates by 4 to 5

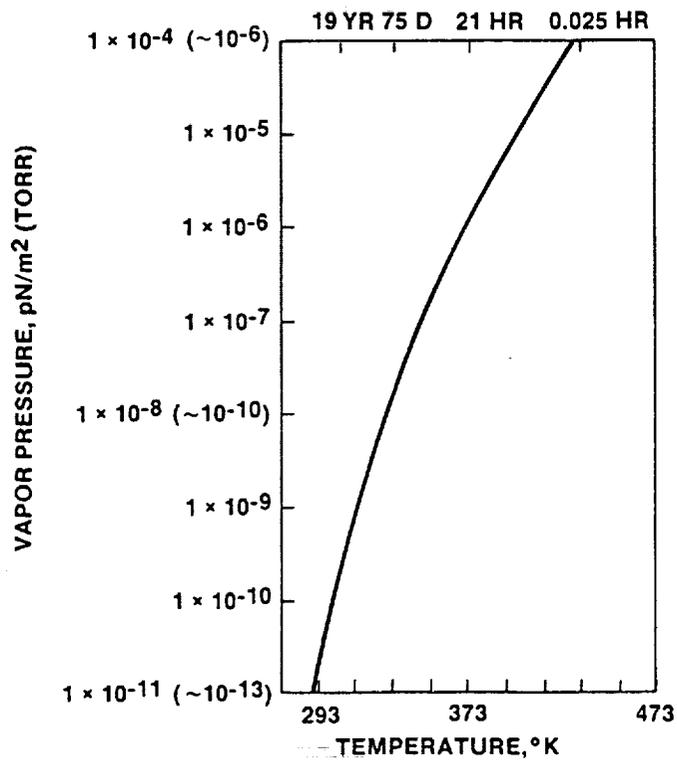


FIGURE 3. DEPENDENCE OF VAPOR PRESSURE ON TEMPERATURE FOR A PERFLUORO ETHER LUBRICANT (REFERENCE 2)

orders of magnitude compared with mineral oils (hydrocarbons) or diesters⁽¹³⁾. On the basis of loss rates by evaporation, the selection of lubricants is limited to the perfluoro ethers or silicones -- both from the standpoint of retaining the lubricant on the bearing surfaces where they are needed and of preventing contamination of optical systems by condensation of the evaporated lubricant. As discussed in the next section, the wear performance of the various lubricants combined with the creep behavior of silicones further limits the practical choice to the perfluoro ether fluids.

Friction and Wear Performance

Among the many properties provided by liquid lubricants for bearing applications, the ability to generate hydrodynamic films to separate the surfaces in relative motion and the ability to maintain low wear rates if the operating conditions prevent film formation are two of the most important properties. Film formation capabilities are largely determined by the viscosity, which is generally high in fluids selected to have low evaporation rates. Therefore, the ability to lubricate under very thin film (boundary) conditions is the performance property of interest.

Table 9 presents a summary of comparison data obtained from LFW-1 ring-on-block tests with various lubricants and 440C rings and blocks⁽¹⁴⁾. Of particular interest is the performance of the perfluoro ether (Krytox 143AB oil and Krytox 240AB grease) relative to the silicones, mineral oil, and diesters. No instances of galling were observed with the perfluoro ethers, but several instances of galling were encountered with the silicones, diesters, and mineral oils. Although the lowest wear was measured with the FS1265 silicones, its viscosity was also the highest, which may have influenced the results. The wear with the perfluoro ether was less than that measured with the mineral oils and was considered acceptable since no instances of galling occurred. As shown in Table 10, similar results were obtained in slow-speed sliding tests using a ball-on-flat geometry⁽¹⁵⁾. The perfluoro ether (Fomblin Z25) had a friction coefficient tying the lowest of the five and a specific wear rate only slightly

TABLE 9. COMPARISON OF VARIOUS LUBRICANTS BASED ON WEAR OF 440C BLOCKS
IN LFW-1 RING-ON-BLOCK TESTS (FROM REFERENCE 14)

Common Name	Type	Viscosity (cs at °C)	Revs (K) ^a	Scar Width (mm) (mm)
Versilube F50	Methylchlorophenylsilicone	52/38	10.4 ^b	1.52 ^b
FS1265	Fluorosilicone	300/25	> 1 x 10 ³	0.76
SF1147	Methylalkylpolysiloxane	49/25	30.8 ^b	1.52 ^b
SRG40	Mineral	27/38	> 1 x 10 ³	2.03
Krytox 143AB	Perfluoroalkylpolyether	85/38	> 1 x 10 ³	1.78
L245-x	Diester	20/25	2.1 ^b	0.76 ^b
P-10	Diester + 5% TCP	15/38	> 1 x 10 ³	1.27
Andok-C	Mineral Oil Grease	--	40.8 ^b	1.27 ^b
Krytox 240AB	Perfluoroalkylpolyether Grease	--	> 1 x 10 ³	1.27

a. Revolutions to reach friction coefficient of 0.33 at 667N (150 lb) load, in thousands.

b. Test terminated because of galling; scar width at time of galling

TABLE 10. FRICTION AND WEAR RESULTS WITH EN31 STEEL BALLS SLIDING ON EN31 STEEL DISKS IN PRESENCE OF FIVE DIFFERENT LUBRICANTS (REFERENCE 15)

Lubricant	Vapour Pressure at 20° C (torr)	Description and Comments	Mean Friction Coefficient	Specific Wear Rate ($\text{m}^3/\text{Nm} \times 10^{-15}$)
Apiezon C	4×10^{-9}	Mineral oil with no additives. Used as a reference oil	0.20	0.93
BP 135	7.9×10^{-9}	Synthetic, tri-ester base. Boundary lubricant and anti-oxidant additives.	0.12	1.04
BP 110	3.7×10^{-8}	Mineral oil base, high viscosity. Refined to give low vapour pressure. Boundary lubricant additives.	0.13	0.41
KG80	$< 10^{-8}$	Petroleum base with boundary lubricant additive, Tricresyl-phosphate. Also anti-oxidant.	0.13	0.69
Fomblin Z25	$< 5 \times 10^{-12}$	Synthetic fluorinated oil. High density, low surface tension. High temperature/viscosity index. No boundary lubricant additives.	0.12	0.49

Notes:

1. EN31 is equivalent to 52100 steel.
2. Total load on three balls was 4N.
3. Sliding speed was 3×10^{-3} m/s.

higher than the lowest recorded (a mineral oil with boundary lubricant additives). Based on friction and wear results such as these, a much lower propensity to creep, and satisfactory flight experience, the perfluoro ethers have displaced the silicones in spacecraft applications⁽¹⁶⁾.

Besides the Krytox and Fomblin fluids, the Bray Oil Co. has produced a series of lubricants by further distilling and refining the Fomblin fluid base stock to produce Bray 815Z oil and 3L38RP grease, which have had flight experience in spacecraft mechanisms. The products are now available through the Bray Products Division of Burmah-Castrol, Inc. Braycote 601 is the new designation for the 3L38RP grease.

IDENTIFICATION OF SPECIFIC TRIBOLOGICAL COMPONENTS

SRS Technologies, as a major subcontractor on the Space Station Long Term Lubrication Analysis program, undertook the effort to identify the specific wear/bearing points in the overall Space Station and obtain design information to permit a review from a tribology standpoint. From the relatively broad range of assemblies, components, and mechanisms, a group of 14 target projects were identified as being likely to contain wear points of interest. A listing of these 14 target projects is presented in Table 11. A discussion of the efforts to attain detailed design data are presented on pages 22 through 29 of the SRS Technologies report in Appendix A. Specific tribological components identified in the survey are summarized in Table 2, pages 35 through 39 of the SRS Technologies report. While 72 specific components were identified, insufficient details were available at the time the survey was done to permit a review of their likely performance.

While sufficient design details were not forthcoming, the survey clearly identified the large alpha and beta joints to be critical components without comparable prior flight performance. Therefore, the bearings comprising these joints are worthy of more detailed examination. The design of the hundreds of other tribological contacts will range in difficulty from relatively routine (conservative designs with flight history) to the complexity and size of the alpha and beta joints. To assist the process, the preparation of a lubrication guide for use by the designers as a practical guide for choosing lubricants suitable for various types of applications appeared to be a useful and attainable goal. Therefore, the remainder of the program was structured to be:

- Phase II. Evaluation of Lubrication of the alpha and beta joints
- Phase III. The preparation of a lubricant guide for NASA and/or contractors to use in selecting and evaluating component lubrication techniques.

TABLE 11. SUMMARY OF TARGET PROJECTS IN SPACE STATION

● OMV Mechanisms
● Mobile Servicing Center
● SS Control Moment Gyro
● MDAC Berthing Mechanism
● GE Compressor
● Rockwell Rotary Joint Mechanism
● MDAC Rotary Joint Mechanism
● Sperry Rotary Joint Mechanism
● Space Station Alpha Rotary Joint Mechanism
● Space Station Propulsion Thruster/Storage Tank Assembly
● Space Station Solar Array and Solar Dynamic Concentrator
● OMV and Payload Mechanisms
● Antenna System Rotary Joints
● Health Maintenance Facility

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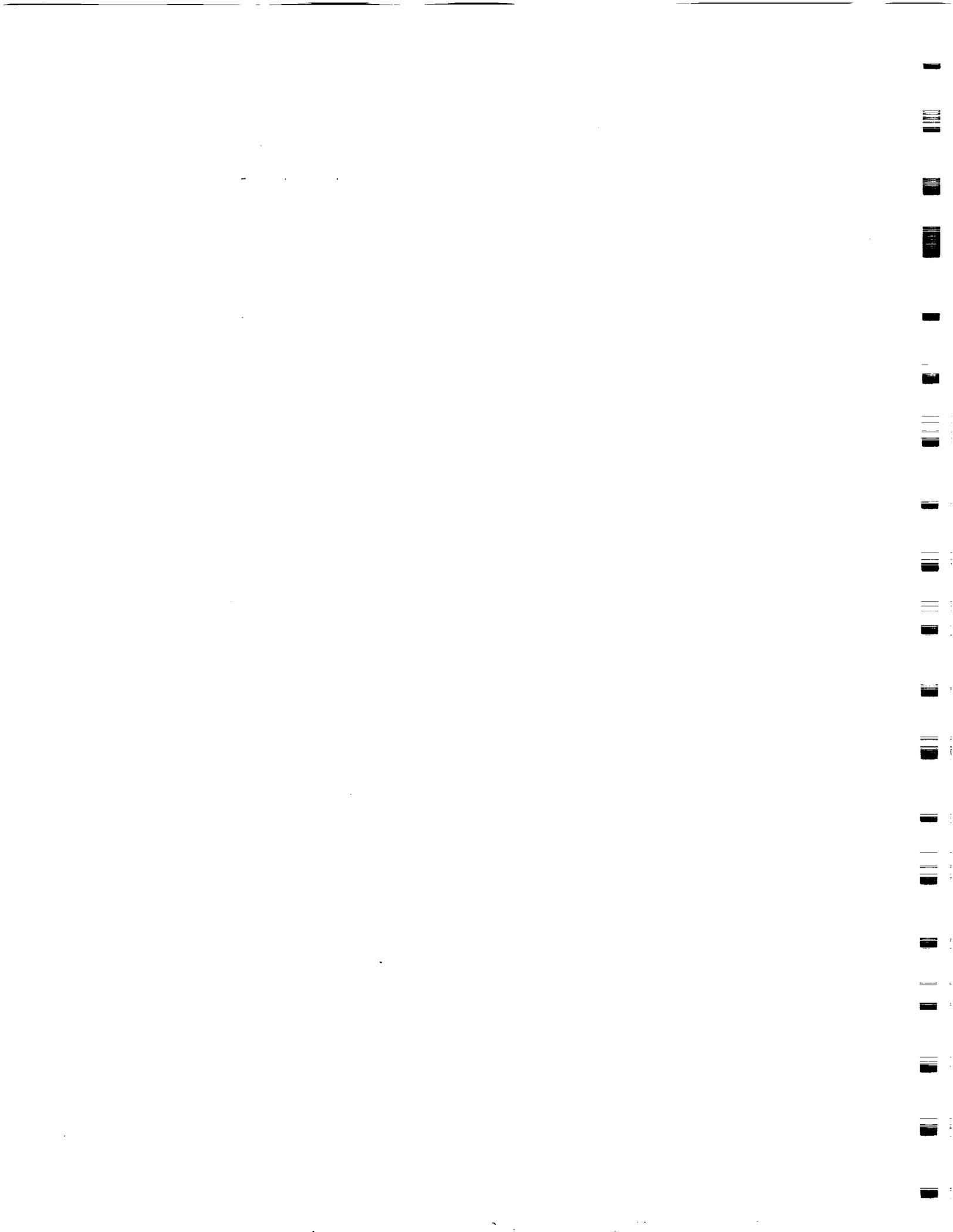
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APPENDIX A

SRS TECHNOLOGIES SUBCONTRACTOR REPORT



SRS/STG TR90-84

Space Station Long Term Lubrication Study Final Report

June, 1990

BY

EDWARD. E. MONTGOMERY

SRS Purchase Contract Number: G-9038
NASA Contract: NAS8-36655

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Foreword

This document was prepared by SRS Technologies under subcontract to Battelle Columbus Laboratories for work relating to the "Space Station Long Term Lubrication Study" for the Materials and Processes Laboratory at NASA's George C. Marshall Space Flight Center. The work was performed under the supervision of Mr. Keith Dufrane, the Battelle project manager and the NASA COTR, Mr. Fred Dolan. It presents a comprehensive final reporting of the Phase I activities of this project. The study was performed by SRS during the period from August, 1985 to May, 1990. The SRS contract manager is Jay H. Laue of SRS Technologies' Systems Technology Group, Aerospace Systems Directorate, in Huntsville, Alabama. The SRS project leader is Edward E. Montgomery. Other significant contributors at SRS included:

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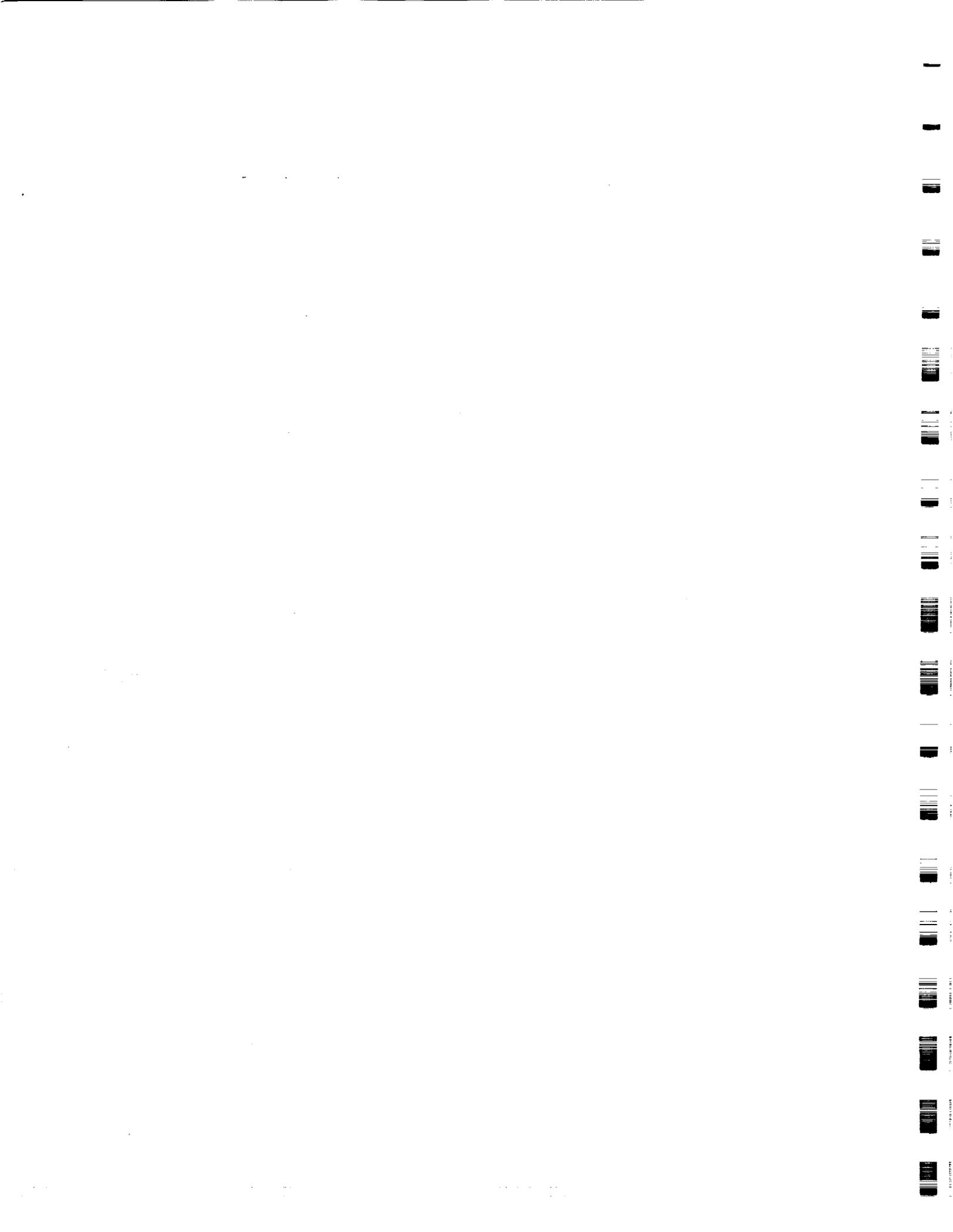
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1.0 INTRODUCTION

Under subcontract to Battelle Columbus Laboratories (SRS purchase contract G-9038) SRS performed selected activities in support of the "Space Station Long Term Lubrication Study" (NAS8-36655), a contract effort with the Materials and Processes Laboratory at MSFC/NASA. Mr. Fred Dolan is the NASA COTR, Mr. Keith Dufrane is the BCL Project Leader, and Mr. Edward Montgomery is the SRS Project Leader.

The Space Station Long Term Lubrication Study (LTLS) is part of the Advanced Development Program (ADP) within the Space Station Program. In fiscal Year 1985, the ADP was initiated to accelerate the development of a series of new technologies. Its objectives were to enhance the performance of the Space Station, reduce the life cycle cost during the operations phase, and reduce the risk that might be encountered with new technologies during the development phase.

1.1 Background

Work on the contract began on August 1, 1985. The project was scheduled to be conducted over a 3-year time frame in two phases. Phase I was a preliminary analysis conducted in parallel with the preliminary design phases of the Space Station program. Phase II was to be a more detailed analysis conducted during Phase C/D when the design became more established. Phase I was to conclude 20 months into the study (April 1987) and be based primarily on design data packages planned to be released along with the Space Station Phase B SDR in September 1986. However, the Space Station program schedule began to slip significantly. In addition, major configuration changes continued to occur. As a result, the design information was not maturing to a level sufficient to support the kinds of analyses planned in the LTLS. An interim approach was taken in which the LTLS utilized preliminary design data resulting from other ADP programs in lieu of mainstream SSF design review packages (which were not available). The problem became evident as early as October 1986 when it was first addressed in monthly progress reports to the government. In April, 1987, at the originally planned date for conclusion of the phase I activities, effort was suspended on the study to allow the Space Station program schedule to catch up. Unfortunately, further SSF slippages and major redesign efforts continued to keep the Long Term Lubrication Study dormant through the original LTLS contract end date in August, 1988.

In October, 1988, an SRS representative attended a Space Station Space Environmental Effects Data Exchange meeting at MSFC which included a lubrication concerns topic on the agenda. Otherwise, no effort was expended on the contract by SRS after April, 1987 until April, 1990. At that time, NASA extended the LTLS contract schedule and, subsequently, a new subcontract was negotiated between SRS and BCL. The scope and deliverables of the renegotiated effort were reduced from previous subcontract. During the renegotiations, it was mutually agreed that one of the previously established deliverables, a magnetic tape containing data from SRS survey results for use by Battelle in performing tribological assessments in Phase II, would no longer be a significant benefit to BCL or NASA. The remaining deliverable for SRS would be this final report.

1.2 Objectives/Approach

The overall objective of the Space Station Long Term Lubrication Study was:

"to assure NASA that all Space Station mechanical, electrical, and electromechanical equipment will function as intended for their required lives."

The goals of the study were to document and evaluate each wear point on the Space Station and assist in the improvement/resolution of suspected excessive wear or high maintenance areas. The prime duties for SRS were to gain access to design data for each wear point and assemble, organize, and document tribological data for analysis and evaluation. Battelle's prime duties were to analyze and evaluate each wear point and report on the estimated life of the hardware and the probability of success.

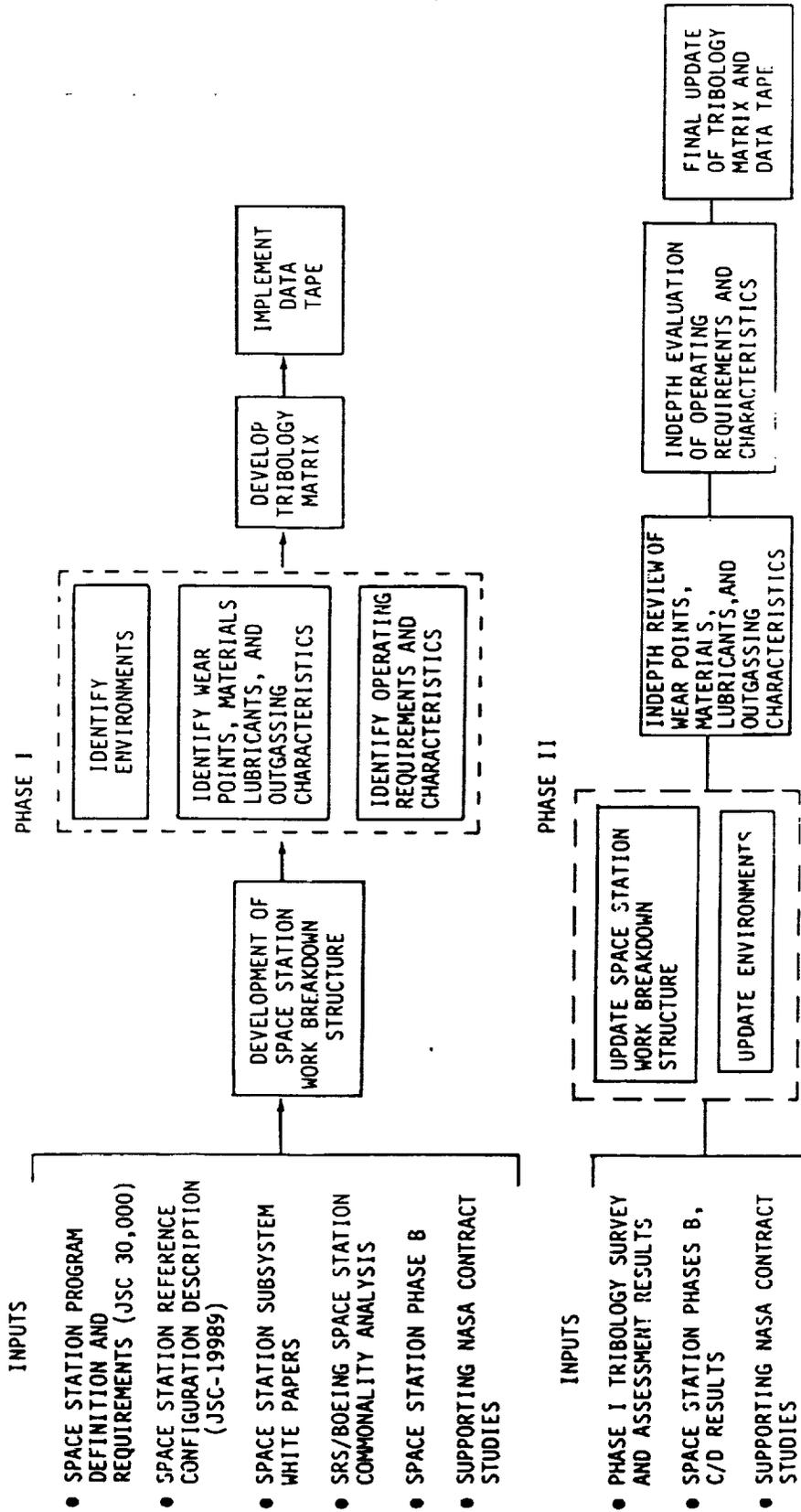
The Phase I approach established for the SRS activities included five major tasks:

- 1.2.1.1 Develop a system/component work breakdown structure for the Space Station to identify major moving mechanical assemblies.
- 1.2.1.2-4 Identify wear points and operating characteristics as to materials and surface treatments, lubrication scheme, operating characteristics, and environments.

- 1.2.1.4 Indicate the location of potentially contamination-causing fittings, lubricants, and bearing materials.
- 1.2.1.5 Develop a detailed tribology matrix containing the synthesized data to support the tribology assessment.
- 1.2.3.3 Implement a survey data tape on the NASA/MSFC computer system to include the results of the tribology survey and assessment.

Phase II tasks for SRS were originally planned to accomplish an update of these same tasks after the release of Space Station Phase B SDR data packages and again near the Phase C/D Space Station Preliminary Design Review. Figure 1 illustrates the logic flow of this approach. SRS deliverables were to include a comprehensive work breakdown structure for the Space Station identifying major moving mechanical assemblies (MMA), a complete list of operating characteristics, materials, and environments for each identified MMA used to support tribology assessment, an identification of potential contamination sources on Space Station (location and distribution of greases and lubricants), a data tape containing the results of the tribology survey and assessment, and an identification of high priority areas for further long-term lubrication study.

SPACE STATION LONG-TERM LUBRICATION APPROACH (SRS)

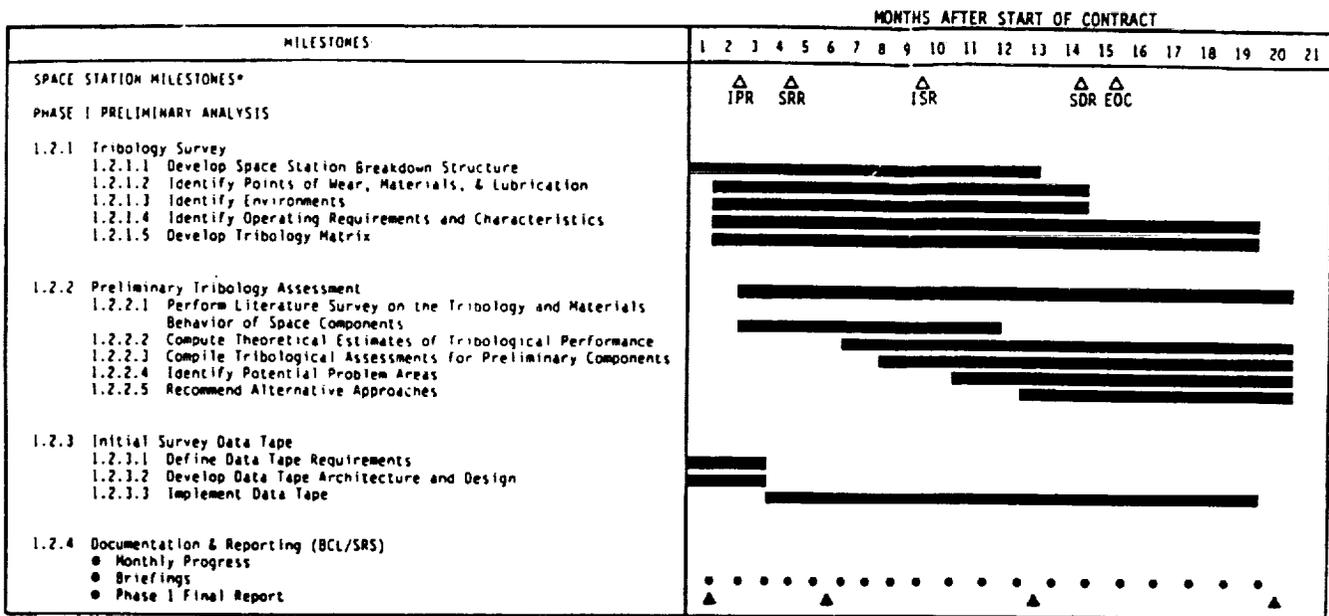


1.3 Project Schedule

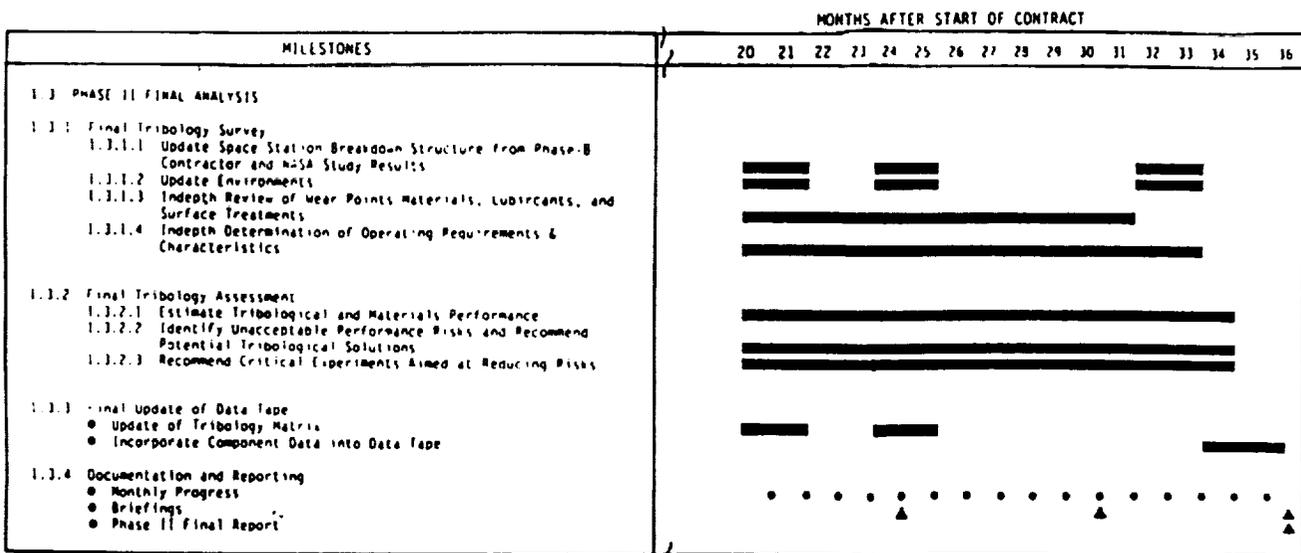
The original schedule for Phases I and II of the Space Station Long Term Lubrication Study and major milestones are shown in Figure 2. Also shown on the figure are the dates of major Space Station Program milestones as planned at the start of the study. Close coordination was planned between the activities of this study and the progress of design activities. Figures 3 through 6 show the expected publication dates for major outputs of the four major space station work packages.

Figure 2. Planned Phase I and II Project Schedules

SCHEDULE PHASE I



SCHEDULE PHASE II



* Assuming Long Term Lubrication Contract Starts 3 Months After Space Station Contract Start Date

FIGURE 1-3. PROPOSED MILESTONE SCHEDULE FOR LONG TERM LUBRICATION STUDY

1985						1986											
April	May	June	July	Aug	Sept	Oct	Nov	Dec	Jan	Feb	March	April	May	June	July	Aug	Sept

Phase B Milestones

△ CSD

- WP-01: MSFC

△ DP 1.1
● Reference configuration analysis results

● Common module commonality analysis preliminary results

△ DP 1.2

- Module interface analysis - preliminary module pattern analysis
- Update common module commonality analysis
- Subsystem trade options and preliminary analysis

△ DP 1.3

- Conceptual design layout
- Preliminary maintenance/Maintainability analysis
- Preliminary structural loads/Dynamic analysis
- Commonality assessment

△ DP 1.4

- Updated conceptual design layouts
- Preliminary interface requirements definition study results
- Subsystem analyses results
- Baseline system recommendations
- Proposed specification tree
- Updated maintenance/maintainability analysis

△ IRR

△ DP 1.5

- Baseline system and interface requirement

△ DP 1.6

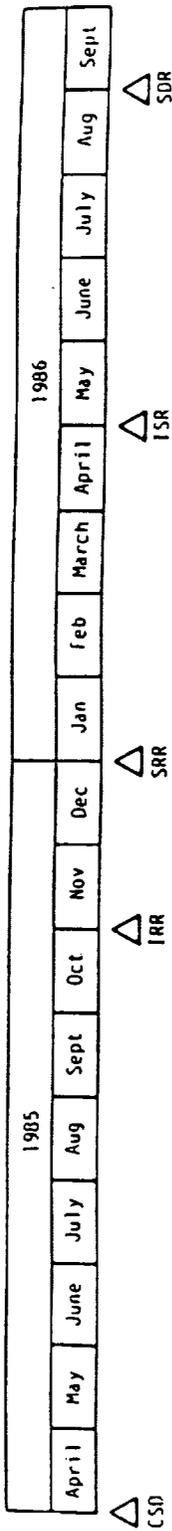
- CEI updates based on IRR/SRR reviews and adjustments
- Definition of sizing parameters and envelopes for all systems, subsystems, major sub-assemblies and modules
- System schematics
- Final trade study results

△ DP 1.7

- Preliminary ICD update
- Preliminary design documentation updates

△ SDK

△ ISR



- WP 02 JSC
- △ DP 2.1
 - Reference configuration analysis results
 - Preliminary trades study results
 - Conceptual design lay-outs and/or approaches for WP Hardware/Software elements
- △ DP 2.2
 - Overall configuration(s) sizing parameters and general arrangement of components and major assemblies for Space Station
 - Trade study results
 - Preliminary structural loads/ Dynamic analysis results
- △ DP 2.3
 - Preliminary overall architecture analysis
 - Preliminary IDMS, system sizing and requirements definition
 - Update conceptual design layouts for all WP elements
- △ DP 2.4
 - Subsystem and major assembly cost data
 - Preliminary LCD's for the Space Station major assemblies, modules, and subsystems
 - Preliminary baseline system recommendations
 - Trade study results
 - Preliminary specification tree
- △ DP 2.5
 - Baseline system interface requirements
- △ DP 2.6
 - Definition of sizing parameters and design envelopes for all systems, subsystems, major sub-assemblies, and modules
- △ DP 2.7
 - Draft descriptions of LCD's for subsystems, major assemblies, and modules

1985						1986											
April	May	June	July	Aug	Sept	Oct	Nov	Dec	Jan	Feb	March	April	May	June	July	Aug	Sept
△ CSD						△ IRR		△ SRR				△ ISR					△ SRR

- WP 03 GSFC

△ DP 3.1

- Reference configuration analysis results
- Preliminary trade studies
- Conceptual design lay-outs and/or approaches for WP Hardware/Software elements consisting of schematics, sketches, brief narratives

△ DP 3.2

- Overall configuration(s) sizing parameters and general arrangement of components and major assemblies for platforms and service center
- Trade study results
- Preliminary structure loads/dynamic analysis results

△ DP 3.3

- Supporting analysis results (configuration, structural loads)
- Update conceptual design lay-outs for all WP elements
- Fluid systems analysis

△ DP 3.4

- Integrated system overall configuration analysis
- Subsystem and major assembly cost data
- Preliminary ICD's for platforms, major assemblies, lab module, subsystems
- Preliminary baseline system recommendations
- Trade study results

△ DP 3.5

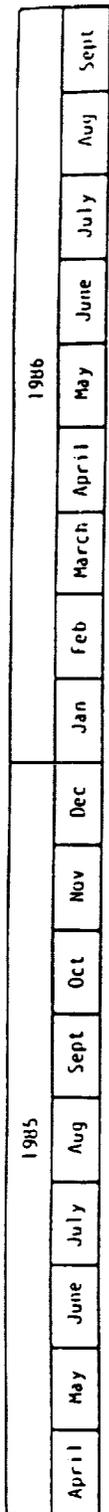
- Baseline system interface requirements

△ DP 3.6

- System requirements update based on IRR/SRR
- Definition of sizing parameters and design envelopes for all systems, subsystems, major sub-assemblies, and modules
- Trade study results

△ DP 3.7

- Draft descriptions of ICD's for subsystems, major assemblies and modules
- Draft of CCI specifications



△ CSD

● WP-04 LeRc

△ DP 4.1

- Reference Space Station and platform configuration analysis results
- Preliminary trade studies
- Preliminary definition of the overall architecture of solar dynamic and photovoltaic-based power systems

△ DP 4.2

- Overall sizing parameters and general arrangement of WP components for Space Station and platform
- Update DP 4-1
- Trade studies

△ DP 4.3

- Detailed architecture of power system within modules, attached payloads, and subsystem attach areas
- WP trades update
- Update conceptual design for all WP elements

△ DP 4.4

- Integrated overall architecture analysis
- Preliminary subsystem cost estimates of solar dynamic and photovoltaic-based power systems
- Preliminary definition of standard power system parts list
- Trade study results
- Proposed specification tree

△ DP 4.5

- Baseline system interface requirements

△ DP 4.6

- System requirements updates based on IRR/SRR reviews and adjustments
- Definition of sizing parameters and design envelopes for all systems, subsystems, subassemblies, and modules
- Initial ICD's
- Trade study results

△ DP 4.7

- Draft of ICD's for subsystems, major assemblies, and modules

△ SIDR

△ ISR

△ SRR

△ IRR

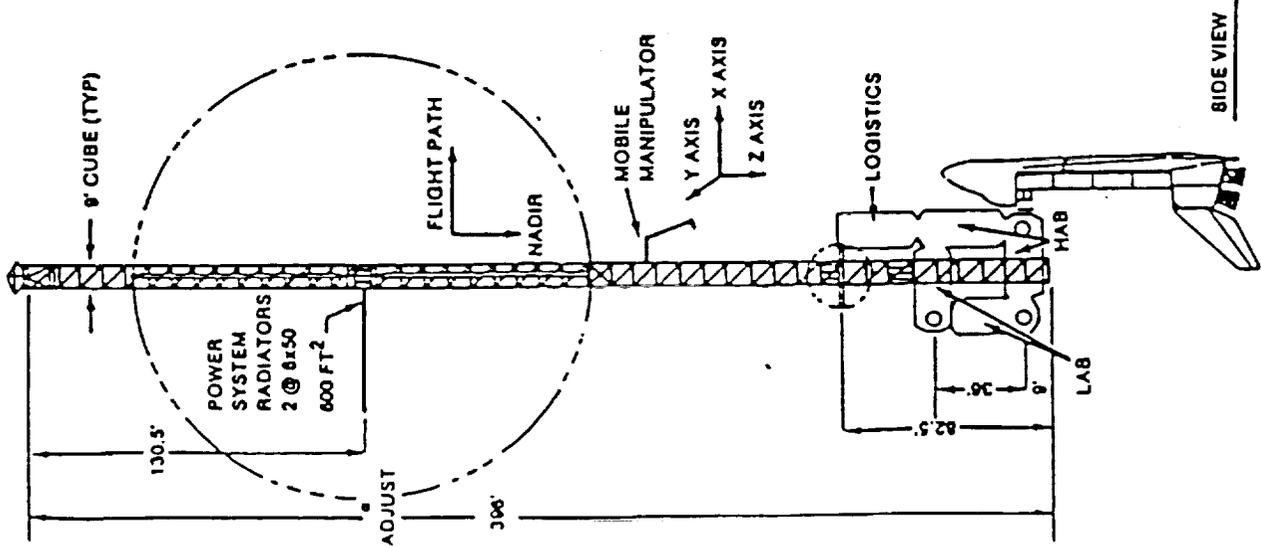
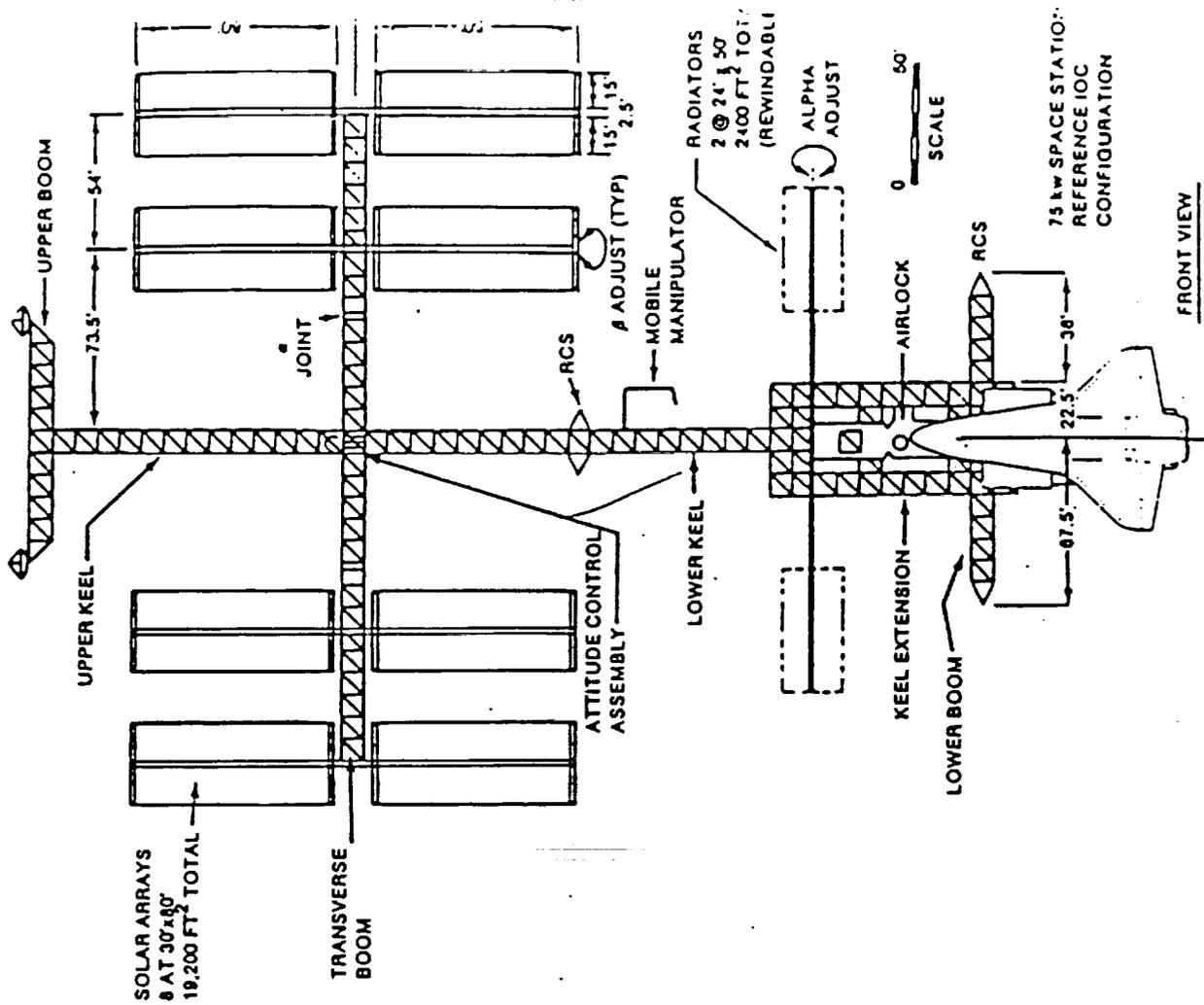
2.0 TECHNICAL ACCOMPLISHMENTS

During the Phase I efforts, a number of achievements occurred despite the difficulties in obtaining solid detailed design information. Some of the results are no longer applicable due to differences in the station configuration at the start of the study and now, the "75 kW Space Station reference IOC" concept represented in Pre-Phase B White Papers document and the current Space Station Freedom Configuration. Also, atomic oxygen effects were just coming under close analysis through data obtained from shuttle experiments. Since the time of these analyses, the understanding of physical mechanisms and orbital environments has increased significantly and may advance further in the near future as the results of the recently recovered Long Duration Exposure Facility are analyzed and documented. The results of the task activities are presented in greater detail in the following sections of this report. Included are discussions of the development of a Space Station Work Breakdown Structure, environment identification, wear points, materials, lubricants, & outgassing, operating requirements and characteristics, and the data tape.

2.1 Development of Space Station Work Breakdown Structure

A comprehensive listing of the Work Breakdown including all major system, subsystems, equipment, was developed and used as the basis for identifying wear points and critical lubrication concerns. The Space Station IOC reference configuration shown in figure 7 served as the baseline configuration for this analysis. This configuration is based on a set of deployed linear trusses constituting the keel and booms to which the modules, subsystems, and equipment are connected. It contains numerous mechanisms and joints that provide interfaces with and support to external payloads and equipment for various experiments and servicing functions (e.g., satellite, orbital/space transfer vehicle, and orbital maneuvering vehicle servicing). Phase I of this program began about the time Phase B of the Space Station program began in which there were still a number of parallel contractors involved in each work package. SRS obtained design data products from the Martin Marietta and Boeing Work Package efforts early in the study and compared these with the reference configuration. Some variances occurred as can be expected, especially at the lower levels. It should be noted that there existed an official Space Station Work Breakdown Structure for the program, the top levels of which are shown in figure 8. For the purposes of this study it was necessary to modify the program WBS which includes functional elements (e.g., safety, reliability, etc.). The WBS for this study was intended as a cataloging tool to promote a complete identification of wear points in space hardware and therefore only includes the breakdown of on-orbit hardware items. Much similarity exists in the two items. Figure 9 depicts the difference in those structures at the top level. Both agree at the program level and the project level definition was retained in anticipation of continuing further with WP-1 than the others.

The basic concept of the hierarchical network as shown in figure 10 was used to identify the Space Station breakdown structure levels and contents. The use of this system to catalog and describe the large number of equipment hardware items in the space station provided a systematic approach which was easily automated for computer analyses of the interrelationships of all elements within a breakdown structure level. Figure 11 illustrates the manner in which each level is subdivided to provide a detailed listing of components and parts for identifying wear points. The WBS structure levels were defined along the approach NASA uses.



Preliminary Space Station Work Breakdown Structure

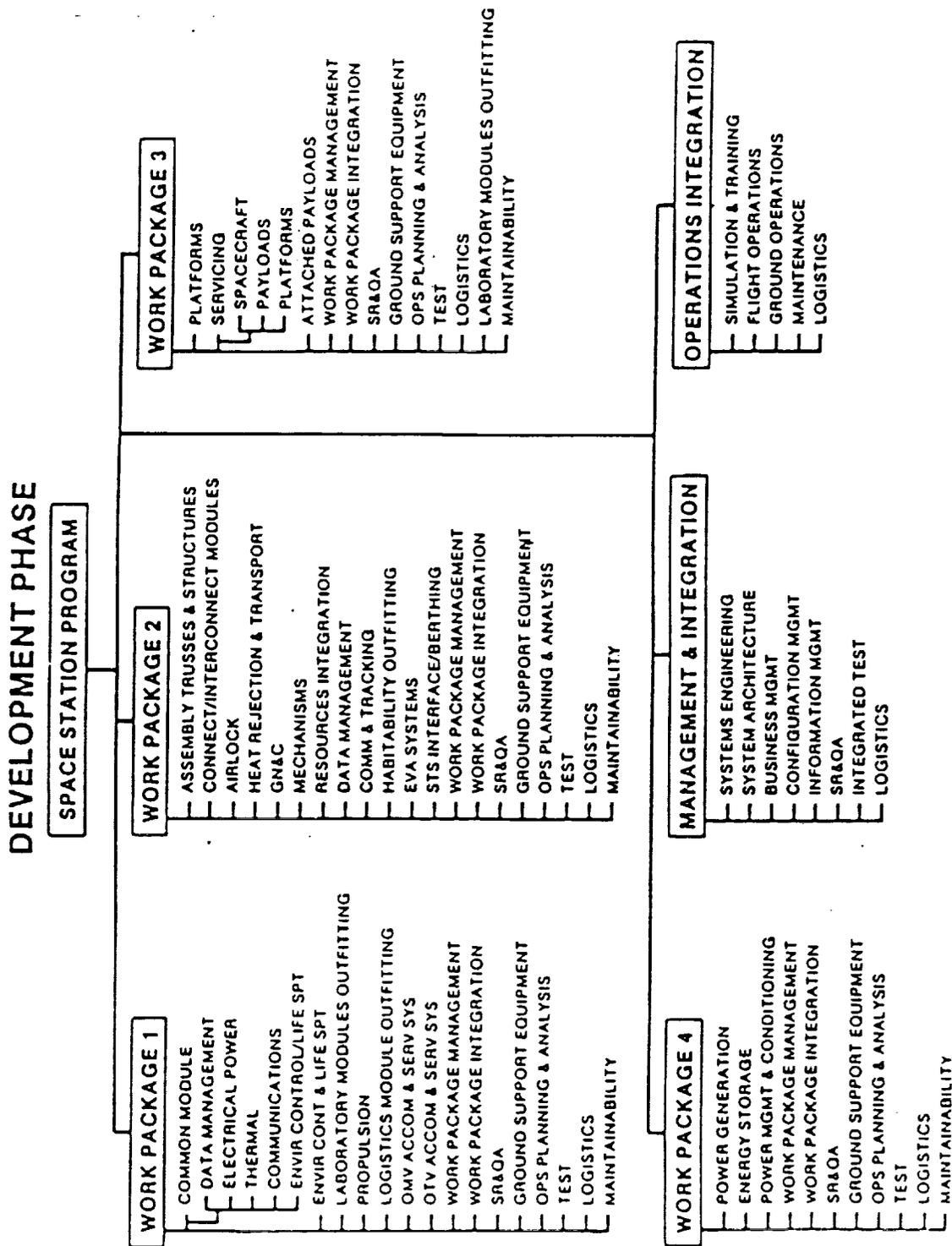


Figure 9. Redefined LTLS Work Breakdown Structure

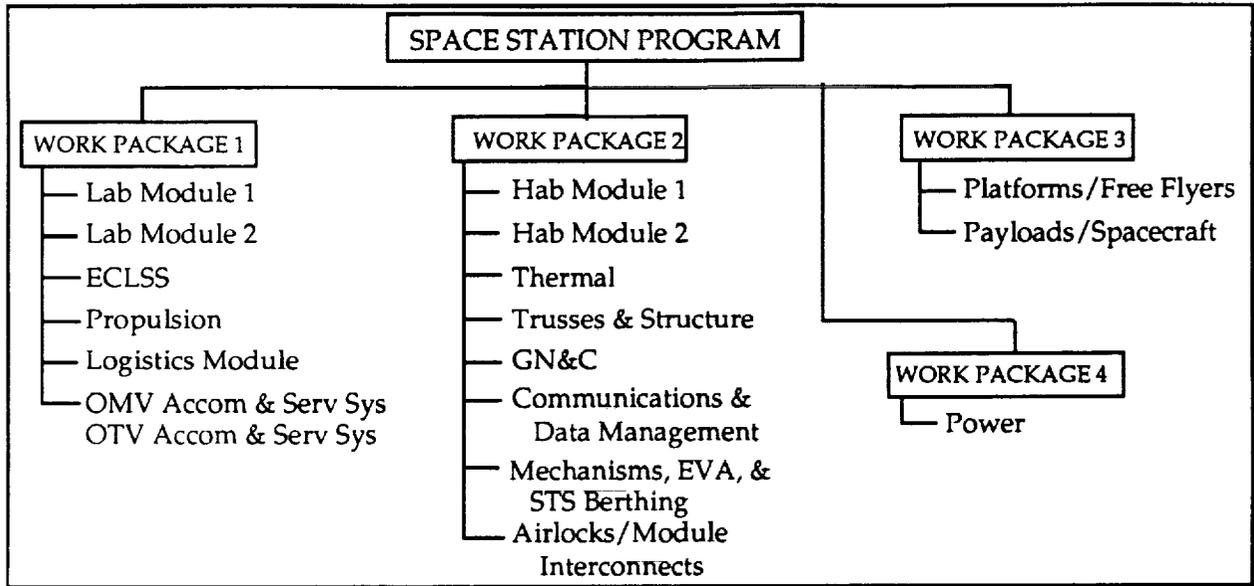


Figure 10. Hierarchy for Development of SSF Systems, Subsystems, and Component Listing

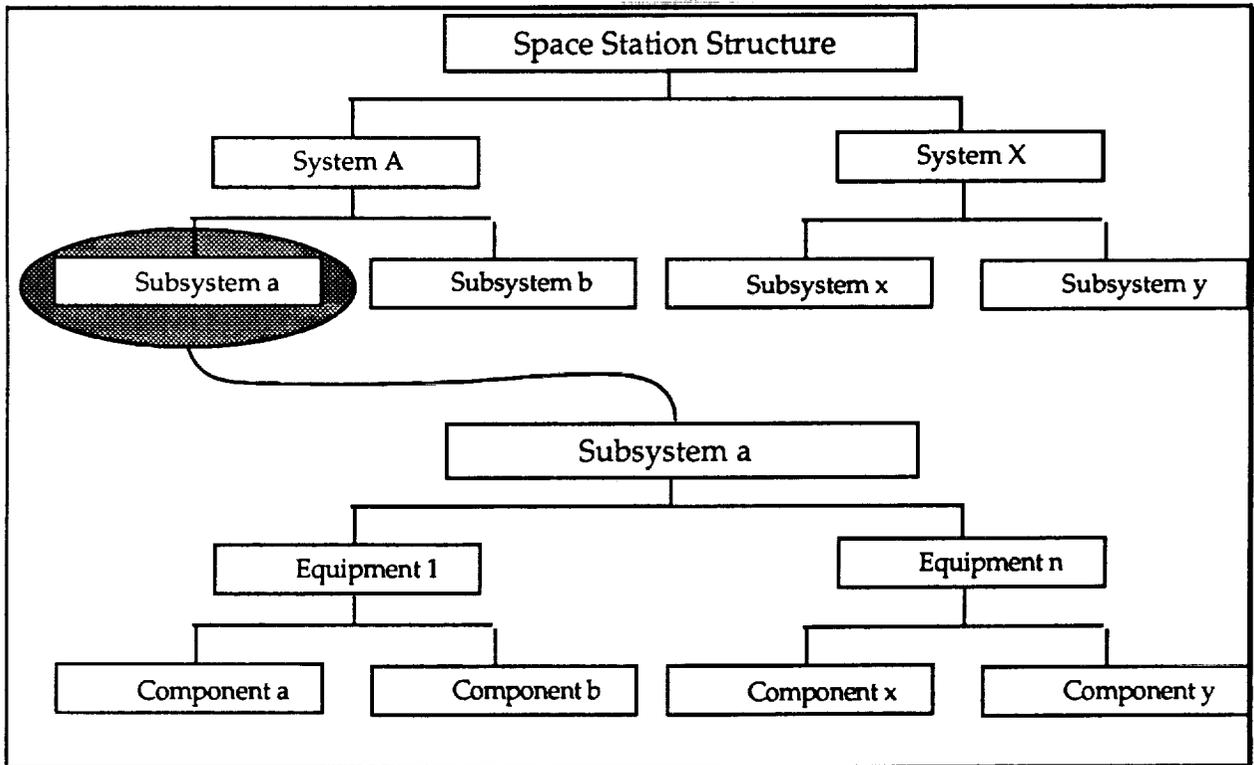
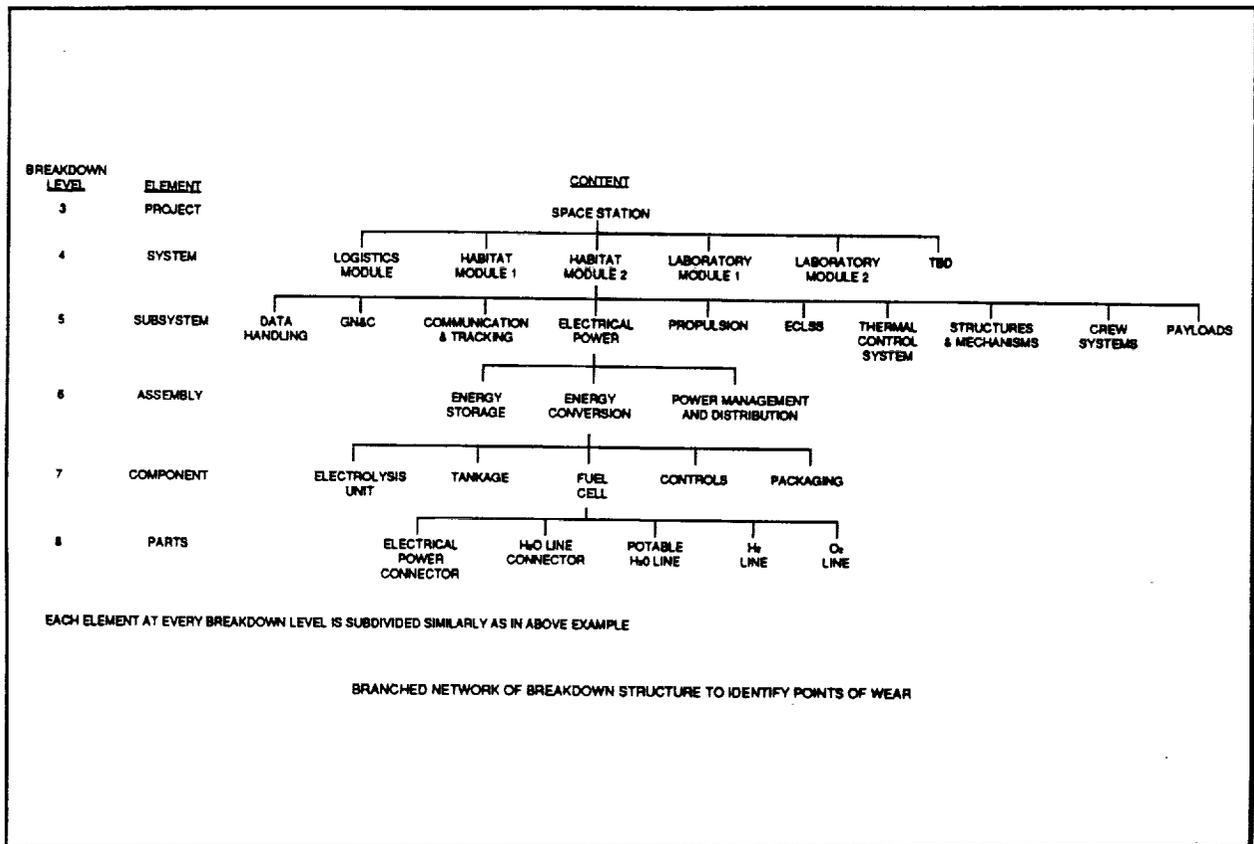


Figure 11. Branched Network of Breakdown Structure to identify Points of Wear



The development of the LTLS WBS began at the program level 2 and proceeded with analysis of each element at a level to identify major pieces of equipment at that level. The depth to which the tree could be defined was limited by the availability of design information (related to the maturity of the design) and study resources. The "Space Station Reference Configuration Description" document (JSC-19989) addressed the growth of the manned and unmanned station concepts and contained a configuration evaluation and subsystem definition of the IOC configuration. Being the most comprehensive and programmatically substantial source of reference information at the time the study began, it was used as a guideline for the definition of levels 3-6. Similarly, a companion volume, the "Space Station Subsystem White Papers" document (JSC-20054), was used in the definition of the 7-8 levels of the WBS. Even so, it was not possible to complete the WBS for all work packages completely down to the 7th or 8th levels. Instead, the

structure was defined to the 4th and 5th level, which are the System/modules and subsystems, then an evaluation of the elements was performed to determine the major Moving Mechanical Assemblies (MMAs). These were selected in a process of evaluation in which each element was assessed against the selection criteria shown in figure 12. Those elements which registered in the largest number of criteria categories, for which significant design data existed, and were Work Package 1 elements were prioritized and selected for further WBS level definition and evaluation.

Figure 12. Criteria for Ranking Risks or Probability of Failure

Code	Selection Criteria
0	SELECTION CRITERIA NOT APPLICABLE.
1	OPERATES UNDER HIGH LOADS/CONTACT STRESSES, FREQUENT OR CONSTANT. MOTION, ENVIRONMENTAL EXTREMES, AND/OR HIGH DUTY CYCLES.
2	POTENTIAL SOURCE FOR CONTAMINATION RESULTING FROM PREMATURE FAILURE.
3	MISSION OR LIFE CRITICAL IF PREMATURE FAILURE OCCURS.
4	UNIQUE PRELIMINARY DESIGN CONCEPT/NEW TECHNOLOGY INVOLVED.
5	WILL OR LIKELY TO REQUIRE MAINTENANCE/REFURBISHMENT.
EXAMPLES:	
00000:	No elements of selection criteria applicable.
10000:	WBS component only operates under high loads/contact stresses, frequent or constant motion, environmental extremes, and/or high duty cycles.
02340:	WBS component is a potential source of contamination resulting from premature failure, is mission or life critical if premature failure if premature failure occurs, and involves a unique preliminary design concept or new technology.

2.2 Environment Identification

Establishment of the environments for each wear point was required for the assessment of performance and life characteristics of the particular surfaces of interest in the identified wear points. In general, environments enclosing surfaces in relative motion can be different depending on the duty cycle of the mechanism. For example, when the mechanism is not operating, the environment may be at a different temperature and pressure than when the mechanism is operating. Mechanisms at rest may, however, be under static load and, depending on materials and environments, failures such as long term stress corrosion may occur. If high loads and/or high relative speeds are present such as in the contacts of high speed rolling bearings, temperature in the contacts will be vastly different from those for static conditions. These thermal environments and temperature cycling set stringent requirements on long term lubricants and the ability of the lubricants to provide the necessary elastohydrodynamic for proper lubrication at operation temperature. In general, the life of these contact surfaces is dependent on the elastohydrodynamic conditions in the contacts which is a complex function of load levels, speeds, and operating environment.

In some specific applications, such as the Space Station refrigeration systems, the lubrication system for refrigerant compressors may require isolation to prevent lubricant contamination of evaporator and condenser surfaces. In some designs this may involve dynamic seals in a refrigerant and high temperature lubricant environment. These environments must be identified and characterized to support wear and life assessments of these components. Other mechanisms, such as external fluid rotary joints and docking mechanisms, will have external surfaces exposed to the space environment for long periods of time. Lubricants used for these applications must therefore be compatible with these environments. The assessment of environmental effects on contacts and/or wear point performance, durability, and life requires a definition of these environments over the entire duty cycle of the mechanism and/or component.

The objectives of this effort were to identify the environment for each wear point over the duty cycle of the mechanism, characterize the environment for assessment of wear point durability and life, provide comprehensive rationale to support findings, data base, and documentation requirements. Environments over the complete duty cycle were determined for each Space Station MMA component.

These environments were then characterized to determine the specific environmental components that affected wear point performance and durability.

Many components, such as thermal control coolant pumps, essentially operate continuously at relatively steady loads. Other components, such as docking mechanisms and latches, may not operate over several times per year. Mechanisms such as Space Station Solar Array actuators and drive mechanisms will operate several times per orbital period. These are examples of mechanisms that have relatively steady environments and large fluctuations in environments depending on duty cycles. Surfaces exposed to the orbital environment (latches, docking probes, etc.) will be exposed to many elements that could cause surface degradation leading to loss of performance and decreased durability. The space environmental effects most likely to affect materials and lubricants in Low Earth Orbit are:

- Vacuum Outgassing (1×10^{-6} to 1×10^{-7} Torr (per specification SP-R-0022A)
- Solar Radiation
 - Proton - 2.9×10^3 rad (c), Energy >100 keV to 2.2×10^3 rad (Si)
 - Electron - 7.5×10^5 rad (C), Energy >40 keV to 6.8×10^5 rad (Si)
 - Ultraviolet - 55,000 Sun Hours
- Atomic Oxygen
 - Concentration - Approximately 1×10^9 atoms/cm³ @ 290 nmi. altitude
 - Exposure - perpendicular to Flight Path
 - 2.36×10^{15} atoms/cm²/sec
 - 3.5×10^{20} atoms/cm² in 41.0 hours
- Micrometeoroids - 8200 impacts/m² (mass < 10-12 gm)
- Meteoroids & Debris - 4 impacts/m² (diameter > 0.01 cm)
- Thermal Cycles - 58,500 over 10 years

The space vacuum has varied effects on materials depending on the material characteristics. For example, it has a beneficial effect on the fatigue life of alloys susceptible to stress corrosion (removed or absorbed molecules eliminating the electrolyte). Vacuum and thermal effects can cause removal of surface gases from materials with detrimental effects. Losses of absorbed gases for materials in sliding surfaces can cause appreciable changes in friction coefficients. Lubrication characteristics may also change significantly when exposed to space vacuum over long periods of time. A lubricant's volatility is a function of its vapor pressure, and vapor pressure is related inversely to molecular weight. In general, lower weight

fractions of a lubricant will be lost through vaporization before the heavier constituents. This does not necessarily mean that those with very low vapor pressure and evaporation rates will give the longest life in vacuum service. The only conclusive means in determining lubricant service life in vacuum is by experimentation. Long term tests of grease type lubricants are underway at MSFC and will provide valuable information for the assessment of long term vacuum effect on these lubricants.

In low earth orbit the principal source of electromagnetic radiation is the sun. Solar radiation covers the wave length region from around $0.01 \mu\text{m}$ to greater than $10^4 \mu\text{m}$. This region includes low energy gamma ray, x-ray, UV, visible, infrared, and microwave photons with an energy density of 1353 W/m^2 . In developing the solar environments for specific components the location of the component relative to the sun and earth is important as well as shading and/or blockage from adjacent structures. For estimates of material degradation due to sun exposure, total exposure times will be estimated based on orbital time averages of solar exposure for each mechanism. Solar flux distributions for outer surfaces of the Space Station will be developed by Phase B contractors for thermal design of SSF. This will be the principal source of data for this environmental information. In specific cases where data may not be available, SRS has the capability to develop this information.

The trapped particles of the South Atlantic Anomaly are essentially electrons and protons trapped in magnetic flux lines and reach a low point at about 30 degrees inclination or greater, there will be approximately five passes through this anomalous area each 24 hours. Material degradation occurs through ionization of the atoms or molecules by the electrons and energy deposition in the materials by protons.

Atomic oxygen effects on materials and surface degradation is a strong function of the direction of the surface with respect to the velocity vector. The greatest amount of degradation occurs on the forward surface of the STS Orbiter perpendicular to the velocity vector. Therefore, atomic oxygen fluxes must be estimated based on the relative location of the surface of interest. Materials experiments have been flown on many STS flights and data reported in the literature. The recently retrieved Long Duration Exposure Facility should provide some very relevant information from AO experiment results.

During the LTLS study, the responsibility for leading space station propulsion system design was being transferred to JSC. Mr. Walt Karakulko at NASA/JSC was

contacted and provided these guidelines on operating characteristics of the space station thruster/storage tank assembly.

gaseous fuel: <3000 psi; (20 deg. F.)<T<(150 deg. F)

liquid fuel: 400-500 psi; (20 deg. F.)<T<(150 deg. F)

2.3 Identification of Wear Points

In this section, the activities performed in pursuit of the tribology survey data and the MMA's identified in this process are discussed. This first subsection summarizes these activities and includes a list of the MMA's selected. Succeeding subsections discuss each MMA separately .

2.3.1 Summary

From a relatively broad range of assemblies, components, and mechanisms, a group of items were selected for focus in the evaluation and assessment activities. For these items, a contacts were established, usually with the NASA technical monitor and the contractor (or subcontractor) performing technology studies or advanced developments for the items. In many cases SRS met with project leaders and discussed lubrication concerns directly with the individuals involved in the studies. This was made possible in most cases by scheduling splinter sessions in conjunction with SS Phase B Work Package Reviews and SS Advanced Development Program Reviews held at MSFC. The list of areas below comprise this set of target projects.

Collection of data and survey of lubrication aspects of:

- o OMV Mechanisms,
- o Mobile Servicing Center,
- o SS Control Moment Gyro,
- o MDAC Berthing Mechanism,
- o GE Compressor,
- o Rockwell Rotary Joint Mechanism,
- o MDAC Rotary Joint Mechanism,
- o Sperry Rotary Joint Mechanism,
- o Space Station Alpha Rotary Joint Mechanism,
- o Space Station Propulsion Thruster/Storage Tank Assembly,
- o Space Station Solar Array and Solar Dynamic Concentrator,
- o OMV and Payload Mechanisms,
- o Antenna System Rotary Joints, and
- o Health Maintenance Facility.

Lubrication issues, although of concern to all such projects, were not the main goal of most of these projects. Typically, the objectives were to assess relevant technologies in areas like mechanisms, power generation, or LOX/LH2 propulsion concepts and produce proof-of-concept prototypes. Therefore, the effort which these projects could put forth in coordinating with the LTLS was limited. In all cases, SRS, in conjunction with the COTR for this study, Mr. Fred Dolan, sent written requests for pertinent information and provided lubrication survey forms. In several instances, the projects were able to make an assessment and provide very useful feedback. In other cases, the projects provided data products (technical reports, concept descriptions, design drawings, etc.) which allowed SRS to perform assessments for lubrication concerns connected with the item. Also, through alternate sources and basic research, enough information was obtained to make some evaluation. Some projects never reached a level of maturity to support definition of all the the structural, mechanical, lubricant selection, outgassing information necessary for a complete survey of the concern. Looking forward to updates planned for later in the LTLS project, SRS retained these items in the tribology data base with as much information as was available.

In addition, SRS continued to track the the global program to monitor for the identification of additional lubrication concerns/items as they became apparent. Those activities included acquisition and assessment for wear points in Boeing WP-1 and SS ADP Review Data, participation in Space Station Materials Sub-Panel Meeting at NASA/Langley Research Center, communications with MSFC, Phase B contractors, and ADP's to obtain environment and operating characteristics of MMA's, and coordination with ADP centers to request inclusion of the completed Survey Forms as part of Phase C/D PDR data requirements.

At the Langley Materials Sub-Panel Meeting, the issue of design maturity and the timeliness of LTLS study products was addressed. In response to the need for a tribology assessment for the Major Moving Mechanical Assemblies (MMA's) prior to their respective Critical Design Reviews (CDRs), the LTLS study approach was revised. The initial approach was based on making use of Design data published in Preliminary Design Review (PDR) data packages. However, this data has been neither complete nor mature enough to support a tribology assessment. It has been determined a more direct method of obtaining environmental and operating characteristics of the MMAs is needed if the tribology assessments are to be made before CDR.

To resolve this problem, the tribology survey forms previously developed in this study for in-house use, were sent to the responsible COTR's and contractors prior to design reviews. Letters of explanation and Survey Forms were sent to:

- o Sperry -Alpha and Beta Rotary Joints
- o Boeing -Gimballed Radiator and Two Phase Thermal Bus
- o Grumman -Space Erectable Radiator and Two Phase Thermal Bus
- o General Electric -Compressor for SS Refrigeration system
- o MacDonnell Douglas -Berthing Mechanism

In addition, a letter and survey form encouraging their inputs to this study were sent to David Thomasson of the Huntsville Boeing office. Boeing Aerospace Company is an MSFC WP-1 Prime Contractor. Also, a complete set of the red-books containing the presentation materials from the Space Station Advanced Development Program Review were obtained and reviewed for information indicating possible Major Moving Mechanical Assembly candidates for further lubrication study in this program.

2.3.2 Orbital Maneuvering Vehicle & Mobile Servicing Center

John Forbes/MSFC, the manager for Orbital Maneuvering Vehicle (OMV) Mechanisms was contacted and agreed to follow through in providing wear point information towards the objective of developing a survey response. The OMV became an LTLS topic of focus when SRS attended the annual review of the Grumman Aerospace "Advanced Orbital Servicing Technology" effort at MSFC on June 24. No hardware design activities pertinent to this study were identified as part of that AOST study, but during the review suggestions were made to research the mechanisms on the Mobile Remote Manipulator System (MRMS) and the Orbital Maneuvering Vehicle (OMV). SRS followed up in the pursuit of lubrication aspects of the mechanisms on the Mobile Remote Manipulator System (MRMS) and the Orbital Maneuvering Vehicle (OMV) by contacting Luther Powell, then director of the MSFC Space Station Program Office. He provided SRS with a contact in the MRMS program-Brian Erb of the NASA Johnson Space Center. JSC was to develop the base with SPAR of Canada providing the manipulator arm. In initial discussions with Mr. Erb, he indicated project was still very much in the conceptual phase, but he would do some checking and call back on the availability of data. Later, Brian Erb

(also of the Canadian Research Council) was sent LTLS survey forms and forwarded them to appropriate technologists. There was no further response from that source. Additional discussions with Zack Thompson, Stan MacIntyre, and Tom Byrd of MSFC indicated there was some potential for lubrication issues associated with fluid, electrical, and payload quick disconnects. In a follow-up to that meeting, Tom Byrd of NASA/MSFC indicated three design areas with possible lubrication concerns and agreed to supply pertinent data on:

- (1) 3-point latch mechanism for Space Telescope,
- (2) Fluid coupling for propulsion system, and
- (3) Payload gimbaling.

Also, on July 16, 1986, SRS sent a letter to Mr. Jeffrey Corbin of Martin Marietta requesting contacts to provide data on OMV/OTV Accommodations items relating to docking mechanisms and pumps from their Phase B work:

2.3.3 Space Station Control Moment Gyro

Early conversations with Lewis Cook, COTR for this Bendix project indicated that a survey was underway for supporting mechanisms with an expected time of completion is two weeks. In the Attitude Control and Stabilization session of an ADP program review held later, Mr. Cook of MSFC presented the status of a Control Moment Gyro design activity. Parts of this system were currently on order for a CMG prototype. However, only limited design data pertinent to lubrication was available.

2.3.4 GE Compressor

Lockheed Missiles and Space Company studies of mechanical assemblies in the Thermal Storage Unit and Refrigerator System focused on the GE compressor. Mr. Rudy Trabanino of NASA/JSC agreed to forward a survey data form to the appropriate individuals at the General Electric operations in Houston. A survey was sent for the advanced development program they were pursuing on a Compressor for the Space Station Refrigeration system. Later, Mr. Trabinino requested additional survey forms to send to General Electric. During the project, GE Houston operations underwent a cut in workforce. The original request and form may have been lost.

2.3.5 Truss & Rotary Joint Mechanism

Discussions with Leon Smith of the McDonnell Douglas Astronautics Company identified Lockheed Missiles and Space Company as the provider of a rotary joint design as part of the Work Package 2 Phase B activity. However, not enough detail was then available to justify a survey. The competition sensitive nature of the design/configuration was expected to limit the availability of significant design information until mid to late 1987.

Lubrication survey data forms were provided to Mr. Harley Rockoff of Rockwell for their Rotary Joint Mechanism work. A second request was made as a result of an Space Station Advanced Development Program Review meeting held at LaRC.

Conversation with Fred Jankowski of NASA/MSFC indicated that little pertinent information to support a tribological analysis of the McDonnell Douglas Rotary Joint Mechanism was available at the ADP review. Detail drawings were only beginning to be prepared at the time.

John Gustasson of NASA/LaRC was contacted and agreed to send a package of documentation containing all available design information on the Space Station Truss and Joint Designs being sponsored by LaRC.

Mr. Obie Bradley of the NASA Langley Research Center provided SRS with a report on testing and design drawings of three Rotary Joint Mechanism concepts. Upon Evaluation of the Sperry Alpha Rotary Joint Design Drawings, seven areas of potential wear problems were identified:

- (1) Power/Signal Transfer Bearing
- (2) Pinion Gear/Bearing Housing Gear Interface
- (3) Shaft Bearing (P/N 3HKR102 P7)
- (4) Pinion Bearing (P/N KAA10 AG)
- (5) Follower Bearing (P/N KB020 CPO)
- (6) β Solar Array Joint 36 inch Bearing Assembly
(Alternate Construction)
- (7) Alpha Solar Array Joint 108 inch Bearing Assembly
(Alternate Construction)

Along with the preliminary survey data forms, other pertinent drawings and specifications were sent to Battelle Columbus Laboratories for further tribology assessment.

Harold Bush of NASA/LaRC) was contacted and agreed to send a package of documentation containing all available design information. SRS received a data package on a preferred truss concept (several were still being considered). After review of this package, additional information on a few specifics was requested.

The rotary joints in the antenna system had similarities to the Ball Aerospace and Sperry designs in the solar array application. Jim Kelly of NASA/JSC, agreed to send lubrication reports and drawings on the shuttle KU band antenna gimbal system. That system was produced by Hughes Aircraft in El Segundo, California.

2.3.6 Berthing Mechanism

The McDonnell Douglas Astronautics Company began Berthing Mechanism design activities in mid 1986. In July, at the request of SRS, Mr. Fred Jankowski MSFC/EP33 forwarded a survey data form to MDAC for completion. Results were expected on or about August 1986.

2.3.7 Radiator

Through Mr. Richard Parish of NASA/JSC, LTLS survey forms were distributed to appropriate individuals at the Boeing Aerospace Company for the Two Phase TCS and Gimbal Radiator and Grumman for the Two Phase TCS and Space Erectable Radiator. In follow-up discussions it was reported that they were being prepared and might be available within a month.

2.3.8 Coarse Pointing Mechanism

In telephone contacts with Kevin Grady of the NASA Goddard Space Flight Center, the Coarse Pointing Mechanism project was still in a very conceptual stage. He suggested contacting the work package 2 contractors General Electric and Ball Brothers again around the first of September when more mature data may be available.

2.3.9 Propulsion System Thrusters

Mr. Ralph Burns MSFC/EP33 provided SRS with a data package containing a schematic and drawing of portions of the space station thruster/storage tank assembly. SRS reviewed this information to identify possible wear point candidates. Initially, potential areas of concern included O-rings, V-seals, K-seals, screw threads, fuel/oxidizer quick disconnects, and solenoid and latching valves. In the end, the SRS evaluations resulted in the identification of 36 wear points with possible long-term lubrication concerns. The points identified were associated with the fuel and oxygen solenoid valves, the oxygen and nitrogen latching valves, and the thruster assembly.

During this time, the responsibility for leading space station propulsion system design was being transferred to JSC. Walt Karakulko at NASA/JSC was contacted and provided these guidelines on operating characteristics.

gaseous fuel: <3000 psi; (20 deg. F.)<T<(150 deg. F)

liquid fuel: 400-500 psi; (20 deg. F.)<T<(150 deg. F)

On July 16, 1986 SRS sent a letter to Mr. Jeffrey Corbin of Martin Marietta requesting contacts to provide data on items from their Phase B work on the Space Station Propulsion System including an electromechanical solenoid quick disconnect fitting, thruster valves, isolation valves, and propellant/pressurant quick disconnects.

2.3.10 Solar Array

Mr. Wayne Bartlett of the NASA Lewis Research Center provided SRS with three sets of detailed notes on lubrication aspects of the LMSC Space Station Solar Array. His notes included a drawing and addressed mechanical assemblies at the base of the array connecting to the Beta joint. Another set of notes prepared by Mr. Rick Elms of LMSC identified a large number of wear points and proposed lubricants for locations on the array, base, and deployment mechanism. Also included was a copy of a memo from Mr. Fred Dolan MSFC/EH14 to Ann Whitaker MSFC/EH12 on the lubrication recommendations for the Solar Electric Propulsion / Solar Array Flight Experiment dated February 15, 1979. Although the depth of information was limited in these notes, they were sufficient to identify the location of potential lubrication concerns and some candidate lubricants for evaluation. From this information, SRS identified a number of wear points and collected

tribology data associated with the items. SRS concentrated mainly on the component description and materials summary. Mr. Bartlett later met with SRS contract representatives, providing survey forms completed by Rocketdyne also on components of the Solar array structure. The information was complementary since Rocketdyne concentrated on operating characteristics and also served to verify the points identified. Mr. Bartlett anticipated sending more detailed drawings/pictures of the individual components as they became available. A total of eight wear points were identified and surveyed in connection with the solar array and solar dynamic generator components.

2.3.11 Health Maintenance Facility

The in-flight clinical medical care facility onboard the Space Station was still in the definition process, according to Mr. James Logan, Medical Operations Branch Chief, at NASA's Johnson Space Center when he was contacted by SRS. Some small pumps were potential sources of lubrication concerns, but design candidates had not yet been defined.

2.4 Operating Requirements and Characteristics

Lubrication schemes may be classified according to the general type of lubricants, whether dry film or fluids. Typical dry film lubricants include MoS₂, Teflon, and graphite. Typical fluid lubricants include oil and greases. Determining the type of lubrication scheme for a specific element and the performance characteristics of the lubricant to reduce surface wear is an important factor in support of tribology assessments. However, the type of lubrication scheme depends on the operating characteristics of the component such as loads, rotational speeds, relative velocities and environment. Consequently, a characterization of lubricants and operating characteristics was necessary.

Lubricants operating near optical or thermal control surfaces were assessed against the outgassing requirements defined in SP-R-0022A, "General Specification Vacuum Stability Requirements of Polymeric Material for Spacecraft Application". This outgassing requirement restricts the use of polymeric materials near critical optical and thermal control surfaces to those materials having a maximum volatile condensed material (VCM) content of 0.1 percent and a total mass loss (TML) of 1.0 percent or less when tested under the following controlled test conditions:

- o Pressure: 10⁻⁶ torr or less
- o Temperature of Specimen: 125 deg. C - 1 deg. C
- o Temperature of Condensable Plates: 25 deg. C - 1 deg. C
- o Vacuum Exposure Time: 24 hours

Representative outgassing data for typical greases and lubricants are listed in Table 1. SP-R-0022A defines specific procedures for NASA approval of polymeric material selected for application near optical and thermal control surfaces. These procedures served as a baseline guide to evaluate the performance characteristics of lubricants identified during the survey. The pertinent survey data base data consisted of listing:

- o Manufacturer's trade name
- o Manufacturer of the material
- o Thermal vacuum stability (VCM and TML) data
- o Rationale for use of material which failed outgassing requirement.

OUTGASSING DATA FOR TYPICAL GREASES AND LUBRICANTS

Material	MFR Code	STML	SCVDN	Cure Time	Cure Temp	Atmos	Application
Aerodag G Colloidal Graphite/Isopropyl Alcohol	ACH	1.78	.11	24H	25	Air	Lubricant
Aerodag G Colloidal Graphite/Isopropyl Alcohol	ACH	2.05	.04	24H	60	Air	Lubricant
Aerodag M Colloidal MOS2/Isopropyl Alcohol	ACH	4.97	.19	24H	25	Air	Lubricant
Aplazon C 011 Vacuum Degassed	BID	81.19	47.47	1H	65	E-2	Grease
Aplazon H Hydrocarbon Grease	BID	.25	.02				Grease
Aplazon L Grease	BID	.04	.01				Grease
Aplazon M Grease Vacuum Degassed	BID	.34	.29	1H	65	E-2	Lubricant
Aplazon N Grease	BID	.08	.00				Grease
Aplazon T Grease	BID	.76	.12				Grease
Black Magic Moly Spray/Foil - Aerosol MOS2	SPR	.27	.02	24H	25	Air	Lubricant
Brayco 813 Clear 011 Batch E1A3	BOC	.74	.32				Lubricant
Brayco 8152 Clear 011 Batch DLM3	BOC	.03	.00				Lubricant
Braycote 31-38 Grease Batch DLD1 White	BOC	.07	.03				Grease
Braycote 31-38 Grease White	BOC	.07	.03	7H	100	E-6	Lubricant
Braycote 31-38-MS Grease Batch DLG1 Gray	BOC	.04	.01				Grease
Braycote 31-38-RE Grease Batch CLT7 Yellow	BOC	.09	.04				Grease
Braycote 31-38-ZH Grease Batch ELM4 Black	BOC	.02	.01				Lubricant
Braycote 803 Grease Batch ELJ12 White	BOC	.24	.13				Grease
Crown 6065 Green TNE Aerosol Coating/F	CIP	.52	.09	20H	177	Air	Lubricant
C6-1103 Silicone Grease	DCC	.17	.00	24H	121	E-3	Grease
DC 20-057 Silicone Optical Coupling Compound	DCC	.86	.27				Grease
Dri-Slip Powdered Lubricant - Aerosol	MM	10.73	4.60				Lubricant
DriLube 822 Fluorosilicone Lubricating Grease	DM	6.51	2.47	1H	66	Air	Epoxy Lube
Eccostip TH-24 Low Friction Epoxy	EM	.40	.00	1H	80	Air	
				24H	116	Air	
EXCEL MR-25 Aromatic Polyester/TFE Blend Tan	CRB	.01	.00				Solid Lube
Emvex 1000	ROG	1.94	.01				Bearing Material
Emvex 1000X	ROG	1.74	.01	24H	204	Air	Bearing Material
Emvex 1000X Polyimide	ROG	1.87	.01	24H	204	Air	Bearing Material
Emvex 1115	ROG	1.83	.00				Bearing Material
Emvex 1115	ROG	1.67	.01				Bearing Material
Emvex 1228	ROG	1.29	.03	24H	204	Air	Bearing Material
Emvex 1315	ROG	1.29	.00				Bearing Material
Emvex 1315	ROG	1.25	.02	24H	204	Air	Bearing Material
Emvex 1315	ROG	2.03	.00				Bearing Material
Emvex 1315	ROG	1.36	.02	24H	240	Air	Bearing Material
Fluorom Dry Film Lubricant Aerosol	BAP	15.05	7.59	70	25	Air	Lubricant
Fluoroglide FB Aerosol Dry Film Lubricant/F	CHE	9.49	3.09	70	25	Air	Lubricant
Fluorogint 500 Mica Filled TFE	PPC	.05	.00				Bearing Material
GE 1147 Methyl Alkyl Silicone Lube 011	GEC	4.28	2.41				011
MI-Vac Grease	DCC	1.52	.34				Lubricant/Grease
MI-Vac Grease 970V Silicone	DCC	.73	.22				Lubricant/Grease

2.5 Tribology Matrix / Data Tape

The results of the tribology survey and assessment were to be stored on a data tape implemented on the MSFC computer system. The RFP required the delivery of the data tape at the completion of Phase II. However, SRS proposed an option to implement the data tape during the conduct of Phase I and incrementally update the data tape as more mature data became available. By implementing the data tape early in the study, it was hoped a listing of major Space Station elements could be provided to MSFC facilitating later updating efforts. Supporting this option, a lubrication evaluation tool was to be made available to the MSFC for use in identifying probable problem areas early into the Space Station studies. This would support test plan development and conduct in critical, deficient, or advanced technologies areas.

A description of the proposed formats of the data tape and descriptions of the individual data items to be recorded are discussed below. The actual format of the tape would have been to a small degree dependent on the computer system on which it was to be loaded. It was assumed that the Univac 1100 Series computers at MSFC would be the host system. After the start of the study, MSFC upgraded the central computer facility with a Cray mainframe. However, no significant impacts on the data tape plans resulted from another computer being chosen since interfaces to Univac peripherals and systems were retained. The SRS staff maintains familiarity with all the major computer systems at MSFC.

The standard tapes used in the MSFC tape library are the 9 Track, 1600 BPI, 10-inch diameter magnetic tapes. The tapes were planned to be recorded in a standard format, such as unlabeled Univac COPY, G format (the format of COSMIC UNIVAC NASTRAN TAPES), or the standard Univac COPIN format. SRS staff have used these formats when transferring data recorded on tapes from several sources including COSMIC, the UAH Univac, and IBM machines to MSFC computers.

The organization of the data was structured to take on a matrix format. The results of the tribology survey were to be synthesized into a row column table. The tribology matrix included an identification of the components, points of wear, and data in five major categories: structural, lubrication, environment, operating conditions and characteristics, and failure mechanisms and consequences. An illustration of the planned matrix structure is provided in figure 13. Shown is a representative of a fluid coupling design presented in JSC-19989. The fluid coupling

has been broken down to the structured element level, and materials, surface finishes, environments, and operating characteristics have been identified.

Character- istic Point of Wear	Structural		Lubrication			Environment		Operating Conditions				Failure Mechanism/Consequences	
	Material	Surface Finish	Type	Designation	Outgassing Characteristics	Type	Effects on Operating Performance	Loads/Stresses	Torque	Rotational Speeds	Required Life	Mechanism	Consequences
Ammonia Fluid Coupling Seal	Virgin TFE teflon Against 316 Stainless Steel	6-8 - inch milled 6-8 - inch polished	Dry	unlubricated	OK	Int. Inside Air outside	•Corrosive to certain material •Leads to corrosion failure and coupling failure		865± inch-pounds	1 Rev/90 min	10 yrs	Wear, Corrosion	Leakage to habitable areas could cause health problems

3.0 CONCLUSIONS

Upon completion of the Phase I activities, the information was organized into the tribology matrix format and compiled into a preliminary tribology matrix as depicted as table 2 in the following pages. The information necessary to provide meaningful indications of failure mechanisms and consequences was not available and was deleted from the format. Originally, there were planned to be two updates to the matrix. Those updates were eliminated in the renegotiations of the contracted effort. Since the LTLS program was brought to focus on a few items early in the program, there no longer existed a need for a computer system to host and manipulate vast amounts of data. Early data base programming efforts using the RIM 5 relational data base software were discontinued when it became apparent that a different scope was prudent in the data requirements of the project. Appendix A contains the individual data sheets prepared from the survey results for each of the line items summarized below.

Most of the wear points identified were in conjunction with the RCS thruster assemblies and the solar array. A few wear points were identified for the Beta Joint, the Solar Dynamic Concentrator, and pumps in the fluid control system.

Survey Number	Function	Description	Contact	Lubrication	Load, Life, and Motion	Sun/Temperature (deg F)	Operating Conditions	Comment
1	SMART BEARING DRIVE ASSEMBLY	Shaft Bearing P/N 3K6R102P7	Surface #1 440 C Surface Finish Set by Sperry Surface #2 440 c Surface Finish Set by Sperry	Lubrication Scheme: Fluid Lubricant Micronic 601 Yes <input checked="" type="radio"/> No <input type="radio"/>	Operating Peak Rotate Speed (rpm) 0.5 Occasion Type None	Exposure None Percent Time 0 % Low: 8 F Environment High: 105 F Vacuum	DN value at 8 RPM min	Single 5mm bore ball bearing (catalog number 3K6R102P7 by MFG Corp) 400 lb max static load capacity. Cage is Conard machined phenolic.
2	Solar array deployment	Bearings in PV array deployment motor	Surface #1 Unspecified Surface Finish Surface #2 Unspecified Surface Finish	Lubrication Scheme: Fluid Lubricant Boreoil 601 Yes <input checked="" type="radio"/> No <input type="radio"/>	Operating Peak Rotate Speed (rpm) 1200 Occasion Type Unspecified	Exposure Continuous Percent Time 100 % Low: 112 F Environment High: 176 F Vacuum		S7S Cargo Bay vibration qualification level.
3	Solar array mast extension	Rolling nut for solar array mast extension	Surface #1 Unspecified Surface Finish Surface #2 Unspecified Surface Finish	Lubrication Scheme: Dry Film Lubricant Everbond 620 Yes <input checked="" type="radio"/> No <input type="radio"/>	Operating Peak Rotate Speed (rpm) 0 Occasion Type Unspecified	Exposure Intermittent Percent Time 50 % Low: 112 F Environment High: 176 F Vacuum		S7S Cargo Bay vibration qualification level.
4	Provides line pointing actuation of Solar Dynamic Concentrator reflector.	Actuator motor bearings, threads, nuts	Surface #1 T80 steel Surface Finish TBD Surface #2 T80 steel Surface Finish TBD	Lubrication Scheme: Fluid Lubricant TBD Yes <input checked="" type="radio"/> No <input type="radio"/>	Operating Peak Rotate Speed (rpm) 100 Occasion Type Unspecified	Exposure None Percent Time 0 % Low: 140 F Environment High: 140 F Vacuum	< 1 Hz continuous oscillation.	For additional information, contact Sundstrand Energy Systems, Rockwell, Wichita, 61725. Dual failure tolerant electromechanical actuator.
5	Solar Dynamic Concentrator deployment latch catch mechanism.	Catch Journal	Surface #1 TBD Surface Finish Unspecified Surface #2 TBD Surface Finish Unspecified	Lubrication Scheme: Dry Film Lubricant TBD Yes <input checked="" type="radio"/> No <input type="radio"/>	Operating Peak Rotate Speed (rpm) 0 Occasion Type Intermittent	Exposure Intermittent Percent Time 70 % Low: 0 F Environment High: 0 F Vacuum		Required to function only during deployment or retraction of concentrator. Journal bearing with <0.5 inch radius.
6	Fire Panning for the Solar Dynamic Concentrator	Universal Joint Journal Bearings	Surface #1 TBD A1 Surface Finish Unspecified Surface #2 TBD A1 Surface Finish Unspecified	Lubrication Scheme: Dry Film Lubricant Unspecified Yes <input checked="" type="radio"/> No <input type="radio"/>	Operating Peak Rotate Speed (rpm) 0 Occasion Type TBD	Exposure Intermittent Percent Time 70 % Low: 150 F Environment High: 150 F Vacuum	Continuous, low load, low velocity, random motion.	Journal bearing radius given as "TBD 2 inches".
7	Concentrator Reflector Deployment	High Head Bearing	Surface #1 TBD Surface Finish Unspecified Surface #2 TBD Surface Finish Unspecified	Lubrication Scheme: Dry Film Lubricant Unspecified Yes <input checked="" type="radio"/> No <input type="radio"/>	Operating Peak Rotate Speed (rpm) 150 Occasion Type Unspecified	Exposure Intermittent Percent Time 70 % Low: 150 F Environment High: 150 F Vacuum		Roller bearing.
8	Alternator ball journal and thrust bearings for the Closed Brayton Cycle Solar Dynamic Power Generation Power Conversion Unit.	Roller/Housing Interface	Surface #1 Roller Journal Surface Finish Unspecified Surface #2 Fall Leaves Surface Finish Unspecified	Lubrication Scheme: Fluid Lubricant Hawley/700 (MIL-46) Yes <input checked="" type="radio"/> No <input type="radio"/>	Operating Peak Rotate Speed (rpm) 32000 Occasion Type Continuous	Exposure None Percent Time 0 % Low: 100 F Environment High: 300 F Pressurized	Required to function only during deployment and retraction on-orbit.	Journal Bearing with 2.726 inch radius operating at a 760 lb relative velocity. Volatile fluid is contained in sealed housing. Extreme operating conditions may include shims -100.
9	Pump to circulate radiator fluid. The Radiator Coolant Pump Package of the Solar Dynamic Power Generation System has an integrated motor shutoff/impeller Centrifugal Pump.	Hydrodynamic Journal Bearings	Surface #1 Motor Shaft Surface Finish Unspecified Surface #2 Housing Journal Surface Finish Unspecified	Lubrication Scheme: Fluid Lubricant FC75 Immersion Yes <input checked="" type="radio"/> No <input type="radio"/>	Operating Peak Rotate Speed (rpm) 0 Occasion Type Continuous	Exposure None Percent Time 0 % Low: 0 F Environment High: 100 F Pressurized		Rotor lubricant inventory is approximately 4 lbs of lubricant. Operating load is given as approximately 100 lbf shaft pressure in housing is 20 to 60 psi.
10	To position the Propulsion System Thruster assembly solenoid valve when solenoid is not engaged.	Ends of spring at shaft and spring stop	Surface #1 Unspecified Surface Finish Unspecified Surface #2 Unspecified Surface Finish Unspecified	Lubrication Scheme: Unspecified Lubricant Unspecified Yes <input checked="" type="radio"/> No <input type="radio"/>	Operating Peak Rotate Speed (rpm) 0 Occasion Type Unspecified	Exposure Unspecified Percent Time 0 % Low: 0 F Environment High: 0 F Unspecified		High temperature of 200 deg F in extreme operating conditions. Operating load given as "2 lb/sec 60 psi" under consideration.
11	Fluid connector for Propulsion System Thruster assembly.	Screw threads on propellant feed lines	Surface #1 Mils connector Surface Finish Unspecified Surface #2 Female connector Surface Finish Unspecified	Lubrication Scheme: Unspecified Lubricant Unspecified Yes <input checked="" type="radio"/> No <input type="radio"/>	Operating Peak Rotate Speed (rpm) 0 Occasion Type Unspecified	Exposure Unspecified Percent Time 0 % Low: 0 F Environment High: 0 F Unspecified		
12	Affect motion of the valve shaft on the Propulsion System Thruster assembly solenoid valve.	Electromagnetic Plates	Surface #1 Fixed Plate Surface Finish Unspecified Surface #2 Moving Plate Surface Finish Unspecified	Lubrication Scheme: Unspecified Lubricant Unspecified Yes <input checked="" type="radio"/> No <input type="radio"/>	Operating Peak Rotate Speed (rpm) 0 Occasion Type Unspecified	Exposure Unspecified Percent Time 0 % Low: 0 F Environment High: 0 F Unspecified		
13	Permit or restrict fluid flow in the solenoid valve head of the Propulsion System Thruster assembly.	Valve Seats/Stems	Surface #1 Valve Seat Surface Finish Unspecified Surface #2 Valve Stem Surface Finish Unspecified	Lubrication Scheme: Unspecified Lubricant Unspecified Yes <input checked="" type="radio"/> No <input type="radio"/>	Operating Peak Rotate Speed (rpm) 0 Occasion Type Unspecified	Exposure Unspecified Percent Time 0 % Low: 0 F Environment High: 0 F Unspecified		
14	Guide motion of valve stem in solenoid valve of the Propulsion System Thruster assembly.	Valve Shaft Guides	Surface #1 Valve Stem Surface Finish Unspecified Surface #2 Valve Stem Guides Surface Finish Unspecified	Lubrication Scheme: Unspecified Lubricant Unspecified Yes <input checked="" type="radio"/> No <input type="radio"/>	Operating Peak Rotate Speed (rpm) 0 Occasion Type Unspecified	Exposure Unspecified Percent Time 0 % Low: 0 F Environment High: 0 F Unspecified		
15	To form a connection between tubing and the dummy spark igniter in the Propulsion System Thruster assembly.	Dummy spark igniter/wiring connection	Surface #1 Typen tubing Surface Finish Unspecified Surface #2 Dummy Spark Igniter Surface Finish Unspecified	Lubrication Scheme: Unspecified Lubricant Unspecified Yes <input checked="" type="radio"/> No <input type="radio"/>	Operating Peak Rotate Speed (rpm) 0 Occasion Type Unspecified	Exposure Unspecified Percent Time 0 % Low: 0 F Environment High: 0 F Unspecified		

Survey	Function	Description	Contact	Lubrication	Load, Use, and Motion	Surf Temperature (deg F)	Operating Conditions	Comment
Number 16 Contractor: Rockwell	Hardware attachment device: Hex head bolt assembly in the valve bracket assembly of the Propulsion System Thruster assembly.	Screw threads/bolt	Surface #1: Screw threads Surface #2: Hex head bolt Surface #3: Unspecified	Lubrication Scheme: Fluid Lubricant: MoS2 Compliance: SF R 0022A Yes <input type="radio"/> No <input checked="" type="radio"/>	Operating: 0 Life (years) Peak: 0 Rotary Speed (rpm): 0 Excitation Type: Unspecified	Exposure Unspecified Percent Time: 0 % High: 0 F Environment Low: 0 F Unspecified		
Number 17 Contractor: Rockwell	Hardware attachment device: screwdriver assembly in the Propulsion System Thruster assembly power head.	contact surfaces and screw threads	Surface #1: Screw threads Surface #2: Screw Cap Socket Surface #3: Unspecified	Lubrication Scheme: Fluid Lubricant: MoS2 Compliance: SF R 0022A Yes <input checked="" type="radio"/> No <input type="radio"/>	Operating: 0 Life (years) Peak: 0 Rotary Speed (rpm): 0 Excitation Type: Unspecified	Exposure Unspecified Percent Time: 0 % High: 0 F Environment Low: 0 F Unspecified		Socket torqued to 22 (± 1) inch lbs
Number 18 Contractor: Rockwell	Swaglock to a union (CRES) in propellant fill line connectors of the Propulsion System Thruster assembly.	contact surfaces and screw threads	Surface #1: Screw threads Surface #2: Unspecified Surface #3: Unspecified	Lubrication Scheme: Unspecified Lubricant: Unspecified Compliance: SF R 0022A Yes <input type="radio"/> No <input checked="" type="radio"/>	Operating: 0 Life (years) Peak: 0 Rotary Speed (rpm): 0 Excitation Type: Unspecified	Exposure Unspecified Percent Time: 0 % High: 0 F Environment Low: 0 F Unspecified		
Number 19 Contractor: Rockwell	To seal the full line connection between the seal solenoid valve and the valve assembly bracket in the Propulsion System Thruster assembly.	V Seal	Surface #1: V-Seal-Inco 718 Surface #2: Valve Assy Bracket Surface #3: Unspecified	Lubrication Scheme: Dry Film Lubricant: Teflon Compliance: SF R 0022A Yes <input checked="" type="radio"/> No <input type="radio"/>	Operating: 0 Life (years) Peak: 0 Rotary Speed (rpm): 0 Excitation Type: Unspecified	Exposure Unspecified Percent Time: 0 % High: 0 F Environment Low: 0 F Unspecified		
Number 20 Contractor: Rockwell	Seal the connection between the Propulsion System Thruster assembly power head and combustion chamber.	V Seal	Surface #1: V-Seal-Inco 718 Surface #2: Power Head Surface #3: Unspecified	Lubrication Scheme: Unspecified Lubricant: Unspecified Compliance: SF R 0022A Yes <input type="radio"/> No <input checked="" type="radio"/>	Operating: 0 Life (years) Peak: 0 Rotary Speed (rpm): 0 Excitation Type: Unspecified	Exposure Unspecified Percent Time: 0 % High: 0 F Environment Low: 0 F Unspecified		
Number 21 Contractor: Rockwell	Seal the threaded connection between the Dummy Spark Igniter and the Power Head in the Propulsion System Thruster assembly.	K Seal	Surface #1: K Seal Surface #2: Power Head Surface #3: Unspecified	Lubrication Scheme: Unspecified Lubricant: Unspecified Compliance: SF R 0022A Yes <input type="radio"/> No <input checked="" type="radio"/>	Operating: 0 Life (years) Peak: 0 Rotary Speed (rpm): 0 Excitation Type: Unspecified	Exposure Unspecified Percent Time: 0 % High: 0 F Environment Low: 0 F Unspecified		Lubricant in under pressure at the contact surface and in vacuum at the outer edge of the contact surface.
Number 22 Contractor: Rockwell	O-ring seal for the threaded connector on the Dummy Spark Igniter of the Propulsion System Thruster assembly.	O-ring seal	Surface #1: Unspecified Surface #2: Unspecified Surface #3: Unspecified	Lubrication Scheme: Unspecified Lubricant: Unspecified Compliance: SF R 0022A Yes <input type="radio"/> No <input checked="" type="radio"/>	Operating: 0 Life (years) Peak: 0 Rotary Speed (rpm): 0 Excitation Type: Unspecified	Exposure Unspecified Percent Time: 0 % High: 0 F Environment Low: 0 F Unspecified		
Number 23 Contractor: Moog	Valve guides for the Oxygen Solenoid Valve of Propulsion system.	Poppet/Punger Teflon Guides	Surface #1: Poppet Surface #2: Plunger Surface #3: Unspecified	Lubrication Scheme: Dry Film Lubricant: Teflon Compliance: SF R 0022A Yes <input checked="" type="radio"/> No <input type="radio"/>	Operating: 0 Life (years) Peak: 0 Rotary Speed (rpm): 0 Excitation Type: Unspecified	Exposure Unspecified Percent Time: 0 % High: 0 F Environment Low: 0 F Unspecified		
Number 24 Contractor: Moog	Position the Oxygen Solenoid Valve of the APFCS control system in the Space Station Propulsion System when the solenoid is not engaged.	Low stress helical spring	Surface #1: Helical Spring Surface #2: Valve Seat Surface #3: Unspecified	Lubrication Scheme: Unspecified Lubricant: Unspecified Compliance: SF R 0022A Yes <input type="radio"/> No <input checked="" type="radio"/>	Operating: 0 Life (years) Peak: 0 Rotary Speed (rpm): 0 Excitation Type: Unspecified	Exposure None Percent Time: 0 % High: 0 F Environment Low: 0 F Unspecified		
Number 25 Contractor: Moog	Valve seal for the Oxygen Solenoid Valve of the APFCS control system in the Space Station Propulsion system.	Valve head / seal contact	Surface #1: Valve seat Surface #2: Valve Head Surface #3: Unspecified	Lubrication Scheme: Dry Film Lubricant: Polyimide (Vespal) Compliance: SF R 0022A Yes <input checked="" type="radio"/> No <input type="radio"/>	Operating: 0 Life (years) Peak: 0 Rotary Speed (rpm): 0 Excitation Type: Unspecified	Exposure None Percent Time: 0 % High: 0 F Environment Low: 0 F Unspecified		
Number 26 Contractor: Moog	Seal Oxygen Solenoid Valve Components in the APFCS control system of the Space Station Propulsion system.	Seal O-rings	Surface #1: O-Ring Surface #2: Various Surface #3: Unspecified	Lubrication Scheme: Unspecified Lubricant: Unspecified Compliance: SF R 0022A Yes <input type="radio"/> No <input checked="" type="radio"/>	Operating: 0 Life (years) Peak: 0 Rotary Speed (rpm): 0 Excitation Type: Unspecified	Exposure None Percent Time: 0 % High: 0 F Environment Low: 0 F Unspecified		
Number 27 Contractor: Moog	Connector in the Oxygen Solenoid Valve of the Space Station Propulsion System.	Screw Threads	Surface #1: Screw threads Surface #2: Thread Grooves Surface #3: Unspecified	Lubrication Scheme: Unspecified Lubricant: Unspecified Compliance: SF R 0022A Yes <input type="radio"/> No <input checked="" type="radio"/>	Operating: 0 Life (years) Peak: 0 Rotary Speed (rpm): 0 Excitation Type: Unspecified	Exposure Unspecified Percent Time: 0 % High: 0 F Environment Low: 0 F Unspecified		
Number 28 Contractor: Moog	O-ring seal between leaching mechanism casing and plunger mechanism of the Oxygen and Nitrogen Leaching Valve in the APFCS Control System of the Space Station Propulsion System.	O-ring seal	Surface #1: Casing mechanism Surface #2: plunger mechanism Surface #3: Unspecified	Lubrication Scheme: Unspecified Lubricant: Unspecified Compliance: SF R 0022A Yes <input type="radio"/> No <input checked="" type="radio"/>	Operating: 0 Life (years) Peak: 0 Rotary Speed (rpm): 0 Excitation Type: Unspecified	Exposure None Percent Time: 0 % High: 0 F Environment Low: 0 F Unspecified		
Number 29 Contractor: Moog	Assembly and Adjustment Brazeed connection in the Oxygen and Nitrogen Leaching Valve of the APFCS Control System in the Space Station Propulsion System.	Screw Threads	Surface #1: Screw Threads Surface #2: Thread Grooves Surface #3: Unspecified	Lubrication Scheme: Unspecified Lubricant: Unspecified Compliance: SF R 0022A Yes <input type="radio"/> No <input checked="" type="radio"/>	Operating: 0 Life (years) Peak: 0 Rotary Speed (rpm): 0 Excitation Type: Unspecified	Exposure Unspecified Percent Time: 0 % High: 0 F Environment Low: 0 F Unspecified		
Number 30 Contractor: Moog	Flare Seal in the Oxygen and Nitrogen Leaching Valve in the APFCS Control System of the Space Station Propulsion System.	Surface contact	Surface #1: Flare seal Surface #2: Flare seal Surface #3: Unspecified	Lubrication Scheme: Unspecified Lubricant: Unspecified Compliance: SF R 0022A Yes <input type="radio"/> No <input checked="" type="radio"/>	Operating: 0 Life (years) Peak: 0 Rotary Speed (rpm): 0 Excitation Type: Unspecified	Exposure None Percent Time: 0 % High: 0 F Environment Low: 0 F Unspecified		

Survey	Function	Description	Contact	Lubrication	Load, Life, and Motion	Sun/Temperature (deg F)	Operating Conditions	Comment
Number 31 Contractor: Moog	O Ring between end cap mechanism and Valve Adjustment mechanism in the Oxygen and Nitrogen Latching Valve of the APCCS Control System in the Space Station Propulsion System	O-Ring	Surface #1 O-Ring Surface Finish Unspecified Surface #2 End cap/Valve adjust Surface Finish Unspecified	Lubrication Scheme: Unspecified Lubricant Unspecified SP R 0022A Compatible Yes <input checked="" type="radio"/> No <input type="radio"/>	Operating 0 Life (years) Peak 10 Percent Time 0 % Rotate Speed (rpm) 0 Excitation Type Unspecified	Exposure None Percent Time 0 % Low: 0 F Environment High: 0 F Environment 0 F Pressurized		
Number 32 Contractor: Moog	Seal the Oxygen & Nitrogen Latching Valve casing from the electric drive motor in the APCCS Control System of the Space Station Propulsion System	Belleville Spring Seal	Surface #1 Belleville Spring Inner Surface Finish Unspecified Surface #2 Belleville Spring Outer Surface Finish Unspecified	Lubrication Scheme: Unspecified Lubricant Unspecified SP R 0022A Compatible Yes <input checked="" type="radio"/> No <input type="radio"/>	Operating 0 Life (years) Peak 0 Percent Time 0 % Rotate Speed (rpm) 0 Excitation Type Unspecified	Exposure Unspecified Percent Time 0 % Low: 0 F Environment High: 0 F Environment 0 F Pressurized		
Number 33 Contractor: Moog	O Ring to seal the Relief Valve to the motor case in the APCCS Control System Oxygen and Nitrogen Latching Valve of Space Station Propulsion System	O-Ring	Surface #1 O-Ring Surface Finish Unspecified Surface #2 Relief Valve Surface Finish Unspecified	Lubrication Scheme: Unspecified Lubricant Unspecified SP R 0022A Compatible Yes <input checked="" type="radio"/> No <input type="radio"/>	Operating 0 Life (years) Peak 0 Percent Time 0 % Rotate Speed (rpm) 0 Excitation Type Unspecified	Exposure None Percent Time 0 % Low: 0 F Environment High: 0 F Environment 0 %		
Number 34 Contractor: Moog	Electric Drive motor actuator for the Oxygen and Nitrogen Latching Valve in the Space Station Propulsion System APCCS Control System	Bearings/Planetary Gears	Surface #1 Unspecified Surface Finish Unspecified Surface #2 Unspecified Surface Finish Unspecified	Lubrication Scheme: Unspecified Lubricant Unspecified SP R 0022A Compatible Yes <input checked="" type="radio"/> No <input type="radio"/>	Operating 0 Life (years) Peak 0 Percent Time 0 % Rotate Speed (rpm) 0 Excitation Type Unspecified	Exposure Unspecified Percent Time 0 % Low: 0 F Environment High: 0 F Environment 0 %		
Number 35 Contractor: Lockheed	Wire Wrap transfers power and data across Beta Joint	Relative Motion & Contact w/ Structure	Surface #1 Wire Wrap Surface Finish Unspecified Surface #2 Aids Housing Surface Finish Unspecified	Lubrication Scheme: None Plasm Lubricant Unspecified SP R 0022A Compatible Yes <input checked="" type="radio"/> No <input type="radio"/>	Operating 0 Life (years) Peak 0 Percent Time 0 % Rotate Speed (rpm) 0 Excitation Type Unspecified	Exposure Intermittent Percent Time 0 % Low: 0 F Environment High: 0 F Environment 0 F Vacuum		
Number 36 Contractor: Lockheed	Beta Joint Control Motor and Gears	Bearings / Gears	Surface #1 Unspecified Surface Finish Unspecified Surface #2 Unspecified Surface Finish Unspecified	Lubrication Scheme: Unspecified Lubricant Unspecified SP R 0022A Compatible Yes <input checked="" type="radio"/> No <input type="radio"/>	Operating 0 Life (years) Peak 0 Percent Time 0 % Rotate Speed (rpm) 0 Excitation Type Unspecified	Exposure None Percent Time 0 % Low: 0 F Environment High: 0 F Environment 0 %		
Number 37 Contractor: Lockheed	Main Bearings in the housing of the Beta Joint of the Electrical Power System	Main Bearings	Surface #1 Inner Race Surface Finish Unspecified Surface #2 Outer Race Surface Finish Unspecified	Lubrication Scheme: Unspecified Lubricant Unspecified SP R 0022A Compatible Yes <input checked="" type="radio"/> No <input type="radio"/>	Operating 0 Life (years) Peak 0 Percent Time 0 % Rotate Speed (rpm) 0 Excitation Type Unspecified	Exposure Intermittent Percent Time 0 % Low: 0 F Environment High: 0 F Environment 0 F Vacuum		
Number 38 Contractor: Lockheed	Bearings in pump of water supply assembly of water electrolysis system	Bearings	Surface #1 Surface Finish Unspecified Surface #2 Surface Finish Unspecified	Lubrication Scheme: Unspecified Lubricant Unspecified SP R 0022A Compatible Yes <input checked="" type="radio"/> No <input type="radio"/>	Operating 0 Life (years) Peak 0 Percent Time 0 % Rotate Speed (rpm) 0 Excitation Type Unspecified	Exposure Unspecified Percent Time 0 % Low: 0 F Environment High: 0 F Environment 0 %		
Number 39 Contractor: Lockheed	Cable bar lead pulleys in blanket tensioning and guidewire system of solar array of electrical power	Cable bar lead pulleys	Surface #1 pin/val Surface Finish Unspecified Surface #2 bushing/ai Surface Finish Unspecified	Lubrication Scheme: Dry Film Lubricant Unspecified SP R 0022A Compatible Yes <input checked="" type="radio"/> No <input type="radio"/>	Operating 0 Life (years) Peak 0 Percent Time 0 % Rotate Speed (rpm) 0 Excitation Type Unspecified	Exposure: Intermittent Percent Time 0 % Low: 0 F Environment High: 0 F Environment 0 F Vacuum		
Number 40 Contractor: Lockheed	Drive motors in line pointing mechanism of solar dynamic system of electrical power system	Drive motors	Surface #1 Surface Finish Unspecified Surface #2 Surface Finish Unspecified	Lubrication Scheme: Unspecified Lubricant Unspecified SP R 0022A Compatible Yes <input checked="" type="radio"/> No <input type="radio"/>	Operating 0 Life (years) Peak 0 Percent Time 0 % Rotate Speed (rpm) 0 Excitation Type Unspecified	Exposure Unspecified Percent Time 0 % Low: 0 F Environment High: 0 F Environment 0 %		
Number 41 Contractor: Lockheed	Gearbox of the pointing mechanism in solar dynamic system in electrical power system	Gearbox	Surface #1 Surface Finish Unspecified Surface #2 Surface Finish Unspecified	Lubrication Scheme: Unspecified Lubricant Unspecified SP R 0022A Compatible Yes <input checked="" type="radio"/> No <input type="radio"/>	Operating 0 Life (years) Peak 0 Percent Time 0 % Rotate Speed (rpm) 0 Excitation Type Unspecified	Exposure Unspecified Percent Time 0 % Low: 0 F Environment High: 0 F Environment 0 %		
Number 42 Contractor: Lockheed	Ballscrew in the pointing mechanism of solar dynamic system of electrical power system	Ballscrew	Surface #1 Surface Finish Unspecified Surface #2 Surface Finish Unspecified	Lubrication Scheme: Unspecified Lubricant Unspecified SP R 0022A Compatible Yes <input checked="" type="radio"/> No <input type="radio"/>	Operating 0 Life (years) Peak 0 Percent Time 0 % Rotate Speed (rpm) 0 Excitation Type Unspecified	Exposure Unspecified Percent Time 0 % Low: 0 F Environment High: 0 F Environment 0 %		
Number 43 Contractor: Lockheed	Actuator to concentrator of line pointing mechanism of solar dynamic system of electrical power system	Actuator to concentrator VF	Surface #1 Surface Finish Unspecified Surface #2 Surface Finish Unspecified	Lubrication Scheme: Unspecified Lubricant Unspecified SP R 0022A Compatible Yes <input checked="" type="radio"/> No <input type="radio"/>	Operating 0 Life (years) Peak 0 Percent Time 0 % Rotate Speed (rpm) 0 Excitation Type Unspecified	Exposure Unspecified Percent Time 0 % Low: 0 F Environment High: 0 F Environment 0 %		
Number 44 Contractor: Lockheed	Gas or foil bearings in heat engine of solar dynamic system of electrical power system	Gas or foil bearings	Surface #1 Surface Finish Unspecified Surface #2 Surface Finish Unspecified	Lubrication Scheme: Unspecified Lubricant Unspecified SP R 0022A Compatible Yes <input checked="" type="radio"/> No <input type="radio"/>	Operating 0 Life (years) Peak 0 Percent Time 0 % Rotate Speed (rpm) 0 Excitation Type Unspecified	Exposure Unspecified Percent Time 0 % Low: 0 F Environment High: 0 F Environment 0 %		
Number 45 Contractor: Lockheed	Main base spider bushing in mast cantilever of solar array of electrical power	Mast base spider bushing	Surface #1 spider bushing/Al Surface Finish Unspecified Surface #2 spider/Al Surface Finish Unspecified	Lubrication Scheme: Dry Film Lubricant Unspecified SP R 0022A Compatible Yes <input checked="" type="radio"/> No <input type="radio"/>	Operating 0 Life (years) Peak 0 Percent Time 0 % Rotate Speed (rpm) 0 Excitation Type Intermittent	Exposure Intermittent Percent Time 0 % Low: 0 F Environment High: 0 F Environment 0 F Vacuum	Radication type NASA TM 86440 Normal operating environment 160	

Survey	Function	Description	Contact	Lubrication	Load, Life, and Motion	Surf Temperature (deg F)	Operating Conditions	Comment
Number 46 Contractor: Lockhead	Longer on end being in mast carrier of solar array of electrical power	Ring to spider A1 to A1.	Surface #1 On end being/A1 Surface #2 Pin to spider/A1 Surface #3 Pin to spider/A1 Surface #4 Pin to spider/A1	Lubrication Scheme Dry Film Lubricant Everbrite 820 SP R 0022A Compatible Yes <input type="radio"/> No <input type="radio"/>	Operating Peak Rotate Speed (rpm) 0 Oscillation Type Intermittent Life (years) 0 Exposure Intermittent Percent Time 83 % Environment High 0 F Vacuum	Exposure Intermittent Percent Time 83 % Environment High 0 F Vacuum	Radiation Type: NASA TM 86460 Normal operating Environment: 180°F	
Number 47 Contractor: Lockhead	Mast element/employer interface in mast carrier of solar array of electrical power	Mast element to employer interface	Surface #1 Mast element/A1 Surface #2 Employer nut thread Surface #3 Employer nut thread Surface #4 Employer nut thread	Lubrication Scheme Dry Film Lubricant Everbrite 820 SP R 0022A Compatible Yes <input type="radio"/> No <input type="radio"/>	Operating Peak Rotate Speed (rpm) 0 Oscillation Type Intermittent Life (years) 0 Exposure Intermittent Percent Time 83 % Environment High 0 F Vacuum	Exposure Intermittent Percent Time 83 % Environment High 0 F Vacuum	Radiation Type: NASA TM 86460 Normal operating Environment: 180°F	
Number 48 Contractor: Lockhead	Employer nutting ballbearings in mast carrier of solar array of electrical power	Employer nutting ballbearings	Surface #1 Nutting bearing/8S Surface #2 Nutting bearing/8S Surface #3 Nutting bearing/8S Surface #4 Nutting bearing/8S	Lubrication Scheme Dry Film Lubricant Everbrite 820 SP R 0022A Compatible Yes <input type="radio"/> No <input type="radio"/>	Operating Peak Rotate Speed (rpm) 0 Oscillation Type Intermittent Life (years) 0 Exposure Intermittent Percent Time 83 % Environment High 0 F Vacuum	Exposure Intermittent Percent Time 83 % Environment High 0 F Vacuum	Radiation Type: NASA TM 86460 Normal operating Environment: 180°F	
Number 49 Contractor: Lockhead	Station corner ring in mast of solar array of electrical power	Station corner ring	Surface #1 Station corner ring/A1 Surface #2 Station corner ring/A1 Surface #3 Station corner ring/A1 Surface #4 Station corner ring/A1	Lubrication Scheme Dry Film Lubricant Everbrite 820 SP R 0022A Compatible Yes <input type="radio"/> No <input type="radio"/>	Operating Peak Rotate Speed (rpm) 0 Oscillation Type Intermittent Life (years) 0 Exposure Intermittent Percent Time 83 % Environment High 0 F Vacuum	Exposure Intermittent Percent Time 83 % Environment High 0 F Vacuum	Radiation Type: NASA TM 86460 Normal operating Environment: 180°F	
Number 50 Contractor: Lockhead	Bearing in pump of pressure control assembly of water electrolysis system	Bearings	Surface #1 Bearing/A1 Surface #2 Bearing/A1 Surface #3 Bearing/A1 Surface #4 Bearing/A1	Lubrication Scheme Unspecified Lubricant Everbrite 820 SP R 0022A Compatible Yes <input type="radio"/> No <input type="radio"/>	Operating Peak Rotate Speed (rpm) 0 Oscillation Type Unspecified Life (years) 0 Exposure Intermittent Percent Time 0 % Environment High 0 F Vacuum	Exposure Unspecified Percent Time 0 % Environment High 0 F Vacuum	Operating Load: 300 psi, 180°F. Intermittent as req'd	
Number 51 Contractor: Lockhead	Motor pinion gear in cover blanket preload system of solar array of electrical power	Motor pinion gear	Surface #1 Pinion gear/A1 Surface #2 Pinion gear/A1 Surface #3 Pinion gear/A1 Surface #4 Pinion gear/A1	Lubrication Scheme Dry Film Lubricant Everbrite 820 SP R 0022A Compatible Yes <input type="radio"/> No <input type="radio"/>	Operating Peak Rotate Speed (rpm) 0 Oscillation Type Unspecified Life (years) 0 Exposure Intermittent Percent Time 84 % Environment High 0 F Vacuum	Exposure Intermittent Percent Time 84 % Environment High 0 F Vacuum		
Number 52 Contractor: Lockhead	Motor ballbearings in cover blanket preload system of solar array of electrical power	Motor ballbearings	Surface #1 Bearing/8S Surface #2 Bearing/8S Surface #3 Bearing/8S Surface #4 Bearing/8S	Lubrication Scheme Fluid Lubricant Brylcol 801 SP R 0022A Compatible Yes <input type="radio"/> No <input type="radio"/>	Operating Peak Rotate Speed (rpm) 0 Oscillation Type Unspecified Life (years) 0 Exposure Intermittent Percent Time 84 % Environment High 0 F Vacuum	Exposure Intermittent Percent Time 84 % Environment High 0 F Vacuum		
Number 53 Contractor: Lockhead	Latch system in cover blanket preload system of solar array of electrical power	Latch system	Surface #1 Latch pin in latch/A1 Surface #2 Latch pin in latch/A1 Surface #3 Latch pin in latch/A1 Surface #4 Latch pin in latch/A1	Lubrication Scheme Dry Film Lubricant Everbrite 820 SP R 0022A Compatible Yes <input type="radio"/> No <input type="radio"/>	Operating Peak Rotate Speed (rpm) 0 Oscillation Type Unspecified Life (years) 0 Exposure Intermittent Percent Time 84 % Environment High 0 F Vacuum	Exposure Intermittent Percent Time 84 % Environment High 0 F Vacuum		
Number 54 Contractor: Lockhead	Negative springs to rope reel in blanket tensioning and guide wire system in solar array of electrical power	Negative springs to rope reel	Surface #1 Spring/A1 Surface #2 Spring/A1 Surface #3 Spring/A1 Surface #4 Spring/A1	Lubrication Scheme Dry Film Lubricant Everbrite 820 SP R 0022A Compatible Yes <input type="radio"/> No <input type="radio"/>	Operating Peak Rotate Speed (rpm) 0 Oscillation Type Unspecified Life (years) 0 Exposure Intermittent Percent Time 84 % Environment High 0 F Vacuum	Exposure Intermittent Percent Time 84 % Environment High 0 F Vacuum		
Number 55 Contractor: Lockhead	Limit switches in array motors of solar array of electrical power	Limit switches	Surface #1 Pin/8S Surface #2 Pin/8S Surface #3 Pin/8S Surface #4 Pin/8S	Lubrication Scheme Dry Film Lubricant Everbrite 820 SP R 0022A Compatible Yes <input type="radio"/> No <input type="radio"/>	Operating Peak Rotate Speed (rpm) 0 Oscillation Type Unspecified Life (years) 0 Exposure Intermittent Percent Time 84 % Environment High 0 F Vacuum	Exposure Intermittent Percent Time 84 % Environment High 0 F Vacuum		
Number 56 Contractor: Lockhead	Interface/Mast carrier to beta drive in mast carrier of solar array of electrical power	Interface/Mast carrier to beta drive	Surface #1 Mast carrier/8SAA Surface #2 Mast carrier/8SAA Surface #3 Mast carrier/8SAA Surface #4 Mast carrier/8SAA	Lubrication Scheme Dry Film Lubricant Everbrite 820 SP R 0022A Compatible Yes <input type="radio"/> No <input type="radio"/>	Operating Peak Rotate Speed (rpm) 0 Oscillation Type Unspecified Life (years) 0 Exposure Intermittent Percent Time 84 % Environment High 0 F Vacuum	Exposure Intermittent Percent Time 84 % Environment High 0 F Vacuum		
Number 57 Contractor: Lockhead	Motor ballbearings in mast motor drive assembly of solar array of electrical power	Motor ball bearings	Surface #1 Bearing/8S Surface #2 Bearing/8S Surface #3 Bearing/8S Surface #4 Bearing/8S	Lubrication Scheme Fluid Lubricant Brylcol 801 SP R 0022A Compatible Yes <input type="radio"/> No <input type="radio"/>	Operating Peak Rotate Speed (rpm) 0 Oscillation Type Unspecified Life (years) 0 Exposure Intermittent Percent Time 84 % Environment High 0 F Vacuum	Exposure Intermittent Percent Time 84 % Environment High 0 F Vacuum		
Number 58 Contractor: Lockhead	Clutch or solenoid in locking mechanism of beta joint of electrostatic mechanisms	Clutch or solenoid	Surface #1 Clutch/Solenoid Surface #2 Clutch/Solenoid Surface #3 Clutch/Solenoid Surface #4 Clutch/Solenoid	Lubrication Scheme Unspecified Lubricant Everbrite 820 SP R 0022A Compatible Yes <input type="radio"/> No <input type="radio"/>	Operating Peak Rotate Speed (rpm) 0 Oscillation Type Unspecified Life (years) 0 Exposure Intermittent Percent Time 84 % Environment High 0 F Vacuum	Exposure Unspecified Percent Time 0 % Environment High 0 F Vacuum		
Number 59 Contractor: Lockhead	Pin pulley pinion or solenoid pinion in launch support and release for box ends of solar array of electrical power	Pin pulley pinion or solenoid pinion	Surface #1 Pinion/A1 Surface #2 Pinion/A1 Surface #3 Pinion/A1 Surface #4 Pinion/A1	Lubrication Scheme Dry Film Lubricant Everbrite 820 SP R 0022A Compatible Yes <input type="radio"/> No <input type="radio"/>	Operating Peak Rotate Speed (rpm) 0 Oscillation Type Unspecified Life (years) 0 Exposure Intermittent Percent Time 84 % Environment High 0 F Vacuum	Exposure Intermittent Percent Time 84 % Environment High 0 F Vacuum		
Number 60 Contractor: Lockhead	Bearings in centrifugal pump of V2 pump/separator of fluid control system	Bearings	Surface #1 Bearing/A1 Surface #2 Bearing/A1 Surface #3 Bearing/A1 Surface #4 Bearing/A1	Lubrication Scheme Unspecified Lubricant Everbrite 820 SP R 0022A Compatible Yes <input type="radio"/> No <input type="radio"/>	Operating Peak Rotate Speed (rpm) 0 Oscillation Type Unspecified Life (years) 0 Exposure Intermittent Percent Time 84 % Environment High 0 F Vacuum	Exposure Unspecified Percent Time 0 % Environment High 0 F Vacuum	Operating Load: 80 psi, 140°F. Continuous	

Survey Number	Function	Description	Contact	Lubrication		Load, Life, and Motion		Surf Temperature (deg F)		Operating Conditions	Comment
				Lubrication Scheme: Unspecified	Lubricant	Operating Peak	Life (years)	Exposure Intermittent	Exposure Percent Time		
Number 62 Contractor: Lockheed	Deployer guide in mast carrier of solar array of electrical power	Deployer guide	Surface #1 Fined dep. guide/ Surface Finish Unspecified Surface #2 Mast str. roller/ Surface Finish Unspecified Surface #3 Unspecified	None Yes <input type="radio"/> No <input checked="" type="radio"/>	Operating Peak Rotation Speed (rpm) Excitation Type	0 0 Unspecified	10 0 Unspecified	0 0 0	6.4 % 0 0		
Number 61 Contractor: Lockheed	Bearings in pump of coolant pump of fluid control system	Bearings	Surface #1 Surface #2 Surface #3	Lubrication Scheme: Unspecified SP R 0022A Compatible Yes <input type="radio"/> No <input checked="" type="radio"/>	Operating Peak Rotation Speed (rpm) Excitation Type	0 0 Unspecified	10 0 Unspecified	0 0 0	0 % 0 0	Operating Load 60 psi, 180°F, continuous	
Number 63 Contractor: Lockheed	Bearings in pump of coolant control assembly of water electrolysis system	Bearings	Surface #1 Surface #2 Surface #3	Lubrication Scheme: Unspecified SP R 0022A Compatible Yes <input type="radio"/> No <input checked="" type="radio"/>	Operating Peak Rotation Speed (rpm) Excitation Type	0 0 Unspecified	10 0 Unspecified	0 0 0	0 % 0 0	Operating Load 300 psi, 180°F, continuous	Pumps waste or burnout through electrolyzer
Number 64 Contractor: Lockheed	Interface/containment box to mast in containment box of solar array of electrical power	Interface/containment box to mast	Surface #1 Contain box/SS & Al/ Surface Finish Unspecified Surface #2 Mast/Al/ Surface Finish Unspecified Surface #3 Unspecified	Lubrication Scheme: Dry Film Lubricant Elevate 820 Yes <input type="radio"/> No <input checked="" type="radio"/>	Operating Peak Rotation Speed (rpm) Excitation Type	0 0 Unspecified	10 0 Unspecified	0 0 0	0 % 0 0		
Number 65 Contractor: Lockheed	Motor drive pinion gear in array containment box positioning system of solar array of electrical power	Motor drive pinion gear	Surface #1 Surface #2 Surface #3	Lubrication Scheme: Dry Film SP R 0022A Compatible Yes <input type="radio"/> No <input checked="" type="radio"/>	Operating Peak Rotation Speed (rpm) Excitation Type	0 0 Unspecified	10 0 Unspecified	0 0 0	6.4 % 0 0		Motor drive pinion or worm gear/Al.
Number 66 Contractor: Lockheed	Positioner system in array containment box positioning system of solar array of electrical power	Positioner system	Surface #1 pins in pos. sys/Al/ Surface Finish Unspecified Surface #2 Bushing in p l/Al/ Surface Finish Unspecified Surface #3 Unspecified	Lubrication Scheme: Dry Film Lubricant Elevate 820 Yes <input type="radio"/> No <input checked="" type="radio"/>	Operating Peak Rotation Speed (rpm) Excitation Type	0 0 Unspecified	10 0 Unspecified	0 0 0	6.4 % 0 0		
Number 67 Contractor: Lockheed	Motor ball bearings in array containment box positioning system of solar array of electrical power	Motor ball bearings	Surface #1 Bearings/SS Surface #2 Race/SS Surface #3 Unspecified	Lubrication Scheme: Fluid Lubricant Brylube 601 Yes <input type="radio"/> No <input checked="" type="radio"/>	Operating Peak Rotation Speed (rpm) Excitation Type	0 0 Unspecified	10 0 Unspecified	0 0 0	6.4 % 0 0		Grease
Number 68 Contractor: Lockheed	Latch and release mechanism in array containment box positioning system of solar array of electrical power	Latch and release mechanism	Surface #1 Latch/Al/ Surface Finish Unspecified Surface #2 Release/Al/ Surface Finish Unspecified	Lubrication Scheme: Dry Film Lubricant Elevate 820 Yes <input type="radio"/> No <input checked="" type="radio"/>	Operating Peak Rotation Speed (rpm) Excitation Type	0 0 Unspecified	10 0 Unspecified	0 0 0	6.4 % 0 0		Mechanism holds box position
Number 69 Contractor: Lockheed	Box-to-mast hinges system in array containment box positioning system of solar array of electrical power	Box-to-mast hinges system	Surface #1 Pin/Al/ Surface Finish Unspecified Surface #2 Bushing/Al/ Surface Finish Unspecified Surface #3 Unspecified	Lubrication Scheme: Dry Film Lubricant Elevate 820 Yes <input type="radio"/> No <input checked="" type="radio"/>	Operating Peak Rotation Speed (rpm) Excitation Type	0 0 Unspecified	10 0 Unspecified	0 0 0	6.4 % 0 0		In box-to-mast hinges system
Number 70 Contractor: Lockheed	Pinion gear in mast carrier of solar array of electrical power	Pinion gear	Surface #1 Surface #2 Surface #3	Lubrication Scheme: Dry Film Lubricant Elevate 820 Yes <input type="radio"/> No <input checked="" type="radio"/>	Operating Peak Rotation Speed (rpm) Excitation Type	0 0 Unspecified	10 0 Unspecified	0 0 0	6.4 % 0 0		Deployer nut ring gear Al Drive assembly pinion gear/SS
Number 71 Contractor: Lockheed	Pinion gear in mast carrier of solar array of electrical power	Pinion gear	Surface #1 Surface #2 Surface #3	Lubrication Scheme: Dry Film Lubricant Elevate 820 Yes <input type="radio"/> No <input checked="" type="radio"/>	Operating Peak Rotation Speed (rpm) Excitation Type	0 0 Unspecified	10 0 Unspecified	0 0 0	6.4 % 0 0		Deployer nut ring gear/Al Drive assembly pinion gear/SS
Number 72 Contractor: Lockheed	Cables in rope reel in hoist lowering and guidance wire system of solar array of electrical power	Cables to rope reel	Surface #1 Hoist/Al or Dextrin Surface #2 Airtel/Al Surface #3 Unspecified	Lubrication Scheme: Dry Film Lubricant Elevate 820 Yes <input type="radio"/> No <input checked="" type="radio"/>	Operating Peak Rotation Speed (rpm) Excitation Type	0 0 Unspecified	10 0 Unspecified	0 0 0	6.4 % 0 0		none for Dextrin hub

4.0 REFERENCES

- Dolan, F., "Space Station Long Term Lubrication Studies Contract, 'Kickoff Meeting", Presentation Charts, National Aeronautics and Space Administration, MSFC/EH14, August 1985.
- "Space Station Long Term Lubrication, Orientation Briefing" Presentation Charts, Battelle Columbus Laboratories and SRS Technologies, August 28, 1985.
- "Space Station Reference Configuration Description", JSC-19989.
- "Space Station Subsystem White Papers", JSC-20054.
- "General Specification Vacuum Stability Requirements of Polymeric Material for Spacecraft Application", SP-R-0022A.

Appendix A

Survey Forms



**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

1

Contractor Sperry Survey Number 1

WBS Number 1116112224131 Survey Date 6/18/86

Function SHAFT BEARING DRIVE ASSEMBLY

Description Shaft Bearing P/N 3HKR102P7

Surface #1

Surface #2

Contact Surface 440 C

Contact Surface 440 c

Surface Finish Set by Sperry

Surface Finish Set by Sperry

Surface Treatment _____

Surface Treatment _____

Lubrication Scheme Fluid

SP R 0022A Compatible?

Required Life

Lubricant Micronic 601

Yes No

20 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 11 lb Peak 20 lb Preload 1 lb

Rotational Speed

Oscillation None

0.5 rpm

Frequency 0 cycles/sec

Amplitude 0 inches

Sun Exposure None

Low Temperature 8 degrees F

% Time 0

High Temperature 105 degrees F

Environment

Gas _____

Vacuum _____

Liquid _____

Operating Conditions

DN value at 8 Rpm-min.

Non-Operating Conditions

Comment

Single 15mm bore ball bearing (catalog number 3HKR102P7 by MPB Corp.). 400 lb max static load capacity. Cage is Conrad machined phenolic.

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

2

Contractor Rocketdyne Survey Number 2

WBS Number 16.2.2.1 Survey Date 8/15/86

Function Solar array deployment.

Description Bearings in PV array deployment motor

Surface #1		Surface #2	
Contact Surface	<u>Unspecified</u>	Contact Surface	<u>Unspecified</u>
Surface Finish	<u></u>	Surface Finish	<u></u>
Surface Treatment	<u></u>	Surface Treatment	<u></u>

Lubrication Scheme <u>Fluid</u>	SP R 0022A Compatible?	Required Life
Lubricant <u>Braycote 601</u>	<input type="radio"/> Yes <input checked="" type="radio"/> No	<u>10</u> years

Total Mass Loss 0 % CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed <u>1200</u> rpm	Oscillation <u>Unspecified</u>
Frequency <u>0</u> cycles/sec	Amplitude <u>0</u> inches

Sun Exposure <u>Continuous</u>	Low Temperature <u>-112</u> degrees F
% Time <u>100</u>	High Temperature <u>176</u> degrees F

Environment	Gas <u></u>
<u>Vacuum</u>	Liquid <u></u>

Operating Conditions

Non-Operating Conditions

Comment

STS Cargo Bay vibration qualification level.

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

3

Contractor Rocketdyne Survey Number 3

WBS Number 16.2.2.1 Survey Date 8/15/86

Function Rotating nut for solar array mast extension

Description PV array mast canister rotating nut

Surface #1

Surface #2

Contact Surface Unspecified

Contact Surface Unspecified

Surface Finish _____

Surface Finish _____

Surface Treatment _____

Surface Treatment _____

Lubrication Scheme Dry Film

SP R 0022A Compatible?

Required Life

Lubricant Everlube 620

Yes No

10 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed

Oscillation Unspecified

2 rpm

Frequency 0 ^{cycles/}sec Amplitude 0 inches

Sun Exposure Intermittent

Low Temperature -112 degrees F

% Time 50

High Temperature 176 degrees F

Environment

Gas _____

Vacuum

Liquid _____

Operating Conditions

Non-Operating Conditions

Comment

STS Cargo Bay qualification vibration level

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

4

Contractor Rocketdyne Survey Number 4

WBS Number _____ Survey Date 8/13/86

Function Provide fine pointing actuation of Solar Dynamic Concentrator reflector.

Description Actuator motor bearings, threads, nuts

Surface #1

Surface #2

Contact Surface TBD steel

Contact Surface TBD steel

Surface Finish TBD

Surface Finish TBD

Surface Treatment TBD

Surface Treatment TBD

Lubrication Scheme Fluid

SP R 0022A Compatible?

Required Life

Lubricant TBD

Yes No

10 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 100 lb Peak 200 lb Preload 0 lb

Rotational Speed

Oscillation Intermittent

60 rpm

Frequency 2 ^{cycles/}sec Amplitude 0 inches

Sun Exposure None Low Temperature -140 degrees F

% Time 0 High Temperature 140 degrees F

Environment

Gas _____

Vacuum _____

Liquid _____

Operating Conditions <.1 Hz continuous oscillation.

Non-Operating Conditions NSTS Cargo Bat Environments

Comment for additional information, contact Sundstrans Energy Systems, Rockford, Illinois, 61125. Dual failure tolerant electromechanical actuator.

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

5

Contractor Rocketdyne Survey Number 5

WBS Number _____ Survey Date 8/13/86

Function Solar Dynamic Concentrator deployment latch catch mechanism.

Description Catch Journal

Surface #1

Surface #2

Contact Surface TBD

Contact Surface TBD

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Dry Film

SP R 0022A Compatible?

Required Life

Lubricant TBD

Yes No

15 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed
0 rpm

Oscillation Intermittent

Frequency 0 cycles/sec Amplitude 0 inches

Sun Exposure Intermittent
% Time 70

Low Temperature 0 degrees F

High Temperature 0 degrees F

Environment

Gas _____

Vacuum

Liquid _____

Operating
Conditions

Non-Operating
Conditions

Comment

Required to function only during deployment or retraction of concentrator. Journal bearing with <0.5 inch radius

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

6

Contractor Rocketdyne Survey Number 6

WBS Number _____ Survey Date 8/13/86

Function Fine Pointing for the Solar Dynamic Concentrator

Description Universal Joint Journal Bearings

Surface #1

Surface #2

Contact Surface TBD Al

Contact Surface TBD Al

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Dry Film

SP R 0022A Compatible?

Required Life

Lubricant Unspecified

Yes No

15 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed

0.16667 rpm

Oscillation TBD

Frequency 0 cycles/sec

Amplitude 0 inches

Sun Exposure Intermittent

Low Temperature -150 degrees F

% Time 70

High Temperature 150 degrees F

Environment

Gas _____

Vacuum

Liquid _____

Operating Conditions

Continuous, low load, low velocity, random motion.

Non-Operating Conditions

Comment

Journal bearing radius given as "TBD 2 inches".

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

7

Contractor Rocketdyne Survey Number 7

WBS Number _____ Survey Date 8/13/86

Function Concentrator Reflector Deployment

Description Hinge Needle Bearing

Surface #1

Surface #2

Contact Surface TBD

Contact Surface TBD

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Dry Film

SP R 0022A Compatible?

Required Life

Lubricant Unspecified

Yes No

15 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 100 lb Peak 150 lb Preload 0 lb

Rotational Speed

Oscillation None

0.16667 rpm

Frequency 0 cycles/sec

Amplitude 0 inches

Sun Exposure Intermittent

Low Temperature -150 degrees F

% Time 70

High Temperature 150 degrees F

Environment

Gas _____

Vacuum _____

Liquid _____

Operating Conditions

Required to function only during deployment and retraction on-orbit.

Non-Operating Conditions

Comment

Roller bearing

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

8

Contractor Rocketdyne Survey Number 8

WBS Number _____ Survey Date 8/15/86

Function Alternator foil journal and thrust bearings for the Closed Brayton Cycle Solar Dynamic Power Generation Power Conversion Unit.

Description Rotor/Housing Interface

Surface #1

Surface #2

Contact Surface Rotor Journal

Contact Surface Foil Leaves

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Fluid SP R 0022A Compatible? Required Life
 Lubricant Helium/Xenon (MW=40) Yes No 30 years

Total Mass Loss 0 % CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed 32000 rpm Oscillation Continuous
 Frequency 1060 cycles/sec Amplitude 0 inches

Sun Exposure None Low Temperature 100 degrees F
 % Time 0 High Temperature 300 degrees F

Environment Gas Helium/Xenon (MW=40)
 Pressurized _____ Liquid _____

Operating Conditions Rotor lubricant inventory is approximately 4 lbs of lubricant. Operating load is given as "approximately 100 kW Shaft". Pressure in housing is 20 to 60 psi.

Non-Operating Conditions

Comment

Journal Bearing with 2.726 inch radius operating at a 760 fps relative velocity. Volatile fluid is contained in welded housing. Extreme operating conditions may include temps -100 deg F to 400 deg F.

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

9

Contractor Rocketdyne Survey Number 9

WBS Number _____ Survey Date 8/15/86

Function Pump to circulate radiator fluid. The Radiator Coolant Pump Package of the Solar Dynamic Power Generation System has an integrated motor shaft/impeller Centrifugal Pump.

Description Hydrodynamic Journal Bearings

Surface #1

Surface #2

Contact Surface Motor Shaft

Contact Surface Housing Journal

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Fluid

SP R 0022A Compatible?

Required Life

Lubricant FC75 Immersion

Yes No

20 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed

Oscillation Continuous

0 rpm

Frequency 0 cycles/sec

Amplitude 0 inches

Sun Exposure None Low Temperature 0 degrees F

% Time 0

High Temperature 100 degrees F

Environment

Gas _____

Pressurized

Liquid

FC75 Immersion

Operating Conditions

High temperature of 200 deg F in extreme operating conditions. Operating load given as "2 lb/sec 60 psid".

Non-Operating Conditions

Low temperature extreme of -100 deg F in non-operating mode.

Comment

Hundreds of pounds of volatile coolant/lubricant contained in sealed coolant loop at 10-20 psi. A Moog pump qualified for Peacekeeper program under consideration

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

10

Contractor Rocketdyne Survey Number 10

WBS Number _____ Survey Date 8/4/86

Function To position the Propulsion System thruster assembly solenoid valve when solenoid is not engaged.

Description Ends of spring at shaft and spring stop

Surface #1		Surface #2	
Contact Surface	<u>Unspecified</u>	Contact Surface	<u>Unspecified</u>
Surface Finish	<u>Unspecified</u>	Surface Finish	<u>Unspecified</u>
Surface Treatment	<u>Unspecified</u>	Surface Treatment	<u>Unspecified</u>

Lubrication Scheme Unspecified SP R 0022A Compatible? Yes No Required Life 0 years
 Lubricant Unspecified

Total Mass Loss 0 % CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed 0 rpm Oscillation Unspecified
 Frequency 0 cycles/sec Amplitude 0 inches

Sun Exposure Unspecified Low Temperature 0 degrees F
 % Time 0 High Temperature 0 degrees F

Environment Gas _____
Unspecified Liquid _____

Operating Conditions _____

Non-Operating Conditions _____

Comment _____

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

11

Contractor Rocketdyne Survey Number 11

WBS Number _____ Survey Date 8/5/86

Function Fluid connector for Propulsion System thruster assembly.

Description Screw threads on propellant feed lines

Surface #1

Surface #2

Contact Surface Male connector

Contact Surface Female connector

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Unspecified

SP R 0022A Compatible?

Required Life

Lubricant Unspecified

Yes No

0 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed

Oscillation Unspecified

0 rpm

Frequency 0 cycles/sec

Amplitude 0 inches

Sun Exposure Unspecified

Low Temperature 0 degrees F

% Time 0

High Temperature 0 degrees F

Environment

Gas _____

Unspecified

Liquid _____

Operating
Conditions

Non-Operating
Conditions

Comment

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

12

Contractor Rocketdyne Survey Number 12

WBS Number _____ Survey Date 8/5/86

Function Affect motion of the valve shaft on the Propulsion System thruster assembly solenoid valve.

Description Electromagnetic Plates

Surface #1

Surface #2

Contact Surface Fixed Plate

Contact Surface Moving Plate

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Unspecified

SP R 0022A Compatible?

Required Life

Lubricant Unspecified

Yes No

0 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed

Oscillation Unspecified

0 rpm

Frequency 0 cycles/sec

Amplitude 0 inches

Sun Exposure Unspecified

Low Temperature 0 degrees F

% Time 0

High Temperature 0 degrees F

Environment

Gas _____

Unspecified

Liquid _____

Operating Conditions

Non-Operating Conditions

Comment

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

13

Contractor Rocketdyne Survey Number 13

WBS Number _____ Survey Date 8/5/86

Function Permit or restrict fluid flow in the solenoid valve head of the Propulsion System thruster assembly.

Description Valve Seats/Stems

Surface #1

Surface #2

Contact Surface Valve Seat

Contact Surface Valve Stem

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Unspecified

SP R 0022A Compatible?

Required Life

Lubricant Unspecified

Yes No

0 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed

Oscillation Unspecified

0 rpm

Frequency

0 cycles/sec

Amplitude 0 inches

Sun Exposure Unspecified

Low Temperature 0 degrees F

% Time 0

High Temperature 0 degrees F

Environment

Gas _____

Unspecified

Liquid _____

Operating Conditions

Non-Operating Conditions

Comment

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

14

Contractor Rocketdyne Survey Number 14

WBS Number _____ Survey Date 8/5/86

Function Guide motion of valve stem in solenoid valve of the Propulsion System thruster assembly.

Description Valve Shaft Guides

Surface #1		Surface #2	
Contact Surface	<u>Valve Stem</u>	Contact Surface	<u>Valve Shaft Guides</u>
Surface Finish	<u>Unspecified</u>	Surface Finish	<u>Unspecified</u>
Surface Treatment	<u>Unspecified</u>	Surface Treatment	<u>Unspecified</u>

Lubrication Scheme Unspecified SP R 0022A Compatible? Yes No Required Life _____
 Lubricant Unspecified _____ 0 years

Total Mass Loss 0 % CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed 0 rpm Oscillation Unspecified
 Frequency 0 cycles/sec Amplitude 0 inches

Sun Exposure Unspecified Low Temperature 0 degrees F
 % Time 0 High Temperature 0 degrees F

Environment Gas _____

 Unspecified Liquid _____

Operating Conditions _____

Non-Operating Conditions _____

Comment _____

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

15

Contractor Rocketdyne Survey Number 15

WBS Number _____ Survey Date 00/00/00

Function To form a connection between tubing and the dummy spark igniter in the Propulsion SYstem thruster assembly.

Description Dummy spark igniter/tubing connection

Surface #1

Surface #2

Contact Surface Tygon tubing

Contact Surface Dummy Spark Igniter

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Unspecifie SP R 0022A Compatible? Yes No Required Life 0 years
Lubricant Unspecified

Total Mass Loss 0 % CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed 0 rpm Oscillation Unspecified
Frequency 0 cycles/sec Amplitude 0 inches

Sun Exposure Unspecified Low Temperature 0 degrees F
% Time 0 High Temperature 0 degrees F

Environment Gas _____
Unspecified Liquid _____

Operating Conditions

Non-Operating Conditions

Comment

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

16

Contractor Rocketdyne Survey Number 16

WBS Number _____ Survey Date 8/5/86

Function hardare attachment device. Hex head bolt asembly in the valve bracket assembly of the Propulsion SYstem thruster assembly.

Description Screw threads/bolt

	Surface #1		Surface #2
Contact Surface	<u>Screw threads</u>	Contact Surface	<u>Hex head bolt</u>
Surface Finish	_____	Surface Finish	<u>Unspecified</u>
Surface Treatment	_____	Surface Treatment	<u>Unspecified</u>

Lubrication Scheme Fluid SP R 0022A Compatible? Required Life
 Lubricant MoS2 Yes No _____ 0 years

Total Mass Loss _____ 0 % CVCM _____ 0 %

Loads: Operating _____ 0 lb Peak _____ 0 lb Preload _____ 0 lb

Rotational Speed _____ 0 rpm Oscillation Unspecified
 Frequency _____ 0 cycles/sec Amplitude _____ 0 inches

Sun Exposure Unspecified Low Temperature _____ 0 degrees F
 % Time _____ 0 High Temperature _____ 0 degrees F

Environment Gas _____
 Unspecified Liquid _____

Operating Conditions _____

Non-Operating Conditions _____

Comment _____

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

17

Contractor Rocketdyne Survey Number 17

WBS Number _____ Survey Date 8/4/86

Function hardware attachment device. screw/bolt assembly in the Propulsion System thruster assembly power head.

Description _____

Surface #1

Surface #2

Contact Surface Screw threads

Contact Surface Screw Cap Socket

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Fluid

SP R 0022A Compatible?

Required Life

Lubricant MoS2

Yes No

0 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb

Peak 0 lb

Preload 0 lb

Rotational Speed

0 rpm

Oscillation Unspecified

Frequency 0 cycles/sec

Amplitude 0 inches

Sun Exposure Unspecified

Low Temperature 0 degrees F

% Time 0

High Temperature 0 degrees F

Environment

Gas _____

Unspecified

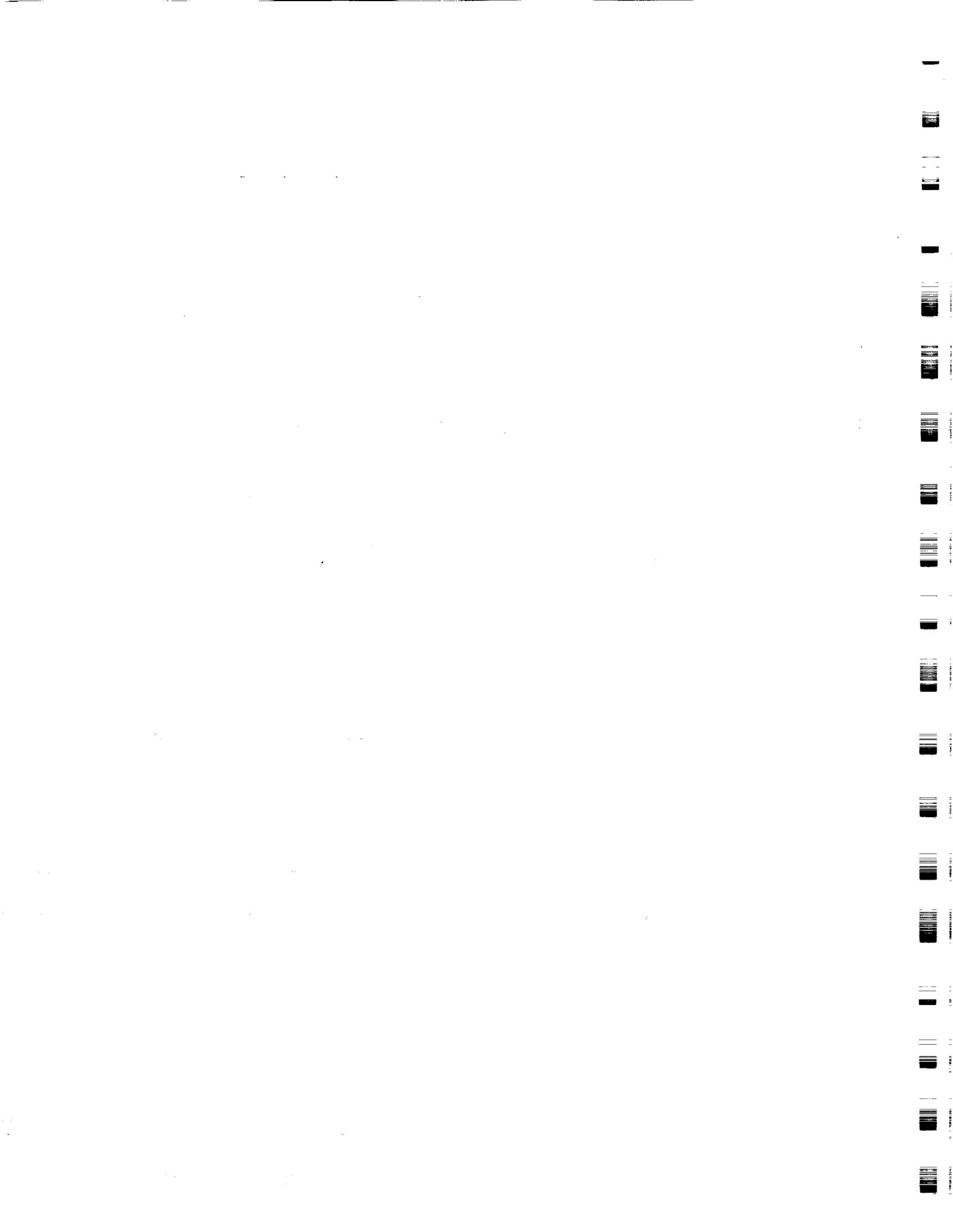
Liquid _____

Operating Conditions

Non-Operating Conditions

Comment

Socket torqued to 22 (+/- 1) inch-lbs.



**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

18

Contractor Rocketdyne Survey Number 18

WBS Number _____ Survey Date 8/6/86

Function Swagelock to a union (CRES) in propellant fluid line connectors of the Propulsion System thruster assembly.

Description contact surfaces and screw threads

Surface #1

Surface #2

Contact Surface Screw threads

Contact Surface Unspecified

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Unspecified

SP R 0022A Compatible?

Required Life

Lubricant Unspecified

Yes No

0 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed

Oscillation Unspecified

0 rpm

Frequency 0 cycles/sec

Amplitude 0 inches

Sun Exposure Unspecified

Low Temperature 0 degrees F

% Time 0

High Temperature 0 degrees F

Environment

Gas _____

Unspecified

Liquid _____

Operating Conditions

Non-Operating Conditions

Comment

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

19

Contractor Rocketdyne Survey Number 19

WBS Number _____ Survey Date 8/5/86

Function To seal the fuel line connection between the fuel solenoid valve and the valve assembly bracket in the Propulsion System thruster assembly.

Description V Seal

Surface #1

Surface #2

Contact Surface V-Seal-Inco 718

Contact Surface Valve Assy Bracket

Surface Finish Teflon Coating

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Dry Film

SP R 0022A Compatible?

Required Life

Lubricant Teflon

Yes No

0 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed

Oscillation Unspecified

0 rpm

Frequency 0 cycles/sec

Amplitude 0 inches

Sun Exposure Unspecified

Low Temperature 0 degrees F

% Time 0

High Temperature 0 degrees F

Environment

Gas _____

Unspecified

Liquid _____

Operating
Conditions

Non-Operating
Conditions

Comment

SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM

20

Contractor Rocketdyne Survey Number 20

WBS Number _____ Survey Date 8/5/86

Function Seal the connection between the Propulsion system thruster assembly power head and combustion chamber.

Description V Seal

Surface #1

Surface #2

Contact Surface V-Seal-Inco 718

Contact Surface Power Head

Surface Finish Modified Gold Plated

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Unspecifie

SP R 0022A Compatible?

Required Life

Lubricant Unspecified

Yes No

0 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed

Oscillation Unspecified

0 rpm

Frequency 0 cycles/sec

Amplitude 0 inches

Sun Exposure Unspecified

Low Temperature 0 degrees F

% Time 0

High Temperature 0 degrees F

Environment

Gas

Unspecified

Liquid

Operating Conditions

Non-Operating Conditions

Comment

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

21

Contractor Rocketdyne Survey Number 21

WBS Number _____ Survey Date 8/5/86

Function Seal the threaded connection between the Dummy Spark Igniter and the Power Head in the Propulsion System thruster assembly.

Description K Seal

Surface #1

Surface #2

Contact Surface K Seal

Contact Surface Power Head

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Unspecified

SP R 0022A Compatible?

Required Life

Lubricant Unspecified

Yes No

0 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed
0 rpm

Oscillation Unspecified

Frequency 0 cycles/sec Amplitude 0 inches

Sun Exposure Unspecified
% Time 0

Low Temperature 0 degrees F

High Temperature 0 degrees F

Environment

Gas _____

Unspecified

Liquid _____

Operating
Conditions

Lubricant is under pressure at the contact surface and in vacuum at the outer edge of the contact surface.

Non-Operating
Conditions

Comment

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

22

Contractor Rocketdyne Survey Number 22

WBS Number _____ Survey Date 8/4/86

Function "O" ring seal for the threaded connector on the Dummy Spark Igniter of the Propulsion System thruster assembly.

Description Two flat ring seals

Surface #1

Surface #2

Contact Surface Unspecified

Contact Surface Unspecified

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Unspecifie

SP R 0022A Compatible?

Required Life

Lubricant Unspecified

Yes No

0 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed

Oscillation Unspecified

0 rpm

Frequency 0 cycles/sec

Amplitude 0 inches

Sun Exposure Unspecified

Low Temperature 0 degrees F

% Time 0

High Temperature 0 degrees F

Environment

Gas _____

Unspecified

Liquid _____

Operating Conditions

Non-Operating Conditions

Comment

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

23

Contractor Moog Survey Number 23

WBS Number _____ Survey Date 8/6/86

Function Valve guides for the Oxygen Solenoid Valve of Propulsion system

Description Poppet/Plunger Teflon Guides

Surface #1

Surface #2

Contact Surface Poppet

Contact Surface Plunger

Surface Finish Teflon Coating

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Dry Film

SP R 0022A Compatible?

Required Life

Lubricant Teflon

Yes No

0 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed
0 rpm

Oscillation Unspecified

Frequency 0 cycles/sec Amplitude 0 inches

Sun Exposure Unspecified

Low Temperature 0 degrees F

% Time 0

High Temperature 0 degrees F

Environment

Gas _____

Unspecified

Liquid _____

Operating
Conditions

Non-Operating
Conditions

Comment

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

24

Contractor Moog Survey Number 24

WBS Number _____ Survey Date 8/6/86

Function Position the Oxygen Solenoid Valve of the ARPCS control system in the Space Station Propulsion System when the solenoid is not engaged.

Description Low stress helical spring

Surface #1

Surface #2

Contact Surface Helical Spring

Contact Surface Valve Seat

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Unspecified

SP R 0022A Compatible?

Required Life

Lubricant Unspecified

Yes No

10 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed

0 rpm

Oscillation Unspecified

Frequency

0 cycles/sec

Amplitude

0 inches

Sun Exposure None

Low Temperature

0 degrees F

% Time 0

High Temperature

0 degrees F

Environment

Gas

Unspecified

Liquid

Operating Conditions

Non-Operating Conditions

Comment

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

25

Contractor Moog Survey Number 25

WBS Number _____ Survey Date 8/6/86

Function Valve seat for the Oxygen Solenoid Valve of the ARPCS control system in the Space Station Propulsion system.

Description Valve head / seat contact

Surface #1		Surface #2	
Contact Surface	<u>Valve seat</u>	Contact Surface	<u>Valve Head</u>
Surface Finish	<u>Unspecified</u>	Surface Finish	<u>Unspecified</u>
Surface Treatment	<u>Unspecified</u>	Surface Treatment	<u>Unspecified</u>

Lubrication Scheme Dry Film SP R 0022A Compatible? Yes No Required Life 10 years

Lubricant Polyimide (Vespel)

Total Mass Loss 0 % CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed 0 rpm Oscillation Unspecified

Frequency 0 cycles/sec Amplitude 0 inches

Sun Exposure None Low Temperature 0 degrees F

% Time 0 High Temperature 0 degrees F

Environment Gas _____

Unspecified Liquid _____

Operating Conditions

Non-Operating Conditions

Comment

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

26

Contractor Moog Survey Number 26

WBS Number _____ Survey Date 8/18/86

Function Seal Oxygen Solenoid Valve Components in the ARPCS control system of the Space Station Propulsion system.

Description Seven O-rings

Surface #1

Surface #2

Contact Surface O Ring

Contact Surface Various

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Unspecified

SP R 0022A Compatible?

Required Life

Lubricant Unspecified

Yes No

10 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed
0 rpm

Oscillation Unspecified

Frequency 0 cycles/sec Amplitude 0 inches

Sun Exposure None
% Time 0

Low Temperature 0 degrees F

High Temperature 0 degrees F

Environment Gas Oxygen

Pressurized Liquid LOX

Operating Conditions

Non-Operating Conditions

Comment

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

27

Contractor Moog Survey Number 27

WBS Number _____ Survey Date 8/19/86

Function Connector in the Oxygen Solenoid Valve of the Space Station Propulsion System

Description Screw Threads

Surface #1		Surface #2	
Contact Surface	<u>Screw threads</u>	Contact Surface	<u>Thread Grooves</u>
Surface Finish	<u>Unspecified</u>	Surface Finish	<u>Unspecified</u>
Surface Treatment	<u>Unspecified</u>	Surface Treatment	<u>Unspecified</u>

Lubrication Scheme Unspecified SP R 0022A Compatible? Yes No Required Life 10 years

Lubricant Unspecified

Total Mass Loss 0 % CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed 0 rpm Oscillation Unspecified
 Frequency 0 cycles/sec Amplitude 0 inches

Sun Exposure Unspecified Low Temperature 0 degrees F
 % Time 0 High Temperature 0 degrees F

Environment Gas _____

 Unspecified Liquid _____

Operating Conditions _____

Non-Operating Conditions _____

Comment _____

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

28

Contractor Moog Survey Number 28

WBS Number _____ Survey Date 8/15/86

Function O-Ring seal between latching mechanism casing and plunger mechanism of the Oxygen and Nitrogen Latching Valve in th ARPCS Control System of the Space Station Propulsion System

Description O- ring seal

Surface #1

Surface #2

Contact Surface Casing mechanism

Contact Surface plunger mechanism

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Unspecifie

SP R 0022A Compatible?

Required Life

Lubricant Unspecified

Yes No

10 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed

Oscillation Unspecified

0 rpm

Frequency 0 cycles/sec

Amplitude 0 inches

Sun Exposure None

Low Temperature 0 degrees F

% Time 0

High Temperature 0 degrees F

Environment

Gas

Pressurized

Liquid

Operating Conditions

Non-Operating Conditions

Comment

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

29

Contractor Moog Survey Number 29

WBS Number _____ Survey Date 8/15/86

Function Assembly and Adjustment threaded connectors in the Oxygen and Nitrogen Latching Valve of the ARPCS Control System in the Space Station Propulsion System

Description Scre Threads

Surface #1		Surface #2	
Contact Surface	<u>Screw Threads</u>	Contact Surface	<u>Thread Grooves</u>
Surface Finish	<u>Unspecified</u>	Surface Finish	<u>Unspecified</u>
Surface Treatment	<u>Unspecified</u>	Surface Treatment	<u>Unspecified</u>

Lubrication Scheme Unspecified SP R 0022A Compatible? Yes No Required Life 10 years
 Lubricant Unspecified

Total Mass Loss 0 % CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed 0 rpm Oscillation Unspecified
 Frequency 0 cycles/sec Amplitude 0 inches

Sun Exposure Unspecified Low Temperature 0 degrees F
 % Time 0 High Temperature 0 degrees F

Environment Gas _____
 _____ Liquid _____

Operating Conditions _____

Non-Operating Conditions _____

Comment _____

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

30

Contractor Moog Survey Number 30

WBS Number _____ Survey Date 8/15/86

Function Filter Seal in the Oxygen and Nitrogen Latching Valve in the ARPCS Control System of the Space Station Propulsion System

Description Surface contact

Surface #1

Surface #2

Contact Surface filter seal

Contact Surface filter seat

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Unspecifie

SP R 0022A Compatible?

Required Life

Lubricant Unspecified

Yes No

10 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed
0 rpm

Oscillation Unspecified

Frequency 0 cycles/sec Amplitude 0 inches

Sun Exposure None

Low Temperature 0 degrees F

% Time 0

High Temperature 0 degrees F

Environment

Gas _____

Pressurized

Liquid _____

Operating Conditions

Non-Operating Conditions

Comment

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

31

Contractor Moog Survey Number 31

WBS Number _____ Survey Date 8/13/86

Function O Ring between end cap mechnism and Valve Adjustment mechanism in the Oxygen and Nitrogen Latching Valve of the ARPCS Control System in the Space Station Propulsion system

Description O-Ring

Surface #1

Surface #2

Contact Surface O-Ring

Contact Surface End cap/Valve adjust

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Unspecifie

SP R 0022A Compatible?

Required Life

Lubricant Unspecified

Yes No

10 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed
0 rpm

Oscillation Unspecified

Frequency 0 cycles/sec Amplitude 0 inches

Sun Exposure None Low Temperature 0 degrees F

% Time 0 High Temperature 0 degrees F

Environment

Gas

Pressurized

Liquid

Operating
Conditions

Non-Operating
Conditions

Comment

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

32

Contractor Moog Survey Number 32

WBS Number _____ Survey Date 8/12/86

Function Seal the Oxygen & Nitrogen Latching Valve casing from the electric drive motor in the ARPCS COntro System of the Space Station Propulsion System

Description Belleville Spring Seal

Surface #1

Surface #2

Contact Surface Bville Spring Inner

Contact Surface Bville Spring Outer

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Unspecifie

SP R 0022A Compatible?

Required Life

Lubricant Unspecified

Yes No

0 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed

0 rpm

Oscillation Unspecified

Frequency 0 cycles/sec

Amplitude 0 inches

Sun Exposure Unspecified

Low Temperature 0 degrees F

% Time 0

High Temperature 0 degrees F

Environment

Gas

Unspecified

Liquid

Operating Conditions

Non-Operating Conditions

Comment

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

33

Contractor Moog Survey Number 33

WBS Number _____ Survey Date 00/00/00

Function O Ring to seal the Relief Valve to the motor case in the ARPCS Control System Oxygen and Nitrogen Latching Valve of Space Station Propulsion System

Description O-Ring

Surface #1

Surface #2

Contact Surface O-Ring
Surface Finish Unspecified
Surface Treatment Unspecified

Contact Surface Relief Valve
Surface Finish Unspecified
Surface Treatment Unspecified

Lubrication Scheme Unspecified SP R 0022A Compatible? Required Life
Lubricant Unspecified Yes No 10 years

Total Mass Loss 0 % CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed 0 rpm Oscillation Unspecified
Frequency 0 cycles/sec Amplitude 0 inches

Sun Exposure None Low Temperature 0 degrees F
% Time 0 High Temperature 0 degrees F

Environment Gas _____
Pressurized Liquid _____

Operating Conditions _____

Non-Operating Conditions _____

Comment _____

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

34

Contractor Moog Survey Number 34

WBS Number _____ Survey Date 8/12/86

Function Electric Drive motor actuator for the Oxygen and Nitrogen Latching Valve in the Space Station Propulsion System ARPCS Control System

Description Bearings/Planetary Gears

Surface #1

Surface #2

Contact Surface Unspecified

Contact Surface Unspecified

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Unspecified

SP R 0022A Compatible?

Required Life

Lubricant Unspecified

Yes No

10 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed

Oscillation Unspecified

0 rpm

Frequency

0 cycles/sec

Amplitude

0 inches

Sun Exposure Unspecified

Low Temperature 0 degrees F

% Time 0

High Temperature 0 degrees F

Environment

Gas

Unspecified

Liquid

Operating Conditions

Non-Operating Conditions

Comment

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

35

Contractor Lockheed Survey Number 35

WBS Number _____ Survey Date 3/1/86

Function Wire Wrap transfers power and data across Beta Joint

Description Relative Motion & Contact w/ Structure

Surface #1

Surface #2

Contact Surface Wire Wrap

Contact Surface Axle Housing

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme None Plann

SP R 0022A Compatible?

Required Life

Lubricant _____

Yes No

10 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed
0.00000405 rpm

Oscillation Continuous

Frequency 0 cycles/sec Amplitude 0 inches

Sun Exposure Intermittent Low Temperature 0 degrees F

% Time 64 High Temperature 0 degrees F

Environment Gas _____

Vacuum _____ Liquid _____

Operating
Conditions

Non-Operating
Conditions

Comment

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

36

Contractor Lockheed Survey Number 36

WBS Number _____ Survey Date 3/1/86

Function Beta Joint Control Motor and Gearbox

Description Bearings / Gears

Surface #1

Surface #2

Contact Surface Unspecified

Contact Surface Unspecified

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Unspecified

SP R 0022A Compatible?

Required Life

Lubricant Unspecified

Yes No

10 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed

Oscillation Unspecified

0 rpm

Frequency 0 cycles/sec

Amplitude 0 inches

Sun Exposure None

Low Temperature 0 degrees F

% Time 0

High Temperature 0 degrees F

Environment

Gas _____

Vacuum

Liquid _____

Operating Conditions

Non-Operating Conditions

Comment

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

37

Contractor Lockheed Survey Number 37

WBS Number _____ Survey Date 3/1/86

Function Main Bearings in the housing of the Beta Joint of the Electrical Power System

Description Main Bearings

Surface #1

Surface #2

Contact Surface Inner Race

Contact Surface Outer Race

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Unspecified

SP R 0022A Compatible?

Required Life

Lubricant Unspecified

Yes No

10 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed

0 rpm

Oscillation Unspecified

Frequency 0 cycles/sec Amplitude 0 inches

Sun Exposure Intermittent

Low Temperature 0 degrees F

% Time 64

High Temperature 0 degrees F

Environment

Gas _____

Vacuum

Liquid _____

Operating Conditions

Non-Operating Conditions

Comment

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

38

Contractor Lockheed Survey Number 38

WBS Number _____ Survey Date 3/1/86

Function Bearings in pump of water supply assembly of water electrolysis system

Description Bearings

Surface #1

Surface #2

Contact Surface _____

Contact Surface _____

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Unspecified

SP R 0022A Compatible?

Required Life

Lubricant Unspecified

Yes No

10 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed

Oscillation Unspecified

0 rpm

Frequency

0 cycles/sec

Amplitude

0 inches

Sun Exposure Unspecified

Low Temperature

0 degrees F

% Time 0

High Temperature

0 degrees F

Environment

Gas _____

Unspecified

Liquid _____

Operating Conditions

Non-Operating Conditions

Comment

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

39

Contractor Lockheed Survey Number 39

WBS Number _____ Survey Date 3/1/86

Function Cable fair lead pulleys in blanket tensioning and guidewire system of solar array of electrical power

Description Cable fair lead pulleys

Surface #1

Surface #2

Contact Surface pin/al

Contact Surface bushing/al

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Dry Film

SP R 0022A Compatible?

Required Life

Lubricant Everlube 620

Yes No

10 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed

0 rpm

Oscillation

Frequency 0 cycles/sec Amplitude 0 inches

Sun Exposure Intermittent

Low Temperature 0 degrees F

% Time 64

High Temperature 0 degrees F

Environment

Gas

Vacuum

Liquid

Operating Conditions

Non-Operating Conditions

Comment

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

40

Contractor Lockheed Survey Number 40

WBS Number _____ Survey Date 3/1/86

Function Drive motors in fine pointing mechanism of solar dynamic system of electrical power system

Description Drive motors

Surface #1

Surface #2

Contact Surface _____	Contact Surface _____
Surface Finish <u>Unspecified</u>	Surface Finish <u>Unspecified</u>
Surface Treatment <u>Unspecified</u>	Surface Treatment <u>Unspecified</u>

Lubrication Scheme <u>Unspecified</u>	SP R 0022A Compatible?	Required Life
Lubricant <u>Working Fluid</u>	<input type="radio"/> Yes <input checked="" type="radio"/> No	<u>10</u> years

Total Mass Loss 0 % CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed <u>0</u> rpm	Oscillation	Amplitude <u>0</u> inches
Frequency <u>0</u> cycles/sec		

Sun Exposure <u>Unspecified</u>	Low Temperature <u>0</u> degrees F
% Time <u>0</u>	High Temperature <u>0</u> degrees F

Environment	Gas _____
<u>Unspecified</u>	Liquid _____

Operating Conditions

Non-Operating Conditions

Comment

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

41

Contractor Lockheed Survey Number 41

WBS Number _____ Survey Date 3/1/86

Function Gearbox of fine pointing mechanism in solar dynamic system in electrical power system

Description Gearbox

Surface #1		Surface #2	
Contact Surface	_____	Contact Surface	_____
Surface Finish	<u>Unspecified</u>	Surface Finish	<u>Unspecified</u>
Surface Treatment	<u>Unspecified</u>	Surface Treatment	<u>Unspecified</u>

Lubrication Scheme Unspecified SP R 0022A Compatible? Yes No Required Life 10 years
 Lubricant Working Fluid

Total Mass Loss 0 % CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed 0 rpm Oscillation Unspecified
 Frequency 0 cycles/sec Amplitude 0 inches

Sun Exposure Unspecified Low Temperature 0 degrees F
 % Time 0 High Temperature 0 degrees F

Environment Gas _____
 Unspecified Liquid _____

Operating Conditions

Non-Operating Conditions

Comment

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

42

Contractor Lockheed Survey Number 42

WBS Number _____ Survey Date 3/1/86

Function Ballscrew in fine pointing mechanism of solar dynamic system of electrical power system

Description Ballscrew

Surface #1

Surface #2

Contact Surface _____

Contact Surface _____

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Unspecified

SP R 0022A Compatible?

Required Life

Lubricant Working Fluid

Yes No

10 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed
0 rpm

Oscillation _____

Frequency 0 cycles/sec

Amplitude 0 inches

Sun Exposure Unspecified

Low Temperature 0 degrees F

% Time 0

High Temperature 0 degrees F

Environment

Gas _____

Unspecified

Liquid _____

Operating Conditions

Non-Operating Conditions

Comment

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

43

Contractor Lockheed Survey Number 43

WBS Number _____ Survey Date 3/1/86

Function Actuator-to-concentrator I/F of fine pointing mechanism of solar dynamic system of electrical power system

Description Actuator-to-concentrator I/F

Surface #1

Surface #2

Contact Surface _____

Contact Surface _____

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Unspecified

SP R 0022A Compatible?

Required Life

Lubricant Working fluid

Yes No

10 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed
0 rpm

Oscillation Unspecified

Frequency 0 cycles/sec Amplitude 0 inches

Sun Exposure Unspecified Low Temperature 0 degrees F

% Time 0 High Temperature 0 degrees F

Environment

Gas _____

Unspecified

Liquid _____

Operating Conditions

Non-Operating Conditions

Comment

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

44

Contractor Lockheed Survey Number 44

WBS Number _____ Survey Date 3/1/86

Function Gas or foil bearings in heat engine of solar dynamic system of electrical power system

Description Gas or foil bearings

Surface #1

Surface #2

Contact Surface _____	Contact Surface _____
Surface Finish <u>Unspecified</u>	Surface Finish <u>Unspecified</u>
Surface Treatment <u>Unspecified</u>	Surface Treatment <u>Unspecified</u>

Lubrication Scheme <u>Unspecified</u>	SP R 0022A Compatible?	Required Life
Lubricant <u>Working fluid</u>	<input type="radio"/> Yes <input checked="" type="radio"/> No	<u>10</u> years

Total Mass Loss 0 % CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed <u>0</u> rpm	Oscillation <u>Unspecified</u>
Frequency <u>0</u> cycles/sec	Amplitude <u>0</u> inches

Sun Exposure <u>Unspecified</u>	Low Temperature <u>0</u> degrees F
% Time <u>0</u>	High Temperature <u>0</u> degrees F

Environment	Gas _____
<u>Unspecified</u>	Liquid _____

Operating Conditions _____

Non-Operating Conditions _____

Comment _____

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

45

Contractor Lockheed Survey Number 45

WBS Number 1.1.4.3.16.16.1.2.2 Survey Date 3/1/86

Function Mast base spider bushing in mast canister of solar array of electrical power

Description Mast base spider bushing

Surface #1

Surface #2

Contact Surface spider bushing/Al.

Contact Surface spider/Al.

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Dry Film

SP R 0022A Compatible?

Required Life

Lubricant Everlube 620

Yes No

10 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed
0 rpm

Oscillation Intermittent

Frequency 0 cycles/sec Amplitude 0 inches

Sun Exposure Intermittent

Low Temperature 0 degrees F

% Time 63

High Temperature 0 degrees F

Environment

Gas

Vacuum

Liquid

Operating Conditions

Radiation type: NASA TM-86460
Normal operating environment 160 °F

Non-Operating Conditions

Comment

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

46

Contractor Lockheed Survey Number 46

WBS Number 1.1.4.3.16.16.1.2.3 Survey Date 3/1/86

Function Longer on end fitting in mast canister of solar array of electrical power

Description Fitting to spider Al. to Al.

Surface #1

Surface #2

Contact Surface On end fitting/Al.

Contact Surface Pin to spider/Al.

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Dry Film

SP R 0022A Compatible?

Required Life

Lubricant Everlube 620

Yes No

10 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed
0 rpm

Oscillation Intermittent

Frequency 0 cycles/sec Amplitude 0 inches

Sun Exposure Intermittent

Low Temperature 0 degrees F

% Time 63

High Temperature 0 degrees F

Environment

Gas

Vacuum

Liquid

Operating Conditions

Radiation type: NASA TM-86460
Normal operating Environment: 160°F

Non-Operating Conditions

Comment

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

47

Contractor Lockheed Survey Number 47

WBS Number 1.1.4.3.16.16.1.2.4 Survey Date 3/1/86

Function Mast element/deployer interface in mast canister of solar array of electrical power

Description Mast element to deployer interface

Surface #1

Surface #2

Contact Surface mast ele. roller/Al.

Contact Surface Deployer nut thread

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Dry Film

SP R 0022A Compatible?

Required Life

Lubricant Everlube 620

Yes No

10 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed
0 rpm

Oscillation Intermittent

Frequency 0 cycles/sec Amplitude 0 inches

Sun Exposure Intermittent Low Temperature 0 degrees F

% Time 63 High Temperature 0 degrees F

Environment

Gas

Vacuum

Liquid

Operating Conditions Radiation Type: NASA TM-86460
Normal Operating Environment: 160°F

Non-Operating Conditions

Comment

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

50

Contractor Lockheed Survey Number 50

WBS Number _____ Survey Date 3/1/86

Function Bearing in pump of pressure control assembly of water electrolysis system

Description Bearings

Surface #1

Surface #2

Contact Surface _____

Contact Surface _____

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Unspecified

SP R 0022A Compatible?

Required Life

Lubricant Unspecified

Yes No

10 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed

Oscillation Unspecified

0 rpm

Frequency 0 cycles/sec

Amplitude 0 inches

Sun Exposure Unspecified

Low Temperature 0 degrees F

% Time 0

High Temperature 0 degrees F

Environment

Gas _____

Unspecified

Liquid _____

Operating Conditions

Operating Load: 300 psi, 180°F, Intermittent as req'd

Non-Operating Conditions

Comment

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

51

Contractor Lockheed Survey Number 51

WBS Number _____ Survey Date 3/1/86

Function Motor pinion gear in cover blanket preload system of solar array of electrical power

Description Motor pinion gear

Surface #1

Surface #2

Contact Surface Worn gear/Al.

Contact Surface Latch member/SS

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Dry Film

SP R 0022A Compatible?

Required Life

Lubricant Everlube 620

Yes No

10 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed
0 rpm

Oscillation Unspecified

Frequency 0 cycles/sec Amplitude 0 inches

Sun Exposure Intermittent

Low Temperature 0 degrees F

% Time 64

High Temperature 0 degrees F

Environment

Gas

Vacuum

Liquid

Operating
Conditions

Non-Operating
Conditions

Comment

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

52

Contractor Lockheed Survey Number 52

WBS Number _____ Survey Date 3/1/86

Function Motor ballbearings in ocver blanket preload system of solar array of electrical power

Description Motor ballbearings

Surface #1

Surface #2

Contact Surface Bearings/SS

Contact Surface Race/SS

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Fluid *

SP R 0022A Compatible?

Required Life

Lubricant Braycote 601

Yes No

10 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed

Oscillation Unspecified

0 rpm

Frequency 0 cycles/sec

Amplitude 0 inches

Sun Exposure Intermittent

Low Temperature 0 degrees F

% Time 64

High Temperature 0 degrees F

Environment

Gas _____

Vacuum _____

Liquid _____

Operating Conditions

Non-Operating Conditions

Comment

* Grease

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

53

Contractor Lockheed Survey Number 53

WBS Number _____ Survey Date 3/1/86

Function Latch system in cover blanket preload system of solar array of electrical power

Description Latch system

Surface #1

Surface #2

Contact Surface pins in latch/Al.

Contact Surface Bushing in latch/Al.

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Dry Film

SP R 0022A Compatible?

Required Life

Lubricant Everlube 620

Yes No

10 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed
0 rpm

Oscillation Unspecified

Frequency 0 cycles/sec Amplitude 0 inches

Sun Exposure Intermittent Low Temperature 0 degrees F

% Time 64 High Temperature 0 degrees F

Environment

Gas _____

Vacuum

Liquid _____

Operating
Conditions

Non-Operating
Conditions

Comment

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

54

Contractor Lockheed Survey Number 54

WBS Number _____ Survey Date 3/1/86

Function Negator springs to rage reel in blanket tensioning and guide wire system in solar array in electrical power

Description Negator springs to rage reel

Surface #1

Surface #2

Contact Surface hub/Al or Delrin

Contact Surface Axle/Al.

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Dry Film

SP R 0022A Compatible?

Required Life

Lubricant Everlube 620 *

Yes No

10 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed

Oscillation Unspecified

0 rpm

Frequency 0 cycles/sec Amplitude 0 inches

Sun Exposure Intermittent

Low Temperature 0 degrees F

% Time 64

High Temperature 0 degrees F

Environment

Gas _____

Vacuum _____

Liquid _____

Operating Conditions

Non-Operating Conditions

Comment

* None for Delrin Hub

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

55

Contractor Lockheed Survey Number 55

WBS Number _____ Survey Date 3/1/86

Function Limit switches in array motors of solar array of electrical power

Description Limit switches

Surface #1

Surface #2

Contact Surface Pins/SS

Contact Surface Brackets/SS

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Dry Film

SP R 0022A Compatible?

Required Life

Lubricant Everlube 620

Yes No

10 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed
0 rpm

Oscillation Unspecified

Frequency 0 cycles/sec Amplitude 0 inches

Sun Exposure Intermittent Low Temperature 0 degrees F

% Time 64 High Temperature 0 degrees F

Environment

Gas

Vacuum

Liquid

Operating
Conditions

Non-Operating
Conditions

Comment

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

56

Contractor Lockheed Survey Number 56

WBS Number _____ Survey Date 3/1/86

Function Interface/Mast canister to beta drive in mast canister of solar array of electrical power

Description Interface/Mast canister to beta drive

Surface #1

Surface #2

Contact Surface Mast canister/SS&Al.

Contact Surface Beta drive/Al

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Dry Film

SP R 0022A Compatible?

Required Life

Lubricant Everlube 620

Yes No

10 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed

Oscillation Unspecified

0 rpm

Frequency 0 cycles/sec Amplitude 0 inches

Sun Exposure Intermittent

Low Temperature 0 degrees F

% Time 64

High Temperature 0 degrees F

Environment

Gas _____

Vacuum

Liquid _____

Operating
Conditions

Non-Operating
Conditions

Comment

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

57

Contractor Lockheed Survey Number 57

WBS Number _____ Survey Date 3/1/86

Function Rotor ballbearings in mast motor drive assembly of solar array of electrical power

Description Rotor ball bearings

Surface #1

Surface #2

Contact Surface Bearings/SS

Contact Surface Race/SS

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Fluid *

SP R 0022A Compatible?

Required Life

Lubricant Braycote 601

Yes No

10 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed

Oscillation Unspecified

0 rpm

Frequency 0 cycles/sec

Amplitude 0 inches

Sun Exposure Intermittent

Low Temperature 0 degrees F

% Time 64

High Temperature 0 degrees F

Environment

Gas _____

Vacuum _____

Liquid _____

Operating Conditions

Non-Operating Conditions

Comment

* Grease

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

58

Contractor Lockheed Survey Number 58

WBS Number _____ Survey Date 3/1/86

Function Clutch or solenoid in locking mechanism of beta joint of electrical/mechanisms

Description clutch or solenoid

Surface #1

Surface #2

Contact Surface clutchpin

Contact Surface sleeve

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Unspecified

SP R 0022A Compatible?

Required Life

Lubricant Unspecified

Yes No

10 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed

Oscillation Unspecified

0 rpm

Frequency 0 cycles/sec

Amplitude 0 inches

Sun Exposure Unspecified

Low Temperature 0 degrees F

% Time 0

High Temperature 0 degrees F

Environment

Gas _____

Unspecified

Liquid _____

Operating Conditions

Non-Operating Conditions

Comment

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

59

Contractor Lockheed Survey Number 59

WBS Number _____ Survey Date 3/1/86

Function Pin puller piston or solenoid piston in launch support and release for box ends of solar array of electrical power

Description Pin puller piston or solenoid piston

Surface #1

Surface #2

Contact Surface Piston/Al

Contact Surface Sleeve/Al.

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Dry Film

SP R 0022A Compatible?

Required Life

Lubricant Everlube 620

Yes No

10 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed

Oscillation Unspecified

0 rpm

Frequency 0 cycles/sec

Amplitude 0 inches

Sun Exposure Intermittent

Low Temperature 0 degrees F

% Time 64

High Temperature 0 degrees F

Environment

Gas _____

Vacuum

Liquid _____

Operating Conditions

Non-Operating Conditions

Comment

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

60

Contractor Lockheed Survey Number 60

WBS Number _____ Survey Date 3/1/86

Function Bearings in centrifugal pump of H2 pump/separator of fluid control system

Description Bearings

Surface #1

Surface #2

Contact Surface _____ Contact Surface _____

Surface Finish Unspecified Surface Finish Unspecified

Surface Treatment Unspecified Surface Treatment Unspecified

Lubrication Scheme Unspecified SP R 0022A Compatible? Yes No Required Life 10 years
Lubricant Unspecified

Total Mass Loss 0 % CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed 0 rpm Oscillation Unspecified
Frequency 0 cycles/sec Amplitude 0 inches

Sun Exposure Unspecified Low Temperature 0 degrees F
% Time 0 High Temperature 0 degrees F

Environment Gas _____
Unspecified Liquid _____

Operating Conditions _____

Non-Operating Conditions _____

Comment Operating Load: 60 psi, 140°F, Continuous

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

61

Contractor Lockheed Survey Number 61

WBS Number _____ Survey Date 3/1/86

Function Bearings in pump of coolant pump of fluid control system

Description Bearings

Surface #1

Surface #2

Contact Surface _____

Contact Surface _____

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Unspecifie

SP R 0022A Compatible?

Required Life

Lubricant Unspecified

Yes No

10 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed
0 rpm

Oscillation Unspecified

Frequency 0 cycles/sec Amplitude 0 inches

Sun Exposure Unspecified
% Time 0

Low Temperature 0 degrees F

High Temperature 0 degrees F

Environment

Gas _____

Unspecified

Liquid _____

Operating
Conditions

Operating Load: 60 psi, 180°F, continuous

Non-Operating
Conditions

Comment

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

62

Contractor Lockheed Survey Number 62

WBS Number _____ Survey Date 3/1/86

Function Deployer guide in mast canister of solar array of electrical power

Description Deployer guide

Surface #1	Surface #2
Contact Surface <u>Fixed dep. gu./delri</u>	Contact Surface <u>Mast ele. roller/al.</u>
Surface Finish <u>Unspecified</u>	Surface Finish <u>Unspecified</u>
Surface Treatment <u>Unspecified</u>	Surface Treatment <u>Unspecified</u>

Lubrication Scheme Unspecifie SP R 0022A Compatible? Yes No Required Life 10 years
 Lubricant None

Total Mass Loss 0 % CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed 0 rpm Oscillation Unspecified
 Frequency 0 cycles/sec Amplitude 0 inches

Sun Exposure Intermittent Low Temperature 0 degrees F
 % Time 64 High Temperature 0 degrees F

Environment Gas _____
 Vacuum _____ Liquid _____

Operating Conditions _____

Non-Operating Conditions _____

Comment _____

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

63

Contractor Lockheed Survey Number 63

WBS Number _____ Survey Date 3/1/86

Function Bearings in pump of coolant control assembly of water electrolysis system

Description Bearings

Surface #1

Surface #2

Contact Surface _____

Contact Surface _____

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Unspecified

SP R 0022A Compatible?

Required Life

Lubricant Unspecified

Yes No

10 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed

Oscillation Unspecified

0 rpm

Frequency 0 cycles/sec Amplitude 0 inches

Sun Exposure Unspecified

Low Temperature 0 degrees F

% Time 0

High Temperature 0 degrees F

Environment

Gas _____

Unspecified

Liquid _____

Operating Conditions

Operating Load: 300 psi, 180°F, continuous

Non-Operating Conditions

Comment

Pumps water or fluorinert through electrolyzer

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

64

Contractor Lockheed Survey Number 64

WBS Number _____ Survey Date 3/1/86

Function Interface/containment box to mast in containment box of solar array of electrical power

Description Interface/containment box to mast

Surface #1

Surface #2

Contact Surface Contain box/SS & Al.

Contact Surface Mast/Al.

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Dry Film

SP R 0022A Compatible?

Required Life

Lubricant Everlube 620

Yes No

10 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed

Oscillation Unspecified

0 rpm

Frequency 0 cycles/sec

Amplitude 0 inches

Sun Exposure Intermittent

Low Temperature 0 degrees F

% Time 64

High Temperature 0 degrees F

Environment

Gas _____

Vacuum _____

Liquid _____

Operating Conditions

Non-Operating Conditions

Comment

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

65

Contractor Lockheed Survey Number 65

WBS Number _____ Survey Date 3/1/86

Function Motor drive pinion gear in array containment box positioning system of solar array of electrical power

Description Motor drive pinion gear

Surface #1

Surface #2

Contact Surface <u>*</u>	Contact Surface <u>Actuator/SS</u>
Surface Finish <u>Unspecified</u>	Surface Finish <u>Unspecified</u>
Surface Treatment <u>Unspecified</u>	Surface Treatment <u>Unspecified</u>

Lubrication Scheme <u>Dry Film</u>	SP R 0022A Compatible?	Required Life
Lubricant <u>Everlube 620</u>	<input type="radio"/> Yes <input checked="" type="radio"/> No	<u>10</u> years

Total Mass Loss 0 % CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed <u>0</u> rpm	Oscillation <u>Unspecified</u>
Frequency <u>0</u> cycles/sec	Amplitude <u>0</u> inches

Sun Exposure <u>Intermittent</u>	Low Temperature <u>0</u> degrees F
% Time <u>64</u>	High Temperature <u>0</u> degrees F

Environment	Gas	_____
Vacuum	Liquid	_____

Operating Conditions _____

Non-Operating Conditions _____

Comment * Motor drive pinion or worm gear/Al.

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

66

Contractor Lockheed Survey Number 66

WBS Number _____ Survey Date 3/1/86

Function Positioner system in array containment box positioning system of solar array of electrical power

Description Positioner system

Surface #1

Surface #2

Contact Surface pins in pos. sys./Al

Contact Surface Bushing in p.s./Al

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Dry Film

SP R 0022A Compatible?

Required Life

Lubricant Everlube 620

Yes No

10 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed

Oscillation Unspecified

0 rpm

Frequency 0 cycles/sec

Amplitude 0 inches

Sun Exposure Intermittent

Low Temperature 0 degrees F

% Time 64

High Temperature 0 degrees F

Environment

Gas _____

Vacuum _____

Liquid _____

Operating Conditions

Non-Operating Conditions

Comment

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

67

Contractor Lockheed Survey Number 67

WBS Number _____ Survey Date 3/1/86

Function Motor ball bearings in array containment box positioning system of solar array of electrical power

Description Motor ball bearings

Surface #1

Surface #2

Contact Surface Bearings/SS

Contact Surface Race/SS

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Fluid *

SP R 0022A Compatible?

Required Life

Lubricant Braycote 601

Yes No

10 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed

Oscillation Unspecified

0 rpm

Frequency 0 cycles/sec

Amplitude 0 inches

Sun Exposure Intermittent

Low Temperature 0 degrees F

% Time 64

High Temperature 0 degrees F

Environment

Gas _____

Vacuum

Liquid _____

Operating Conditions

Non-Operating Conditions

Comment

* Grease

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

68

Contractor Lockheed Survey Number 68

WBS Number _____ Survey Date 3/1/86

Function Latch and release mechanism in array containment box positioning system of solar array of electrical power

Description Latch and release mechanism

Surface #1

Surface #2

Contact Surface Latch/Al

Contact Surface Release/Al

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Dry Film

SP R 0022A Compatible?

Required Life

Lubricant Everlube 620

Yes No

10 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed
0 rpm

Oscillation Unspecified

Frequency 0 cycles/sec Amplitude 0 inches

Sun Exposure Intermittent

Low Temperature 0 degrees F

% Time 64

High Temperature 0 degrees F

Environment

Gas _____

Vacuum

Liquid _____

Operating Conditions

Non-Operating Conditions

Comment

Mechanism holds box position

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

69

Contractor Lockheed Survey Number 69

WBS Number _____ Survey Date 3/1/86

Function Box-to-mast hinges system in array containment box positioning system of solar array of electrical power

Description Box-to-mast hinges system

Surface #1

Surface #2

Contact Surface Pin/Al. *

Contact Surface Bushings/Al. *

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Dry Film
Lubricant Everlube 620

SP R 0022A Compatible?
 Yes No

Required Life
10 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed
0 rpm

Oscillation Unspecified

Frequency 0 cycles/sec

Amplitude 0 inches

Sun Exposure Intermittent
% Time 64

Low Temperature 0 degrees F

High Temperature 0 degrees F

Environment

Gas _____

Vacuum

Liquid _____

Operating Conditions

Non-Operating Conditions

Comment

* in box-to-mast hinges system

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

70

Contractor Lockheed Survey Number 70

WBS Number _____ Survey Date 3/1/86

Function Pinion gear in mast canister of solar array of electrical power

Description Pinion gear

Surface #1

Surface #2

Contact Surface *

Contact Surface **

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Dry Film

SP R 0022A Compatible?

Required Life

Lubricant Everlube 620

Yes No

10 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed

Oscillation Unspecified

0 rpm

Frequency 0 cycles/sec

Amplitude 0 inches

Sun Exposure Intermittent

Low Temperature 0 degrees F

% Time 64

High Temperature 0 degrees F

Environment

Gas _____

Vacuum _____

Liquid _____

Operating Conditions

Non-Operating Conditions

Comment

* Deployer nut ring gear Al.
** Drive assembly pinion gear/SS

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

71

Contractor Lockheed Survey Number 71

WBS Number _____ Survey Date 3/1/86

Function Pinion gear in mast canister of solar array of electrical power

Description Pinion gear

Surface #1

Surface #2

Contact Surface *

Contact Surface **

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Dry Film

SP R 0022A Compatible?

Required Life

Lubricant Everlube 620

Yes No

10 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed

0 rpm

Oscillation Unspecified

Frequency

0 cycles/
sec

Amplitude

0 inches

Sun Exposure Intermittent

Low Temperature 0 degrees F

% Time 64

High Temperature 0 degrees F

Environment

Gas

Vacuum

Liquid

Operating
Conditions

Non-Operating
Conditions

Comment

* Deployer nut ring gear/AL
** Drive assembly pinion gear/SS

**SPACE STATION LONG TERM LUBRICATION STUDY
SURVEY FORM**

72

Contractor Lockheed Survey Number 72

WBS Number _____ Survey Date 3/1/86

Function Cables to rage reel in blanket tensioning and guideline wire system of solar array of electrical power

Description Cables to rage reel

Surface #1

Surface #2

Contact Surface Hub/Al or Delrin

Contact Surface Axle/AL

Surface Finish Unspecified

Surface Finish Unspecified

Surface Treatment Unspecified

Surface Treatment Unspecified

Lubrication Scheme Dry Film

SP R 0022A Compatible?

Required Life

Lubricant Everlube 620 *

Yes No

10 years

Total Mass Loss 0 %

CVCM 0 %

Loads: Operating 0 lb Peak 0 lb Preload 0 lb

Rotational Speed

Oscillation Unspecified

0 rpm

Frequency 0 cycles/sec

Amplitude 0 inches

Sun Exposure Intermittent

Low Temperature 0 degrees F

% Time 64

High Temperature 0 degrees F

Environment

Gas _____

Vacuum _____

Liquid _____

Operating Conditions

Non-Operating Conditions

Comment

* none for Delrin hub

